A Solution to the Inherent List on Nimitz Class Aircraft Carriers

by

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B.S. Marine Engineering Systems, United States Merchant Marine Academy, 1996

Submitted to the Departments of Ocean Engineering and Civil and Environmental Engineering in Partial Fulfillment of the Requirements for the Degrees of

Naval Engineer and
Master of Science in Civil and Environmental Engineering

at the Massachusetts Institute of Technology June 2004

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Abstract

Nimitz class aircraft carriers possess an inherent list to starboard that their list control systems (LCS) are typically unable to correct while under Combat Load Conditions. As a result, it has become necessary to use fresh water ballast in a number of inner bottom voids and damage control voids to augment the LCS. Maintaining liquid ballast in damage control voids is unacceptable, as it reduces the design counter flooding capability of the ship, and thus reduces ship survivability. In order to restore the ships operational flexibility and achieve the necessary/desired list correction, this study determines the effect of adding solid ballast to a series of voids/tanks identified on the 2nd, 4th, and 8th decks.

Based on ballast density, tank location and capacity, ease of ballast installation, minor tank structural modifications, and a decision making cost analysis, solid ballast was determined to be the most advantageous for use in correcting the inherent list on the Nimitz class aircraft carriers. Fresh water ballast was also examined as a possible alternative, but not as extensively due to the large quantity of water required and its limited ability to achieve a list correction.

Nimitz class aircraft carriers currently have an average list of 1.5 degrees and a KG of 47 feet. Since their allowable KG cannot exceed 48.5 feet, the average service life allowance (SLA) for KG is approximately 1.5 feet. This study shows that by adding approximately 400 lton of solid ballast, list can be corrected by 1.5 degrees with only a 0.1 percent increase in KG. Thus, to permanently fix the average Nimitz class aircraft carrier starboard list, there would be a 0.05 foot increase in KG, which in all cases is within the SLA. Additionally, this study shows that this 1.5 degree list correction can be accomplished at a low cost of approximately \$1,200 per lton. Considering the reduction in operational constraints and the benefits to ship survivability, this is truly an inexpensive proposition.

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Table of Contents

Abstract	
Acknowledgements	4
Table of Contents	5
List of Figures	6
List of Tables	6
List of Appendices	6
1.0 Introduction	7
1.0 Introduction	7
1.1 Motivation	7
1.2 Background	7
1.2.1 Past Research	
1.2.2 Stability	10
1.2.3 Survivability	13
1.2.4 Displacement Limits	14
1.2.5 KG Limits	
1.3 Present Options	15
1.3.1 Continue to operate using inner bottom and DC voids	15
1.3.2 Add a list control tank on the port side	17
1.3.3 Move or exchange compartment spaces	18
1.3.4 Convert a current DC void(s) to a JP-5 tank(s) and pump last	18
1.3.5 Future ship alterations and modifications	18
1.3.6 Re-examine current List Control System	19
1.3.7 Ballast Addition	
1.3.7.1 Water Ballast	
1.3.7.2 Solid Ballast	21
1.4 Option Selection	23
2.0 Preliminary Analysis and Results	
2.1 Tank Selection	24
2.2 Modeling Analysis Performed	25
2.3 Preliminary Decision-Making Cost Estimation	31
2.4 Structural Analysis	35
2.4.1 Structural Modifications	
2.4.1.1 Modifications to the 2 nd Deck	39
2.4.1.2 Modifications to the 4 th Deck	41
2.4.1.3 Modifications to the 8 th Deck	
3.0 Final Analysis and Results	
3.1 Final POSSE Results	
3.2 Final Cost Estimation	
3.2.1 Ease of Removal	
3.2.2 Evidence Perma Ballast® is non-corrosive	
4.0 Conclusion	
5.0 Recommendations	
References	53

List of Figures

Figure 1: Stresses in rectangular plates under uniform lateral pressure [2]	38
Figure 2: 2 nd Deck Compartment 2-165-8-V	39
Figure 3: 2 nd Deck Compartment 2-165-8-V, transverse section	41
Figure 4: 4 th Deck Compartment 4-165-4-V	42
Figure 5: 8 th Deck Compartment 8-225-6-V	43
List of Tables	
Table 1: CVN 68 Class Delivery Data and Class Predictions as of 11/17/03	8
Table 2: Deck Location Cost Comparison	32
Table 3: Required Structural Modifications for 2-165-8-V	40
Table 4: Required Structural Modifications for 4-165-4-V	42
Table 5: Required Structural Modifications for 8-225-6-V	44
Table 6: Final Cost, Weight Addition, and Change in KG Comparison	47
List of Appendices	
Appendix A: Tank Location Study	54
Appendix B: Preliminary POSSE Modeling Results	58
Appendix C: Preliminary Cost Estimation Data and Worksheets	62
Appendix D: Preliminary Cost Comparison	86
Appendix E: Complete Structural Analysis	88
Appendix F: Final POSSE Modeling Results	148
Appendix G: Final Cost Estimation Data and Worksheets	150

1.0 Introduction

1.1 Motivation

Aircraft carriers are the largest combatant ships in the world. They are huge floating cities, carrying thousands of sailors and aircraft, each vessel with more military power than many nations. But with such might comes mighty operational requirements.

In order to launch and recover aircraft on the flight deck, most ships will report any change in list in excess of ½ degree to the Engineering Officer of the Watch (EOOW) for correction. The Nimitz class aircraft carrier was designed to maintain a level deck during flight operations through the utilization of the LCS. The LCS is designed to compensate for the operational effects of aircraft movement on the flight deck and in the hangar bays and not intended to compensate for an inherent list. Any inherent list imposes operational constraints on the ship, particularly when the carrier has embarked a full air-wing and full fuel loading (Combat Load Condition). Nimitz class carriers have a history of inherent starboard list, due primarily to the ship's configuration. History shows that modifications to the Nimitz class are have increased the inherent starboard list of each carrier. This thesis will explore various options for finding a permanent solution to the Nimitz class list control issues, particularly, the installation of solid ballast to counter list.

1.2 Background

There are a number of obstacles in finding a permanent solution for the inherent list associated with most Nimitz class carriers, as each carrier has a different inherent list associated with it. This may sound strange, but every ship is slightly different. The ships are so large and have such a high procurement cost that only one is built at a time. It takes an average of five years to build one carrier, and in that time, modernizations, upgrades, and improvements are

introduced into the design. As such, the ships are constantly changing and evolving, all within the same skin designed in the late 1960's. A comparative analysis of the commissioned and predicted current displacement, vertical center of gravity (KG), and list with the latest unknown growth corrections for the Nimitz class aircraft carriers are shown in Table 1.

Table 1: CVN 68 Class Delivery Data and Class Predictions as of 11/17/03

	Delivery	Predicted Current		Predicted	Delivery	Predicted Current
Ch:-	Displacement	Displacement	Delivery	Current	List	List
Ship	(ltons)	(ltons)	KG (ft)	KG (ft)	(degrees)	(degrees)
68	93,283	100,064	45.73	47.22	1.01(S)	0.10 (S)
69	93,832	100,588	45.94	47.1	0.82 (S)	0.25 (S)
70	94,069	100,599	46.29	47.28	1.22 (S)	0.11 (P)
71	96,865	103,700	46.4	47.27	0.27 (S)	0.41 (S)
72	97,497	103,912	46.61	47.2	0.77 (S)	1.81 (S)
73	97,816	104,095	46.54	47.03	0.85 (S)	1.55 (S)
74	97,490	103,419	46.63	47.23	1.63 (S)	2.54 (S)
75	97,944	103,863	46.4	46.93	0.43 (S)	1.61 (S)
76	97,953	101,187	46.66	46.77	0.08 (S)	0.99 (S)

Nimitz class carrier contracts require that an Accepted Weight Estimate (AWE) be negotiated between the Contractor and the Government. A starboard inherent list was seen creeping upwards through CVN 73, such that for the CVN 74/75 contract, the program office invoked a list tolerance of 0.50 degree (P/S) for the combat load condition. As a result of this new requirement, and beginning with the CVN 74/75 contract, the Navy and the contractor agreed to include 850 long tons of contractor-controlled ballast in the AWE to offset the already projected design inherent starboard list. Before any real ballast was to be added on CVN 74, however, the program office wanted to see if there was indeed a list problem after the ship was delivered and fully loaded. It is at this time that CVN 74 was found to a have real inherent starboard list problem. The identified list problem was resolved by the use of fresh water (FW) ballast. To date, permanent ballast has not been installed on any aircraft carrier of the Nimitz

class. Changes to the CVN 76 design resulted in a reduced starboard list and consequently the contractor-controlled ballast included in the AWE was reduced to 650 long tons.

From operational experience, it would be safe to say that there is an inherent starboard list under full fuel load (or virtually full) with the airwing embarked and no flight operations being conducted on most Nimitz class carriers. When all the planes are stacked on the starboard side and the ship has just been refueled, the LCS is unable to level the deck. This is when it has become necessary for some carriers to use fresh water ballast in a number of inner bottom voids and damage control (DC) voids to correct the adverse list condition. But from a ship survivability standpoint, and as their name implies, DC voids are not an acceptable list control measure under normal operating conditions. Maintaining liquid ballast in DC voids reduces the design counter flooding capability of the ship. None the less, this is how some ships operate in order to keep the Nimitz class ships at sea.

In February 1999, PEO Carriers issued a Message to the CVN 68 Class identifying several voids (i.e., non DC voids) that could be used for FW ballasting for ships experiencing list control problems. Prior to using those voids, however, the message requested the ships to confirm that the list control difficulties were not the result of abnormal LCS operations or adverse liquid loading or stores management. Carriers experiencing list control difficulties were to request approval from the Type Commander (TYCOM) to fill the voids listed in the message. The approval would remain in effect for that ship until a permanent solution was identified and implemented.

It is obvious that fixing the inherent list cannot be solved by leaving aircraft behind or by limiting the load of fuel carried. TYCOM directive states that the JP-5 system is required to

remain at least 60% full at all times. These ships have a very specific warfighting requirement that must be achieved, and a reduction in capabilities is unacceptable.

There must be a solution to the inherent list problem beyond limiting ships payload or using the DC voids outside of their designed intent. This report evaluates other alternatives to finding a solution to fix the inherent list on Nimitz class aircraft carriers.

1.2.1 Past Research

A thesis was performed in 2001 by Michael Malone, a graduate of MIT. His thesis, titled "An Alternate Method for the Determination of Aircraft Carrier Limiting Displacement for Strength;" determined that although all Nimitz class aircraft carriers are approaching their limiting displacement for strength, the traditional methods of such calculations are conservative. In fact, the Nimitz class carriers can accommodate more weight than previously thought.

Because of this thesis, the U.S. Navy Aircraft Carrier Program Office (PMS 312) later directed N.S. NAPPI Associates to conduct a detailed analysis on estimating the limiting displacement for strength. Again, it was proven that the Nimitz class hull is capable of sustaining additional weights which would exceed the current established limiting displacement for strength.

1.2.2 Stability

The stability status for USN ships is defined in [4] and is supported by [11]. The Chief of Naval Operations has directed that the Navy's ships will be kept within naval architectural limits to ensure that essential survivability features are maintained. For each ship class, the Naval Sea Systems Command's Weights and Stability Division is to keep track of the weight and stability status, limiting draft and other limitations including being an advocate for weight and moment status and moment compensation necessary to adhere to the established limits. Surface ships are

classified into four status conditions for vertical center of gravity (KG) and limiting drafts. As specified in [4], the definitions for the status listings are:

- STATUS 1 An increase in weight and a rise of the ship's center of gravity are acceptable. Added weight and moment resulting from changes will not require any compensation unless the magnitude of the additions is so large as to make the ship approach stability limits.
- STATUS 2 Neither an increase in weight nor a rise of a ship's center of gravity can be accepted.
- STATUS 3 An increase in the ship's weight is acceptable, but a rise of the ship's center of gravity must be avoided.
- STATUS 4 A rise of the ship's center of gravity is acceptable, but increase in weight must be avoided. Compensation for added weight may be obtained by removal of an equal or greater weight at any level.

Status 1 is the only acceptable ship status. The goal is for ships in the fleet to remain in Status 1. In the other three status conditions, the displacement and KG must be closely monitored to ensure the condition does not worsen and steps must be taken to try to bring the ship back within acceptable design limits. Typically, when an in-service carrier requires a modification that will add weight to the ship, it is the responsibility of the program office and the planning yard to put together a package that includes maintenance and repair items, as well as modernization changes. First, a preliminary weight estimate of the modernization changes are put together in the package and made ready for installation. Once the package is installed, the planning yard or shipyard keeps track of the weights being installed. At the end of the availability, an actual weight report for the installed changes is generated. This report is passed from the shipyard to the program office where it is incorporated into the stability baseline of the pertinent ship.

For any configuration changes, such as adding weight to a carrier in Stability Status 2, the interested party fills out a Justification Cost Form (JCF) with the weight and KG impact as well as the ship's Stability Status. A Configuration Control Board that includes the Program Office, the Ship Design Manager, and the necessary Technical Authorities then meet to approve or disapprove the modification. If approved, an Engineering Change Proposal (ECP) for newly commissioned ships or a Ship Alteration Request (SAR) for in-service ships is then generated.

The Nimitz class was assigned to Stability Status 2 by the Weight Control and Stability Division of the Naval Sea Systems Command. This was done as a result of the stability reanalysis performed on the Nimitz class carriers. The re-analysis revealed the following:

- The originally calculated Allowable KG of 48.50 feet was no longer applicable for this
 class of ships. While 48.50 feet is no longer the allowable KG value, it continues to be
 the KG comparator for CVN 68 77 for contractual reasons.
- The actual damage capability of this class of carriers is not as originally calculated.
- The damage capability of this class decreases as displacement increases.
- It is important for the holding bulkhead to remain intact.
- The 40'-11" limiting draft is based on ship geometry limitations.
 As a result of the above study, it was determined by that a stability status of 2 should continue to be assigned to the Nimitz class in order to:
 - O Control the displacement growth of the class even though some ships are substantially below the displacement associated with the limiting draft. This is an acknowledgement that an increased displacement degrades damage stability capability.

O Control the KG growth of the class even though some ships were substantially below the KG comparator of 48.50 feet. This is an acknowledgement that an increased KG degrades damage stability capability.

1.2.3 Survivability

Survivability is defined in [10] as the capacity of a ship to absorb damage and maintain mission integrity. Typically, most decision making regarding survivability is done during the early trade-off stages of ship design. As a result, it is imperative that naval architectural parameters be considered over the lifetime of the ship in order to successfully withstand designated threat levels. Every attempt must be made to prevent any degradation of a warship's ability to perform its offensive mission, sustain battle damage, and survive. Aircraft carriers are further defined in [10] as 'capital ships' in that they are expected to survive more than one weapons hit and return to some level of mission capability. All other ships are only expected to survive design level damage from a single design level weapon or a single peacetime hazard.

Survivability, weapons effects and operational environments are categorized in terms of three levels of severity. Level I represents the least severe environment anticipated for a class of ship, while Level III represents the most severe environment projected for a combatant battle group and includes their ability to deal with the broad degrading effects of damage from antiship cruise missiles (ASCM). Aircraft carriers and battle force surface combatants are both considered Level III. Therefore, it is imperative that these ships be operated and maintained as they were designed. Any digression from operational restrictions and guidelines makes all analyses irrelevant.

1.2.4 Displacement Limits

Because these aircraft carriers are so important and because any reduction in freeboard will inherently reduce the effectiveness of the torpedo side protection system (TSPS), these ships are usually displacement critical. The limit is typically expressed in terms of the Full Load Condition. Following is the criteria Naval Surface Warfare Center (NSWC) Carderock's Weight Control and Stability Division uses to determine displacement limits for aircraft carriers with a TSPS:

- Strength: The displacement, with an assumed longitudinal weight distribution, at which the longitudinal bending moments caused by a standardized wave will produce the maximum allowable stress in the ship's hull girder.
- Speed: The displacement for surface warships at which the ships machinery, operating at a specified percent of maximum available power, will drive the ship at the original design speed specified by the ships characteristics considering power plant, RPM and torque limits.
- TSPS: The maximum draft for a surface warship which prevents the top of the TSPS from being immersed more than a specified amount.
- Subdivision: The maximum displacement at which a ship with a TSPS will satisfactorily resist the flooding effects of a specified number of torpedo hits or similar weapons without submerging the margin line at the bow or the stern.
- Damage Stability: The maximum displacement at which a ship with a TSPS will satisfactorily resist the flooding effects of a specified number of torpedo hits or similar weapons while providing adequate stability to resist high static heel angles, resist capsizing, and return to some level of mission capability.

1.2.5 KG Limits

The KG Limit for a warship is the maximum height of the vertical center of gravity of the ship in the Full Load condition. Any Full Load KG below this limit is expected to survive the hazards of wind, high speed maneuvering or damage, assuming the ship follows its liquid loading instructions. The Limiting KG is the lowest limit of either the damage stability limit or

the intact stability limit. Following is the DDS-079-1 criteria NSWC Carderock's Weight Control and Stability Division uses to determine stability limits for surface warships with a SPS:

- Intact: 100 Knot Beam Wind The Full Load KG which will permit the ship to operate in any normal loading condition and survive the heeling force of a fully developed hurricane (assumed as a nominal 100 knots). The ship will retain sufficient stability to absorb higher gusts without being knocked over, and to absorb the dynamic effects of wave action without being rolled over.
- Damage: The Full Load KG Limit for large surface combatants, such as battleships and aircraft carriers which have a side protective system, is associated with the worst possible combination of a specified number of hazards (torpedoes/missiles) which will:
 - 1) Cause the ship to heel to a large initial angle of 15 20 degrees,
 - 2) Cause the ship to approach a capsize situation, or
 - 3) Exceed the counter flooding capability to return to limited operation. Limited operation is defined as a heel of less than 5 degrees to operate aircraft (should be 3 degrees for current aircraft) or less than 10 degrees to operate the main turrets.

In any of the above damage conditions the ship must still possess sufficient dynamic stability to resist capsizing moments induced by wind and waves.

1.3 Present Options

1.3.1 Continue to operate using inner bottom and DC voids

Any departure from proper design and operational criteria degrades any analysis performed. For example, most stability analyses performed assumes that DC voids are used correctly, the ship is on an even keel, and that the limiting displacement is not exceeded. In fact, there is an interrelationship between increasing displacement and a number of factors, to include ship strength, survivability, stability and seakeeping. Increasing the weight, or displacement, of carriers is a serious concern because any weight increase only serves to reduce the service life of the ship. It should be noted again that not all carriers use DC voids to augment the LCS, but it is also uncertain as to how many actually will use them as a last resort.

DC void usage results in a reduction in TSPS defense. The purpose of the TSPS on any capital ship is to protect the vital spaces of the ship against flooding and/or detonation of stowed

ordnance. Vital spaces include magazine and propulsion system spaces. The TSPS provides the desired protection by being constructed of a series of longitudinal bulkheads nested transversely. These longitudinal spaces are further subdivided by transverse fluid-tight bulkheads creating a 'honeycomb' arrangement outboard of the vital magazines and machinery spaces. The spaces thus created between the bulkheads are alternately filled with liquids (fuel oil, JP-5, ballast water, etc.) or are left empty (damage control voids). The passive protection afforded by this combination of bulkheads and 'liquid/air layers" serves to absorb, deflect and reject the explosive force generated by weapons such as torpedoes or mines. Placing liquid in the void layer combined with an empty liquid layer will be the worst-case scenario, while one or the other will result in less of a reduction.

Similarly, DC voids are supposed to be used exactly as their name implies, to combat flooding with the ability to adjust for list and trim in order to continue to fulfill mission objectives after sustained damage by being able to continue to launch and recover aircraft. The specified primary purpose of the DC voids is for damage control. Sea chests and valves for flooding underwater side protection system spaces (DC voids) are required to flood within six minutes to fill the space to within 1 foot of the full load waterline per section 529i of [6]. In fact, current stability criteria require that the ship be able to counter flood to return the flight deck to within 5 degrees of upright after damage. When a ship has to resort to maintaining list by flooding DC voids, the crew becomes encumbered by a task that should be unnecessary. The Damage Control Assistant (DCA) and EOOW are drawn away from their normal tasks and into roles as reactionary problem-solvers, keeping an eye on flight deck aircraft placements, fuel tank levels, and ship list. This is a poor utilization of manpower, considering the primary duties of the DCA and EOOW, which are the coordination of damage control actions and the responsibility

for ships propulsion, respectively. Also, a ship's crew is continuously rotating. As a result, there is a continual learning curve that relies on the passage of operational knowledge from generation to generation. This is not the optimal way to operate a multi-billion dollar asset, and it is in our best interest to not put the onus of correcting an evolutionary design fault on a crew who will serve at most a three year tour. Additionally, de-ballasting DC voids is extremely time intensive. If a real casualty was to occur with the wrong or too many DC voids flooded, there may not be sufficient time to correct the condition. It is not hard to imagine the disastrous results possible.

1.3.2 Add a list control tank on the port side

The control of permanent lists (static heel angles) by adding an additional list control tank of the port side could also be considered an option. The LCS is designed to compensate for variable lists generated by load items that are moved about the ship over short periods of time. Such loads could be liquids, cargo, stores, vehicles or aircraft. The current Nimitz class LCS consists of ten list control tanks, two centrifugal pumps each rated at 1800 gpm at 30 psi, and the associated piping and valves required to compensate for lists up to 1.5 degrees in 20 minutes. The system is designed to be filled with seawater (SW) or firemain to 50% of its total capacity of 278,533 gallons. This allows tanks to be filled to 100% on one side while the other side is left empty.

The problems encountered with using SW in tanks may be common knowledge to most people; however, potential corrosion from using salt water has encouraged some carriers to use potable water (PW) as a means to fill their list control tanks. The LCS is designed with PW as a back-up to using SW or firemain. This is advantageous for corrosion issues, but can be a problem for PW inventory. Any leaks in the system or maintenance requiring system fill and

drain could use a large amount of PW. When underway, potable water conservation is of primary importance.

1.3.3 Move or exchange compartment spaces

Another option is to move or exchange compartment spaces, such as relocating heavy equipment on the starboard side to the port side and vice versa. The reduction of starboard moment by the removal or relocation of items of lightship weight would reduce the quantity of ballast required, but would be very small by comparison to the amount of permanent ballast required. In order to achieve the necessary list corrections, an enormous amount of weight would have to be relocated. The costs involved with such an undertaking would be extremely high.

1.3.4 Convert a current DC void(s) to a JP-5 tank(s) and pump last

Converting portside DC floodable voids into fuel tanks is also an option. This option will add to the overall weight of the ship and again a reduction in TSPS capability becomes a large concern as liquid is being placed in the air layer and thus opening a window of vulnerability.

1.3.5 Future ship alterations and modifications

It is also possible that future modifications to the Nimitz class aircraft carrier will serve to provide a port list, thereby reducing the inherent starboard list. Modifications such as replacing the existing Nimitz class starboard island with the new modified CVN 76 island could serve to reduce the starboard list. In fact, CVN 76 has had so many design changes that its starboard inherent list is less than one degree, currently based on preliminary results of its inclining experiment. List for CVN 76 has not been an issue thus far, however, at the time feedback was gained, the ship had not been fully loaded out. CVN 76 is slightly different from the other Nimitz class ships. The weapons elevator is combined into the main structure and the aft mast is

combined with the island which is one level shorter. There is also a more realistic protection scheme such that a space armor concept was employed. On the older islands, such as CVN 73, Level III protection can be seen throughout the entire island. This includes 40# (1") plating, whereas the new CVN 76 island has Level II protection in some areas and is thus lighter.

Another option could be to extend the port side flight deck on the old Nimitz class, as was done on CVN 76. This allowed for more operational flexibility as the forward port jet blast deflector can now be used to its full extent. In any case, it is important to note that any number of ship alterations could be made when time and money become available in the future.

1.3.6 Re-examine current List Control System

Fleet feedback was taken into consideration when performing this study. The design of the current LCS reportedly requires extensive manpower for operations, possesses obsolete components, and is not integrated with other shipboard liquid-movement functions. The installed LCS consists of 10 tanks, 2 pumps and 14 associated valves to distribute seawater ballast as required compensating for lists up to 1.5 degrees. The manually controlled system uses verbal communications and associated sailor actions for operation.

Much of the fleet feedback received noted that the current LCS provides sufficient control when the inherent starboard list has been corrected. The fleet also noted, however, that if correction has not been provided, the LCS alone does not provide ample list correction while at full combat load (average displacement between 98,000 and 100,000 ltons, mean draft approximately between 38'10" and 39'10", even liquid distribution, and standard spotting of aircraft for flight operations). Standard spotting includes F-18s on the 1 and 4 row, hummers (fleet jargon for the E-2C Hawkeye) in the hummer hole, F-14s on the stern of the flight deck,

helicopters and C-2 Greyhounds inboard of the island, cranes aft of the island, and the rest of the aircraft dispersed between them.

The LCS is currently being automated via the Automated List Control System alteration (ALCS 9145K). The ALCS provides remote control and monitoring of the aircraft carrier LCS from Damage Control Central (DCC) and Shaft Alley via a flat panel display. Four existing loop valves are upgraded from manual to motor operated and are integrated into the control system. Existing pump controllers are replaced and control of pumps, valves and tank level monitoring are integrated into a single system with redundant capabilities. The ALCS utilizes commercial-off-the-shelf components and Navy-owned software to decrease the manpower needed for operation and maintenance, increase automation, reduce life cycle costs and significantly improve system reliability. System line-up and tank management is automated and managed such that tanks can be filled faster, utilizing fewer watchstanders and increasing system efficiency. Control and monitoring of the LCS is performed from two human-machine interface (HMI) stations: one in DCC and one in Shaft Alley. The system can also be operated from the Central Control Station, and operates in both remote and semi-automatic modes.

Very little feedback has been obtained regarding the new ALCS and its improvements to overall ship operations. To date, installation has been completed on three of the ten planned carriers. Although the system provides the means to automatically operate pumps and valves, the capacity of the system has not been changed. To really affect the starboard inherent list, more ballasting is going to be required. Once the inherent starboard list is fixed, the ALCS will be extremely advantageous to overall carrier operations.

1.3.7 Ballast Addition

1.3.7.1 Water Ballast

Special preparation would be required for all tanks or voids using locked-in water ballast to correct the starboard inherent list. The sponson voids, in particular, are coated with epoxy over an inorganic zinc primer; this is suitable for installation of water ballast, but only for a period of up to five years if it is intact. Preparation of the sponson voids for installation of water would require inspection and possibly repair of the existing coating systems. Another problem with using water as a source of ballast is that it is difficult to provide a sufficient amount of water ballast in a space judicious manner in order to provide enough permanent list correction.

Although water is an inexpensive material, utilities would have to be relocated to other spaces and the tanks themselves cannot be partially filled.

1.3.7.2 Solid Ballast

For purposes of this project, lead was not considered as an option due to the tremendous handling expenses and associated carcinogenic problems. Instead, a pumpable slurry of iron-ore, known as Perma Ballast®, was examined as a possible source of solid ballast. Ballast Technologies Inc. (BTI) has been a provider and installer of Perma Ballast® since 1983. Their product is widely acknowledged to be the quickest and most cost-effective method of ballast installation. All materials, including the fluid used to install the ballast, are naturally occurring, nontoxic, non-corrosive and environmentally safe. No gases or vapors are generated and no special handling is required upon installation or removal.

Minimal vessel modification is required to the ship, thereby providing savings to the shipyard. Jobs are not generally subcontracted out, thereby cutting out the middle man and making this product more competitive. Prior to installation, engineers and key personnel inspect

the vessel to be ballasted and its location. Requirements such as electrical, water, compressed air, and equipment location are assessed. BTI engineers then submit engineered drawings noting location of equipment and diagrams of the installation system along with a written operating plan for the project. BTI uses its own experienced personnel and equipment during ballast installations to ensure safe, rapid and efficient mobilization, installation, and demobilization.

BTI's ballast materials are mixed with water and pumped to the vessel via a combination of rigid and flexible pipes. The slurry is pumped in at a controlled velocity in order to assure an even distribution of the ballast around projections, pipe, and other objects, leaving no voids. Excess water is removed as the ballast is installed and settles. This ensures that the in-place, fixed ballast is a dense mass which will not move or shift. Due to BTI's materials and placement method, no special handling or tank modifications are required.

BTI pretests materials in its laboratory to ensure proper density and uniformity.

Continuous testing is performed during ballast installation to verify installed density. Densities of materials range from 150lb/ft³ to 350lb/ft³. All processes and materials used by BTI are approved by ASTM, ABS, MARAD, U.S. Navy, U.S. Coast Guard, Del Norske Veritas, and Lloyds. In fact, Perma Ballast® has already been installed on a number of naval vessels, particularly DDG 73, 76, and 77. The reason for installation was to correct a Flight II starboard list. Feedback obtained from these installations described the process as being very smooth and extremely successful.

There are a number of advantages to using this Perma Ballast®. It is possible that Perma Ballast® will absorb shock over a limited area and serve to dampen it. Although the individual particles and the water are incompressible, the Perma Ballast® bed is made up of many very small particles which are not bound together by anything except gravity. There is approximately

2% entrained air (by volume) in the slurry. Some of this air probably remains in the slurry after settling so it would be fair to say that Perma Ballast® contains about 1-2% (by volume) compressible components. Water is removed as the material settles, but even after the Perma Ballast® settles, all of the voids between particles are filled by water. In fact, a settled bed of Perma Ballast® is an extremely effective barrier against permeation of oxygen - either into the bed or through it.

1.4 Option Selection

From the options presented above, the choice was made to use solid ballast to correct the starboard inherent list on the Nimitz class carriers. Based on ballast density, tank location and capacity, ease of ballast installation, minor tank structural modifications, and a decision making cost analysis, the Perma Ballast® appears to be the best option for use in correcting the inherent list on the Nimitz class aircraft carriers and is discussed extensively in Chapter 2.

2.0 Preliminary Analysis and Results

Preliminary analysis began with tank selection to serve as input to the Program of Ship Salvage and Engineering (POSSE) computer analysis program, discussed later in section 2.2. Once tanks were selected for ballasting, a naval architectural analysis using POSSE was performed for 36 scenarios to determine list corrections. These scenarios encompassed ballasting on the 2nd, 4th, and 8th decks with 200 lb/ft³ and 325 lb/ft³ density ballast. Fresh water was also examined as a possible alternative, but not as extensively as the solid ballast option. Once a complete set of data was obtained, a decision making cost analysis was performed. This cost estimation analysis proved that the 200 lb/ft³ density ballast was the best option. As a result, all structural analysis was performed using the 200 lb/ft³ density ballast. Once the structural analysis was complete, modeling had to be performed again using POSSE. In some cases, a

minimum ballast weight was calculated for a particular set of tanks for the prescribed design criteria. Analysis had to then be performed again in POSSE with these new ballast weights. Similarly, some of the structural analysis revealed that stiffening was required for plating in some of the tank bulkheads. This required another cost assessment be performed in order to achieve accurate cost estimations. So, one can see that this project followed a design spiral towards to determining the most cost effective and advantageous solid ballasting solution for correcting the inherent list on the Nimitz class carriers.

2.1 Tank Selection

Tanks selection was based on location and those that were available for ballasting based on Nimitz class carrier drawings. As mentioned previously, ballasting was accomplished to relieve up to 3 degrees of list in half degree increments. 30 tanks, primarily in the aft/port voids, have been identified to accommodate the installation of Perma Ballast® with 200 lb/ft³ and 325 lb/ft³. There are 3 tanks or sponson voids on the 2nd deck, 16 tanks on the 4th deck, and 11 tanks on the 8th deck. The methodology utilized was as follows: to correct the most list possible using the least weight possible. For example, tanks could have been chosen in the order of fill that were further aft to assist in trim correction, but that does not provide the best list correction with the least weight addition. Trim was, however, taken into consideration such that any addition of ballast would not serve to increase the existing trim condition. As a result, all changes in trim are also closely monitored.

Included in Appendix A is a series of tables providing information on the effects that each tank would produce if the tank was individually filled to capacity in a combat load condition with fresh water as well as each of the two ballast density types, 200 lb/ft³ and 325 lb/ft³. This information includes:

- Tank name
- Tank volume
- Ballast type density
- Tank weight
- Initial KG
- Initial trim
- Initial weight
- Final KG
- Final trim
- Final weight
- Δ KG
- Δ trim
- Δ weight

2.2 Modeling Analysis Performed

POSSE was used to analyze the inherent list for the Nimitz class carrier. POSSE is a software package of modeling, naval architectural design tools, and intact loading and salvage analysis tools designed primarily for U.S. Navy salvage response. The Modeling and Naval Architecture modules are Microsoft DOS® applications packaged through a Windows interface called the Ship Project Editor. A plan is then developed so that the engineer can evaluate the ship in various conditions. The collection of conditions represents the steps in the plan to assess the status of the vessel. POSSE provides an efficient means to develop plans using a tree structure to allow a hierarchical definition that permits branching to investigate various potential solutions. Several branches can be developed and viewed concurrently. A condition represents a

particular state of the vessel. It includes all load and strength information associated with that state. For purposes of this thesis, only intact states were analyzed.

Aircraft carrier modeling in POSSE was too cumbersome to analyze for the original DOS version. In fact, any analysis prior to this thesis would have to been done in sections. With POSSE 4.0, the complete Microsoft Windows® version, the sections were able to be combined into a single ship project (.shp) file using the ship project editor. Once the CVN 68 Nimitz Class hull was put together, a weight ordinate was entered to adjust the lightship data so that it coincided with the most recent data available. It has been very difficult to determine the exact inherent list that each carrier exhibits when under combat load conditions due to the "unknown light ship (LS) growth" associated with each carrier and due to the lack of in-service inclining experiments performed on Nimitz class carriers. Unknown LS growth is used to term unattributable changes in displacement and centers of gravity over the ship's service life. Examples of unknown LS growth include but are not limited to: unauthorized alterations or installations of equipment, accumulations of paint, deck covering, dead cable runs, old foundations, undocumented configuration changes affecting weight and KG, and excess and obsolete repair parts, technical manuals, and paperwork. Naval Sea Systems Command's Weights and Stability Division has recently revised the unknown light ship (LS) weight and KG growth data for CVN 71 through CVN 75 based on re-evaluation of the results of the CVN 71 Actual Operating Condition (AOC) Weight Survey and Displacement Test and the CVN 68 Post-Refueling Complex Overhaul (RCOH) inclining experiment.

The CVN 68 inclining experiment was the first in-service inclining experiment of the NIMITZ class in 27 years. The CVN 71 displacement test determined the ships displacement and longitudinal, transverse but not vertical center of gravity. The CVN 68 post RCOH inclining

was successfully performed and the results are considered to be accurate. The results of the inclining show that the ship had an unknown lightship weight growth of 1,818 ltons and 0.47 feet of KG increase. The fully loaded condition was determined to have a displacement of 100,019 ltons with a KG of 47.21 feet. This provides 3,781 ltons (3.78%) and 1.29 feet KG service life allowance remaining when compared to the limiting displacement of 103,800 ltons and a KG comparator value of 48.50 feet.

The following changes were made to all stability baselines for CVN 71-75:

• CVN 71

o Increase the current LS vertical center of gravity by 0.25 feet to account for the unknown LS KG growth.

CVN 72-75

- o Revise the unknown LS weight growth to 2,333 long tons.
- o Locate the unknown LS weight growth at the following centers of gravity:
 - VCG = 60.5 ft
 - LCG = 72.47 ft (A)
 - TCG = 5.62 ft (P)

As a result of this unknown growth, an average Nimitz class carrier baseline condition was modeled in POSSE and used for all analyses. Lightship and Combat Load Condition data was obtained from the CVN 72 Weight and Moment Baseline Report. This report is based on:

- 1) CVN 68 post RCOH inclining experiment load out,
- 2) CVN 71 Actual Operating Condition (AOC) results,
- 3) CVN 73 Aircraft/JP-5 Revisions,
- 4) Scheduled Restricted Availabilities (SRAs) actually accomplished,

- 5) the latest estimates for next overhaul,
- 6) the latest Availability FY01 Planned Incremental Availability (PIA), and
- 7) LCS at 50%.

A Lightship Condition of 81,450.69 ltons and a Combat Load Condition of 104,263.10 ltons were obtained from the CVN 72 Weight and Moment Baseline report and used for modeling purposes. These values are from before the latest correction for unknown growth was applied. Included in Table 1 are the current CVN class predictions with the most recent unknown growth corrections applied. This is an extremely important chart because it shows each carrier's delivery data and class predictions for displacement, KG, and list.

The VCG, LCG, TCG, and volume of each tank being analyzed was then checked and modified if necessary to ensure accurate stability and analytical results. These naval architecture parameters were checked against the compartment and access database for CVN 70. Normally, one would use the ship's lines drawings and develop the hull up to the bulkhead and freeboard decks (i.e. the highest level to which there will be watertight/structural salvage operations). On the basis of this modeling, one checks the hydrostatics and curves of form against the builder's documents to determine if the hull has been modeled correctly. Any freely flooding area above the freeboard deck will generally be left off the hull model unless the deck is a strength deck (i.e. part of hull girder bending). The 2nd deck sponson voids had to be added to the model because they were not included previously and POSSE is normally only utilized to model what is inside the ship's hull. Builder's lines drawings were not used, however, because salvage and hydrostatic work was not needed for purposes of this project. Instead, weights and centers were input to model the CVN 72 data to determine and correct the ship's list. After careful accounting

of each tank's location was added, verified, or corrected, the file was imported into the POSSE program so that a combat load could be applied.

The combat load condition was then modified to reflect the current CVN 72 baseline combat load data mentioned previously as constants in the POSSE plan that was newly created. This includes a correction for:

- 1) crew and effects,
- 2) aviation ammunition,
- 3) ship ammunition,
- 4) provisions and stores,
- 5) general stores,
- 6) lube oil,
- 7) potable water,
- 8) reserve and emergency feed water,
- 9) oxygen and nitrogen production plant,
- 10) JP-5 aviation fuel,
- 11) gasoline,
- 12) aircraft,
- 13) aviation yellow gear,
- 14) aviation lube oil,
- 15) onboard discharge storage tanks, and
- 16) bilge and oily water.

Then, to reflect CVN 71 AOC results, a miscellaneous weight correction factor was added at the combat load condition TCG, VCG, and LCG. Now, the combat load condition was

accurate and ready for the creation of several variant conditions. Specifically, a condition for water ballast, high density ballast (325 lb/ft³), and low density ballast (200 lb/ft³) was created.

Included in Appendix B is a series of tables providing POSSE modeling results in a combat load condition for list correction in half degree increments up to 3 degrees for fresh water as well as each of the two ballast density types, 200 lb/ft³ and 325 lb/ft³. This information includes:

- Tank name
- Tank volume
- Ballast type density
- Tank weight

Degree Increment:

- Weight
- Percent change
- Δ KG
- Δ Trim

A 95% usable volume fraction was utilized for all calculations. This value provides for structural internals as well as for any utilities that may be running through the spaces. There are no losses due to compaction. According to the vendor, BTI, because the water is removed as the material settles, all of the voids between particles are filled by water and remain filled by water. It must be noted again, however, that a settled bed of Perma Ballast® is an extremely effective barrier against permeation of oxygen - either into the bed or through it.

2.3 Preliminary Decision-Making Cost Estimation

All decision making costing or "concept" estimation was performed by Norfolk Naval Shipyard's Structural Engineering and Planning Office (Code 256). This preliminary costing was performed prior to any structural analysis. It was presumed that one type of ballast or one location of ballast could be ruled out solely based on cost alone. As a result, a final cost analysis would still need to be performed after the structural analysis was completed. Cost estimation was performed for a combination of 36 scenarios on the 2nd, 4th, and 8th decks. A specific number of man-hours were allocated under each shop for jobs associated with the installation of Perma Ballast®. These jobs ranged from the opening and closing of voids to the preparing and painting of any additional structures. The cost estimation was broken down by man-hours and then converted to man-days. A total production cost was then derived from the man-days required for tank prepping the Perma Ballast® installation. Similarly, material production costs were calculated for prepping a typical void while additional material costs were additionally applied for canning plate installation on the 2nd deck. A canning plate, used to enclose only the required amount of ballast, would only be utilized in analysis of the 2nd deck because both the 4th and 8th decks are completely filled in all of the modeling scenarios. Any additional weight added due to the addition of internal structures or stiffening was also tabulated and tracked.

Because the 4th and 8th decks did not require canning plates, any costing associated with these decks involved limited material costs and fairly minor production costs. Much of the work associated with all the tanks includes:

- Opening and closing of tanks
- Touch-up painting
- Removal and reinstallation of accesses

- Testing of all accesses
- Removal and reinstallation of any interferences
- Patching and repair of deck coverings

It was assumed that the 2nd deck would require structural modifications due to thinner shell plating on the sponsons (compared to thicker hull shell plating found on the 8th deck voids).

Two sponson voids were analyzed in depth for cost estimation purposes, 2-165-8-V and 2-180-6-V. All production and material costs were based on prepping and work on the entire tank. As a result, the 2nd deck void costs are much higher than the 4th and 8th deck voids due to their size. Similarly, a thumb rule was applied to the 2nd deck voids when filled to 95% and no canning pate work was required. The total structural work, for both production and material costs, is half the cost of the canning plate work. These cost estimates do not take into account relocation of existing utilities that may be running through the spaces. An analysis was performed, however, that determined that ballasting these selected tanks would have little, if any impact on the tanks, and that relocation of utilities would probably not be required.

Table 2 is an example cost comparison of the preliminary cost estimation results found.

Production and material costs are the same regardless of the density of the ballast being used.

Table 2: Deck Location Cost Comparison

	2nd Deck		4th Deck	8th Deck	
	2-165-8-V	2-180-6-V	All Voids	All Voids	
Volume (ft ³)	3,969	6,273	approx. 600	approx. 1,000	
Production Costs	\$328,704	\$525,958	\$21,186	\$21,186	
Material Costs	\$22,126	\$35,402	\$500	\$500	
Weight Added (Iton)	4.71	7.54	0	0	

All further costing calculations for the 2nd deck voids less than 95% filled were then performed as a percentage of the entire tank. For example, if a 2nd deck sponson void was to be filled 22%, then the corresponding voids production and material costs were multiplied by 22%. This was allowable because of the nature of the geometry of the 2nd deck voids as shown in Figure 1. Sponson void 2-165-8-V runs 60 feet long and sponson void 2-180-6-V runs 96 feet long. A detailed production and material cost breakdown for the 36 options mentioned is located in Appendix C. The worksheets used to perform these cost estimations are also located in Appendix C. A complete preliminary cost comparison was then performed for the 2nd, 4th, and 8th decks. This comparison provides the following:

- Total Cost
 - o Production cost
 - o Material cost
 - o Perma Ballast® cost
- Total Weight (Ltons)
 - o Material Weight
 - o Perma Ballast® Weight
- Δ KG

This decision making cost comparison revealed that the 200 lb/ft³ density Perma Ballast® was much more economical than the 325 lb/ft³ density Perma Ballast®. It takes nearly the same amount of ballast to affect each incremental degree change on the same deck regardless if using the 200 lb/ft³ or 325 lb/ft³. Although the number of tanks required to ballast with 200 lb/ft³ was higher, it was still more cost effective to use the 200 lb/ft³ density ballast due to the price per lton of the 325 lb/ft³ ballast. The 325 lb/ft³ ballast has steel shot added to it to help achieve its high

density. It was also discovered at this early stage of analysis that for corrections at or beyond 1.5 degrees, the 2nd deck was a much better location for this ballast addition due to the lesser weight addition resulting from the larger moment arm generated by filling the 2nd deck voids. The costs associated with adding ballast to the 2nd deck is much higher, but the amount of ballast required is significantly less than that required for the 4th and 8th decks. For corrections less than 1.5 degrees, the 2nd deck and the 8th decks have a fairly comparable weight addition, however, ballasting the 8th deck tanks costs significantly less. The complete preliminary cost comparison for the 2nd, 4th, and 8th decks is found in Appendix D.

Adding ballast to the 2nd deck voids does increase KG and thus reduces the KG service life allowance margin; however, the increase seen is fairly small. In fact, as shown in Appendix D, a 1.5 degree list correction results in approximately a 0.1 percent KG increase. On average, the Nimitz class carriers currently have a KG of 47 feet, as shown in Table 1. The goal KG service life allowance is not to exceed 48.5 feet. Thus, to fix an inherent list of 1.5 degrees, there would be a .05 foot increase in KG.

2.4 Structural Analysis

One tank on each deck was chosen for structural analysis due to the geometrical similarities between tanks. Analysis was performed on any existing shell and/or deck plating, as well as all longitudinal and transverse bulkheads for the tank chosen from each deck. The volume of each tank was obtained from POSSE so that the weight of Perma Ballast® required to fill the entire tank could be calculated in Itons. The density used for all the Perma Ballast® calculations was 200 lb/ft³. All structural calculations are located in Appendix E.

All surface ships shall be designed to withstand a number of loading conditions including ship motion loads. Ship motion loads are defined by [6] as the inertia forces and gravity components resulting from the motion of the ship in a seaway. The ship motion factors for this analysis were obtained from Naval Sea Systems Command's Ship Survivability Division.

Although these factors will vary slightly depending on location, they were assumed to be constant. They are as follows:

Ship Motion Factors:

V := 1.25 Vertical A := 0.75 Athwartship

F := 0.4 Fore/Aft

Once the ship motion factors were applied to each direction respectively, the total normal force could be calculated. The force affecting the structural analysis was dependent on the location of the deck or bulkhead being analyzed. For example, the total force acting on the 2nd deck's sponson shell plating had to be resolved from both a vertical and athwartship force. The resulting pressure, or equivalent head, on the plating was then found using this resolved normal force. Each tank was based on a 95% usable volume to account for structural internals or utilities running through the space.

In accordance with [6], panels of plating shall be proportioned so as not to exceed the breadth-thickness ratios indicated by:

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

$$\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$$

C = coefficient that is a function of the plating material and the location of the plating on the ship

K = coefficient that depends on the aspect ratio (AR) of the panel

H = head of salt water (feet)

The coefficients C and K are provided in Table I of Section 100, the General Requirements for Hull Structure, in [6]. There are a number of key assumptions that were made following these design criteria for plate panels subjected to normal loads. The ship plating is divided up into any number of panels, depending on the amount of stiffening that exists. The panel is considered to have clamped or fixed ends and is subject to cylindrical bending, a plate that is subject to bending about one axis only (as usually occurs for long plates). The cross-section of this panel is considered to be rectangular in shape with a depth equal to the plating thickness and a unit width. The K coefficient is important for relatively short panels. The C coefficient is actually derived from standard beam theory for a fixed-fixed beam, by combining allowable stress, specific gravity of salt water, and this K coefficient. An example of a C coefficient derivation starting with beam theory can be seen at the end of the 2nd deck sponson shell plating structural calculations shown in Appendix E.

Also important to consider in the structural calculations, particularly for the 2nd deck sponson shell plating, was the maximum shell plating pressure. According to CVN 76 specifications, the maximum pressure for sponson shell plating is 1000 psf. As a result, the limiting weight of Perma Ballast® for these tanks had to be calculated based on not exceeding

this pressure. This resulted in a reduction of approximately 55 ltons of ballast, for this particular case only. Similarly, the only structural analysis performed for the 4th deck void was on its deck plating. Once it was determined that deck stiffening would be required, the 4th deck tanks were taken out of consideration as a possible location for placing this ballast. Although adding a stiffener down the middle of the deck plating would alleviate any structural concerns, welding to the deck plating would have to be performed. With welding required on the bottom deck plating, a watch would need to be stationed on the other side of the deck. The tanks below these 4th deck tanks are foam filled and a fire watch cannot be stationed in these tanks.

Once a maximum pressure in head of water was calculated, the breadth-thickness ratio formula could be used again to determine the stiffener spacing or limiting short dimension of the panel. If the calculated maximum distance that the stiffeners can be located apart fell less than the actual stiffener spacing, then no plastic deformation would occur and stiffening of the plating was not required. If stiffening was required, then the panel thickness was usually divided in half to accommodate the calculated pressure.

The final calculations involved verifying that the calculated maximum allowable stress was less than the design allowable stress for the particular type of steel, regardless of whether any modifications were made. The basic equation used to calculate the maximum stress governing the behavior of plating under lateral load, or plate bending, can be seen below.

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$

Using small deflection theory, the value of the coefficient k depends on the boundary conditions. For simply supported edges, k=.75 and for clamped edges, k=.5. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On

board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. This equation is used for all plates whether they are long or not, and the coefficient k also accounts for the effect of the aspect ratio a/b. Again, b is the short dimension of the panel and t is the plating thickness. Figure 1 illustrates graphically the stresses in rectangular plates under uniform lateral pressure.

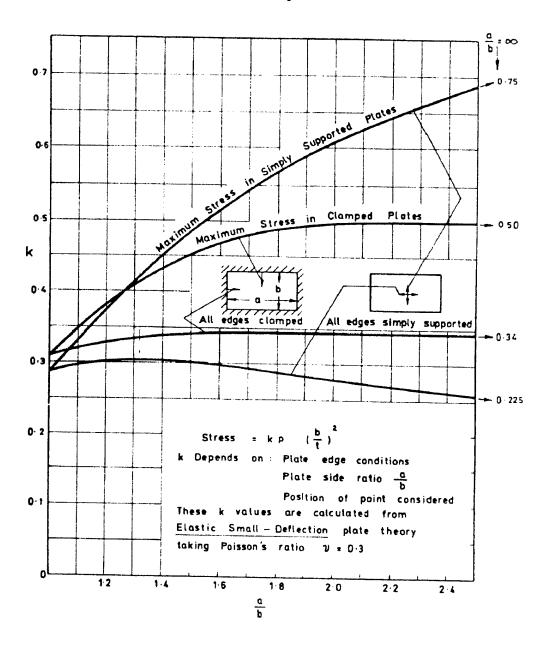


Figure 1: Stresses in rectangular plates under uniform lateral pressure [2]

2.4.1 Structural Modifications

2.4.1.1 Modifications to the 2nd Deck

There were actually three port sponson voids modeled in POSSE on the 2nd deck.

Due to existing geometrical similarities between the tanks, only compartment 2-165-8-V was analyzed for possible structural modifications. Basic dimensions and tank geometry can be seen below in Figure 2.

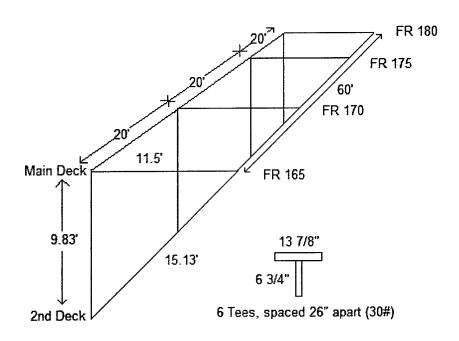


Figure 2: 2nd Deck Compartment 2-165-8-V

It was found that only 281 ltons of ballast could be added to this compartment in order not to exceed the 1000 psf sponson shell plating maximum pressure criteria. Table 3 shows the required structural modifications for compartment 2-165-8-V for a 281 lton ballast addition at 200 lb/ft³. All calculations are located in Appendix E.

Table 3: Required Structural Modifications for 2-165-8-V

Ship Structure	Structural Modifications
	Add 7 vertical stiffeners
Sponson Shell	(5" x 4" x 7.5# x T) (MS)
Plating	Max Panel Plating Size: 13" X 181.56"
Ship Shell Plating	
(Inner Longitudinal	
Bulkhead)	No modifications required
Transverse	
Bulkheads (Frames	
165, 170, 175, and	
180)	No modifications required

The sponson shell plating required the addition of 7 vertical stiffeners due to its thinner plating on the sponsons. The sponson shell plating is only 10.2# (.25") plate. The inner longitudinal bulkhead, which is really the ship shell plating, is 30.6# (.75") and the transverse bulkheads are 20.4# (.5"). Similarly, the transverse bulkheads are extensively stiffened, and the longitudinal stiffener has one large horizontal stiffener running down the center of the bulkhead. As a result, modifications to these bulkheads were not required. A cross-sectional view of the transverse bulkhead at Frame 165 is shown in Figure 3.

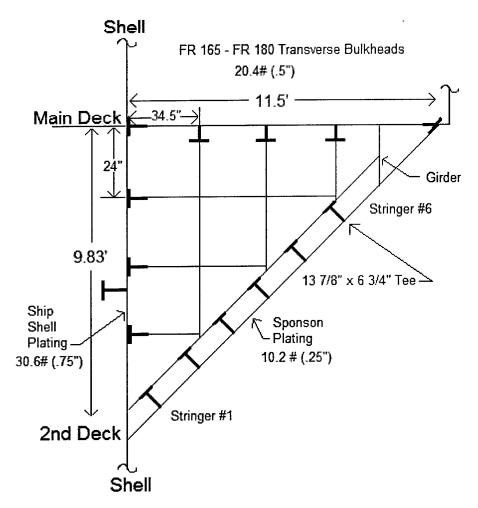


Figure 3: 2nd Deck Compartment 2-165-8-V, transverse section

2.4.1.2 Modifications to the 4th Deck

As mentioned previously, the only structural analysis performed for the 4th deck void was on its deck plating. Once it was determined that deck stiffening would be required, the 4th deck tanks were taken out of consideration as a possible location for placing this ballast. The 4th deck voids do not have any stiffened plating. Although adding a stiffener down the middle of the deck plating would alleviate any structural concerns, welding to the deck plating could not be performed due to the foam filled void below and the inability to station a fire watch there. Table 4, however, shows the structural modifications to the deck plating that would be necessary for

compartment 4-165-4-V for a 50 lton ballast addition at 200 lb/ft³. All calculations are located in Appendix E.

Table 4: Required Structural Modifications for 4-165-4-V

Ship Structure	Structural Modifications
	Add 1 longitudinal stiffener
	(5" x 4" x 7.5# x T) (MS)
	(5" x 4" x 7.5# x T) (MS) Max Panel Plating Size: 24" X
Deck Plating	240"

There were sixteen port voids modeled in POSSE on the 4th deck. Due to existing geometrical similarities between the tanks, only compartment 4-165-4-V was analyzed for possible structural modifications. Basic dimensions and tank geometry can be seen below in Figure 4.

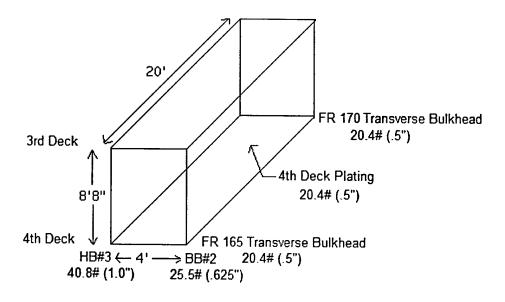


Figure 4: 4th Deck Compartment 4-165-4-V

2.4.1.3 Modifications to the 8th Deck

There were eleven port voids modeled in POSSE on the 8th deck. Due to existing geometrical similarities between the tanks, only compartment 8-225-6-V was analyzed for possible structural modifications. Basic dimensions and tank geometry can be seen below in Figure 5.

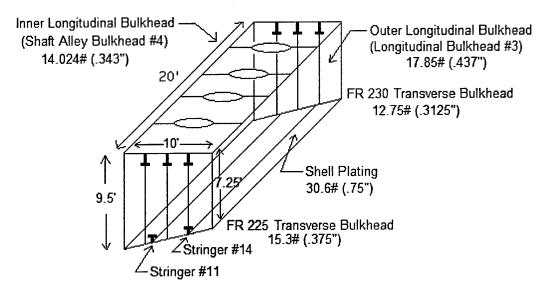


Figure 5: 8th Deck Compartment 8-225-6-V

There is considerable stiffening already in all the 8th deck voids. It was determined, however, that the plating thickness for these voids is much thinner than initially estimated. As a result, most of the bulkheads require stiffening. Structural analysis was first performed on the shell plating. A minimum ballast weight of 227 ltons was calculated so that structural modifications would not be required on the ship shell plating. This weight, however, is still too high for the rest of the bulkheads, as the other bulkheads do require stiffening. It is recommended that a stiffener be placed between each existing stiffener on the longitudinal and transverse bulkheads. Table 5 shows the required structural modifications for compartment 8-225-6-V for a 227 lton ballast addition at 200 lb/ft³. All calculations are located in Appendix E.

Table 5: Required Structural Modifications for 8-225-6-V

Ship Structure	Structural Modifications
G. 11 D	
Shell Plating	No modifications required
Inner Longitudinal	Add 5 vertical stiffeners
Bulkhead (Shaft Alley	(5" x 4" x 7.5# x T) (MS)
Bulkhead #4)	Max Panel Plating Size: 24" X 114"
Outer Longitudinal	Add 5 vertical stiffeners
Bulkhead (Bulkhead	(5" x 4" x 7.5# x T) (MS)
#3)	Max Panel Plating Size: 24" X 87"
	Add 4 vertical stiffeners
	(5" x 4" x 7.5# x T) (MS)
	Max Panel Plating Size: 15" X 114"
Transverse Bulkhead	*Note-panel height is dependent on
(Frame 225)	trapezoidal geometry
	Add 4 vertical stiffeners
	(5" x 4" x 7.5# x T) (MS)
	Max Panel Plating Size: 15" X 114"
Transverse Bulkhead	*Note-panel height is dependent on
(Frame 230)	trapezoidal geometry

3.0 Final Analysis and Results

This project design spiraled toward finding the most cost effective and advantageous solid ballasting solution for correcting the inherent list on the Nimitz class carriers. The preliminary decision-making cost estimation proved that the 200 lb/ft³ density was the most cost effective for the same amount of ballast weight. The 325 lb/ft³ ballast was so much more expensive, that although the number of tanks required to ballast with 200 lb/ft³ was higher, it was still more cost effective to use the 200 lb/ft³ density ballast. Once the structural analysis was complete, the minimum ballasting weights that were found had to be analyzed in POSSE again. For this analysis, the 4th deck was eliminated completely due to its structural limitations. Similarly, any required structural modifications that were not already included in the cost estimations had to be taken in to account for both the 2nd and 8th decks.

3.1 Final POSSE Results

The minimum amount of 200 lb/ft³ ballast that could be added to compartment 2-165-8-V was found to be 281 ltons which corresponds to 79% of that compartment's total capacity. Up until this point, all tanks had been analyzed to 95% of total capacity due to structural internals or possible utilities running through the space. Structural analysis was then performed for compartment 2-180-6-V. A minimum ballast weight of 448 ltons was calculated so that the maximum pressure design criterion of 1000 psf was not violated. Because less ballast could be added to these tanks, the POSSE analysis had to be expanded to include compartment 2-250-4-V. These new results for the 2nd deck had little effect on KG and only enhanced the aft trim correction.

The minimum amount of 200 lb/ft³ ballast that could be added to compartment 8-225-6-V was 227 ltons. This was calculated to avoid performing structural modifications on the ship shell plating. As a result, only 90% of that compartment's total capacity could be utilized which was 5% less than originally analyzed in POSSE. This time, rather than perform a structural analysis on all the 8th deck voids, a maximum capacity thumb rule of 90% was applied to all the 8th deck voids. As a result, different tanks were selected and analyzed to achieve the least lton addition for the same incremental change in degrees. Again, this had little effect on KG and change in trim.

A table providing POSSE modeling results in a combat load condition for list correction in half degree increments up to 3 degrees for 200 lb/ft³ is included in Appendix F. This information includes:

- Tank name
- Tank volume

- Ballast type density
- Tank weight

Degree Increment:

- Weight
- Percent change
- Δ KG
- Δ Trim

3.2 Final Cost Estimation

Once the POSSE modeling analysis was complete, a final cost assessment was made based on the new results for list correction in half degree increments up to 3 degrees for the 2nd and 8th deck tanks. Because the 8th deck tanks now had to be stiffened, the material costs and material weights had to be determined. The production costs were assumed to be the same. This information is included in Appendix G. Also, analysis on the 2nd deck revealed that compartment 2-250-2-V had to be used when trying to correct 3 degrees of list. Code 256 provided production and material costs as well as the material weights for tank prepping the Perma Ballast® installation.

All cost estimations were calculated and derived as previously discussed in the preliminary cost estimation section. This time, however, 2nd deck tanks were based on approximately 80% fillable capacity and 8th deck tanks were based on approximately 90% fillable capacity. When filled to these capacities, canning plate work was not required. Similarly, any additional weight added due to the addition of internal structures or stiffening was again tabulated and tracked. Table 6 shows the total cost, total weight addition, and change in KG for each half degree increment on both the 2nd and 8th decks.

Table 6: Final Cost, Weight Addition, and Change in KG Comparison

List (Degrees)	0.	5	1		1.	.5
Ballast Density	(200 11	os/ft³)	(200 11	os/ft³)	(2001	bs/ft ³)
Tank Location	2nd Deck	8th Deck	2nd Deck	8th Deck	2nd Deck	8th Deck
Production Cost (\$)	115,046	42,372	243,240	84,744	280,062	127,116
Material Cost (\$)	7,744	8,850	16,373	17,700	18,851	26,550
Perma Cost (\$)**	127,900	139,825	158,950	186,400	190,675	227,125
Total Cost (\$)	250,690	191,047	418,563	288,844	489,588	380,791
Material Weight (Ltons)	1.65	0.80	3.49	1.61	4.01	2.41
Perma Weight (Ltons)	124	177	262	384	403	565
Total Weight (Ltons)	125.65	177.80	265.49	385.61	407.01	567.41
Δ KG (ft)**	+0.02	-0.02	+0.04	-0.05	+0.05	-0.08

^{**}Perma cost estimates based on \$100,000 + \$225/lt for 200 lbs/ft^3

^{**(-)} indicates decreasing KG which is a positive effect

List (Degrees)	2	,	2.	5		3
Ballast Density	(200 11	os/ft ³)	(200 1)	bs/ft ³)	(2001	bs/ft ³)
•	2nd	8th	2nd	8th	2nd	8th
Tank Location	Deck	Deck	Deck	Deck	Deck	Deck
Production Cost (\$)	411,552	148,302	427,331	211,860	478,840	233,046
Material Cost (\$)	27,701	30,975	28,764	44,250	32,231	48,675
Perma Cost (\$)**	222,175	295,075	253,675	341,200	288,775	392,050
Total Cost (\$)	661,428	474,352	709,770	597,310	799,846	673,771
Material Weight						
(Ltons)	5.90	2.81	6.13	4.02	6.86	4.42
Perma Weight (Ltons)	543	867	683	1,072	839	1,298
Total Weight (Ltons)	548.90	869.81	689.13	1,076.02	845.86	1,302.42
						•
Δ KG (ft)**	+0.07	-0.13	+0.09	-0.14	+0.11	-0.17

^{**}Perma cost estimates based on \$100,000 + \$225/lt for 200 lbs/ft^3

^{**(+)} indicates increasing KG which is a negative effect

^{**(+)} indicates increasing KG which is a negative effect

^{**(-)} indicates decreasing KG which is a positive effect

3.2.1 Ease of Removal

As mentioned previously, Perma Ballast® is removable. In fact, removal costs were assessed as well for this project on a per tank basis. For the purposes of this thesis, all tank prepping costs for ballast removal by BTI were also calculated by Norfolk Naval Shipyard (Code 256). It was estimated that it would cost approximately \$78,652 for labor and material to remove/reinstall accesses and interferences for one 8th deck tank, and approximately \$63,412 for labor and material to remove/reinstall accesses for one 2nd deck tank.

Ballast removal costs were also obtained from BTI. It was determined that it would be best for BTI to actually remove the ballast and dispose of it due to the high costs involved should a naval shipyard need to dispose of the ballast. For ballast removal, BTI will charge a fixed fee of \$100,000 for locating and utilizing their equipment and approximately \$395 to remove the ballast per lton. They also estimated approximately \$65 per lton to dispose of it. All pricing is based on disposal of 100% of the water used to re-slurry the material.

For example, a 1.5 degree list correction would require approximately 400 lton of ballast and would need to be removed from two 2nd deck tanks. This would result in a total removal cost of \$347,412 of which \$284,000 is for the ballast removal and \$63,412 is for removing and reinstalling accesses. It should be noted that this ballast addition is only as permanent a solution as it needs to be. For example, should future modifications entail replacing the old Nimitz island with a new CVN 76 island, it would then be desirable and cost effective to remove this ballast.

3.2.2 Evidence Perma Ballast® is non-corrosive

Recently, the American Bureau of Shipping (ABS) required a sealift conversion vessel that installed this Perma Ballast® in 1984 to remove its ballast. Concern had arisen because a tank inspection revealed that corrosion coupons had undergone significant weight loss.

Corrosion coupons are placed in all tanks that have Perma Ballast® installed so that periodic tank inspections can be made to determine if corrosion exists. It turned out that the corrosion coupons were improperly attached to the manhole covers and only rested in the tops of the beds of Perma Ballast®. The corrosion coupons should be hung on chains and buried within the bed of ballast. Over the years, the coupons were repeatedly exposed to air and water during inspections, rather than being properly re-installed into the Perma Ballast®. ABS did report, however, that all 4 of the tanks were in perfect condition after the Perma Ballast® removal, almost 20 years after initial installation. Actual cost data was also obtained from this vessel. The cost to cut into 8 compartments and remove the ballast was approximately \$250,000. The cost to replace the steel and reweld the hull after the ballast was reinstalled was approximately \$100,000. As one can see, the removal costs are relatively low and experience proves that corrosion will not be a concern.

4.0 Conclusion

Solid ballast is the most advantageous solution for correcting the inherent list on the Nimitz class aircraft carriers. Table 6 illustrates that there are a number of advantages and disadvantages that must be considered when determining the proper location for the Perma Ballast®.

- The 2nd deck is the choice location for list corrections at or beyond 1.5 degrees. The costs associated with adding ballast to the 2nd deck is much higher, but the amount of ballast required is significantly less than that required for the 4th and 8th decks.
 - o Advantages:
 - significantly less weight added
 - o Disadvantages:
 - more costly
 - increased KG*
- For list corrections less than 1.5 degrees, the choice location is the 8th deck. Less than 1.5 degrees, the 2nd and 8th decks have a fairly comparable weight addition, however, ballasting the 8th deck tanks costs significantly less.
 - o Advantages:
 - significantly less costly
 - decreased KG*
 - o Disadvantages:
 - increased weight addition

^{*}Any positive change in KG serves to reduce the remaining KG service life allowance. The average KG service life allowance remaining for most carriers is approximately 1.5 feet. Even the greatest list correction of 3 degrees is only a 7% reduction in the remaining KG service life allowance for the aircraft carriers.

5.0 Recommendations

- 1. As can be seen in Table 1, most of the Nimitz class aircraft carriers currently have an average list of approximately 1.5 degrees. It is recommended that the ships be ballasted on the 2nd deck at this point to avoid adding excess ballast as mentioned previously. The cost is approximately 22% higher, but the weight addition is approximately 28% less. This is a considerable amount of weight reduction. The displacement service life allowance must also be analyzed. If you assume a contract displacement of 100,000 long tons with a 5% service life displacement allowance for an in-service carrier, then 400 long tons to correct 1.5 degrees of inherent list would be less than 1/10 of the service life (100,000 * .05 = 5,000 ltons and 400 / 5000 is approx. 8%). This can be viewed as a significant change in displacement or a relatively small change in displacement when looking at all the advantages that come with correcting the inherent list on the Nimitz class carriers. Also, the cost per lton for a 1.5 degree list correction is approximately \$1,200. This is truly inexpensive from a survivability standpoint and also reduces the operational constraints placed on the ship.
- 2. The prudent manner in which to handle the inherent list associated with each aircraft carrier is to perform an inclining experiment on each carrier. The inclining experiment data can then be used to assess the amount of ballast necessary for installation and can be conducted when each ship's operational schedule will permit it. As mentioned previously, much of the data for each carrier has been extrapolated from re-evaluation of the results of the CVN 71 Displacement Test and the CVN 68 Post-RCOH inclining experiment. Knowing that these tests are extremely expensive and that CVN 71-75 are the carriers in need of list correction, perhaps an inclining experiment could be performed on one of these carriers so that data could then be extrapolated further.

3. Until this point, there has been little mention of what reducing the inherent list could do for manpower reduction. The number of man-hours that could be saved due to fixing this problem could far outweigh anything discussed in this project so far. This is certainly something that should be studied and investigated as well.

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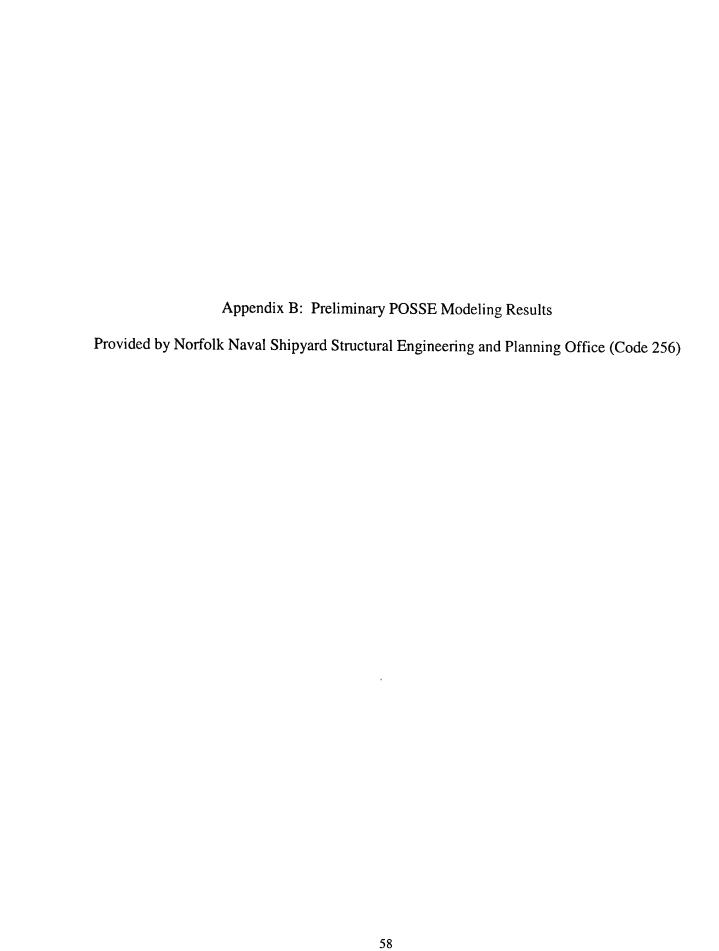
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Appendix A: Tank Location Study

Tanks	Car	Capacity (325 lbs/ft³)	ft³)			<u></u>	Intact Trim and Stability	nd Stab	ility			
	Volume (ft³)	Density (LT/ft³)	Weight (LT)	Initial KG	Initial Trim	Initial List	Final KG	⊲ 🎖	Final Trim	∆ Trim	Final List	∆ List
						(Starboard)						
2nd Deck												
2-165-8-V	3,969	0.1451	576	46.73	2.86	3.0	46.80	0.07	2.55	0.31	1.0	2.0
2-180-6-V	6,273	0.1451	910	46.73	2.86	3.0	46.85	0.12	2.02	0.84	-0.1	3.1
2-250-4-V	3,362	0.1451	488	46.73	2.86	3.0	46.79	90.0	1.80	1.06	1.6	1.4
4th Deck												
4-96-2-V	544	0.1451	62	46.73	2.86	3.0	46.73	0.00	2.94	-0.08	2.8	0.2
4-100-4-V	544	0.1451	- 29	46.73	2.86	3.0	46.73	0.00	2.93	-0.07	2.8	0.2
4-104-2-V	544	0.1451	62	46.73	2.86	3.0	46.73	0.00	2.93	-0.07	2.8	0.5
4-108-2-V	629	0.1451	66	46.73	2.86	3.0	46.73	0.00	2.93	-0.07	2.7	0.3
4-113-4-V	089	0.1451	66	46.73	2.86	3.0	46.73	0.00	2.92	-0.06	2.7	0.3
4-118-2-V	680	0.1451	66	46.73	2.86	3.0	46.73	0.00	2.91	-0.05	2.7	0.3
4-123-6-V	089	0.1451	66	46.73	2.86	3.0	46.73	0.00	2.90	-0.04	2.7	0.3
4-148-2-V	544	0.1451	62	46.73	2.86	3.0	46.73	0.00	2.86	0.00	2.8	0.2
4-152-2-V	544	0.1451	62	46.73	2.86	3.0	46.73	0.00	2.85	0.01	2.8	0.2
4-156-2-V	553	0.1451	80	46.73	2.86	3.0	46.73	0.00	2.84	0.05	2.8	0.2
4-160-2-V	717	0.1451	104	46.73	2.86	3.0	46.73	0.00	2.83	0.03	2.7	0.3
4-165-4-V	200	0.1451	102	46.73	2.86	3.0	46.73	0.00	2.82	0.04	2.7	0.3
4-170-2-V	678	0.1451	86	46.73	2.86	3.0	46.73	0.00	2.81	0.05	2.7	0.3
4-175-6-V	680	0.1451	66	46.73	2.86	3.0	46.73	0.00	2.80	90.0	2.7	0.3
4-180-6-V	681	0.1451	66	46.73	2.86	3.0	46.73	0.00	2.79	0.07	2.7	0.3
4-185-4-V	999	0.1451	97	46.73	2.86	3.0	46.73	0.00	2.78	0.08	2.7	0.3
8th Deck												
8-210-10-V	1,716	0.1451	249	46.73	2.86	3.0	46.70	-0.03	2.53	0.33	2.3	0.7
8-215-8-V	1,238	0.1451	180	46.73	2.86	3.0	46.71	-0.02	2.61	0.25	2.6	0.4
8-215-10-V	1,187	0.1451	172	46.73	2.86	3.0	46.71	-0.02	2.62	0.24	2.5	0.5
8-220-8-V	1,019	0.1451	148	46.73	2.86	3.0	46.71	-0.02	2.64	0.22	2.6	0.4
8-220-10-V	737	0.1451	107	46.73	2.86	3.0	46.72	-0.01	2.70	0.16	2.7	0.3
8-225-6-V	2,814	0.1451	408	46.73	2.86	3.0	46.67	-0.06	2.20	99.0	2.2	0.8
8-225-8-V	1,145	0.1451	166	46.73	2.86	3.0	46.71	-0.02	2.59	0.27	2.6	0.4
8-230-4-V	2,096	0.1451	304	46.73	2.86	3.0	46.69	-0.04	2.34	0.52	2.4	9.0
8-230-6-V	646	0.1451	94	46.73	2.86	3.0	46.72	-0.01	2.70	0.16	2.8	0.2
8-235-6-V	1,491	0.1451	216	46.73	2.86	3.0	46.71	-0.02	2.47	0.39	2.6	0.4
8-235-8-V	2,073	0.1451	301	46.73	2.86	3.0	46.72	-0.01	2.31	0.55	2.3	0.7
Negative Ch	Negative Change in KG is downwards	downwards.		Change i	Change in Trim is Aft by filling tanks	y filling tanks		Cha	Change in List is Port of Centerline	Port of Cer	nterline	

Tanks	ပ္	Capacity (200 lbs/ft³)	ft³)			드	Intact Trim and Stability	and Stat	oility			
	Volume (ft³)	Density (LT/ft³)	Weight (LT)	Initial KG	Initial Trim	Initial List	Final KG	⊲ &	Final Trim	△	Final List	□ □ □ □ □ □ □ □ □ □ □ □ □ □ □ □ □ □ □
						(Starboard)						
2nd Deck												
2-165-8-V	3,969	0.0894	354	46.73	2.86	3.0	46.78	0.05	2.67	0.19	1.8	1.2
2-180-6-V	6,273	0.0894	260	46.73	2.86	3.0	46.80	0.07	2.34	0.52	1.1	1.9
2-250-4-V	3,362	0.0894	300	46.73	2.86	3.0	46.77	0.04	2.21	0.65	2.1	6.0
4th Deck												
4-96-2-V	544	0.0894	49	46.73	2.86	3.0	46.73	0.00	2.91	-0.05	2.9	0.1
4-100-4-V	544	0.0894	49	46.73	2.86	3.0	46.73	0.00	2.91	-0.05	2.9	0.1
4-104-2-V	544	0.0894	49	46.73	2.86	3.0	46.73	0.00	2.90	-0.04	2.9	0.1
4-108-2-V	629	0.0894	61	46.73	2.86	3.0	46.73	0.00	2.91	-0.05	2.8	0.2
4-113-4-V	089	0.0894	61	46.73	2.86	3.0	46.73	0.00	2.90	-0.04	2.8	0.2
4-118-2-V	680	0.0894	61	46.73	2.86	3.0	46.73	0.00	2.89	-0.03	2.8	0.2
4-123-6-V	989	0.0894	61	46.73	2.86	3.0	46.73	0.00	2.89	-0.03	2.8	0.2
4-148-2-V	544	0.0894	49	46.73	2.86	3.0	46.73	0.00	2.86	0.00	2.8	0.2
4-152-2-V	544	0.0894	49	46.73	2.86	3.0	46.73	0.00	2.86	0.00	2.8	0.2
4-156-2-V	553	0.0894	49	46.73	2.86	3.0	46.73	0.00	2.85	0.01	2.8	0.2
4-160-2-V	717	0.0894	64	46.73	2.86	3.0	46.73	0.00	2.84	0.02	2.8	0.2
4-165-4-V	700	0.0894	62	46.73	2.86	3.0	46.73	0.00	2.84	0.05	2.8	0.2
4-170-2-V	678	0.0894	61	46.73	2.86	3.0	46.73	0.00	2.83	0.03	2.8	0.2
4-175-6-V	089	0.0894	61	46.73	2.86	3.0	46.73	0.00	2.82	0.04	2.8	0.2
4-180-6-V	681	0.0894	61	46.73	2.86	3.0	46.73	0.00	2.82	0.04	2.8	0.5
4-185-4-V	999	0.0894	09	46.73	2.86	3.0	46.73	0.00	2.81	0.05	2.8	0.2
8th Deck												
8-210-10-V	1,716	0.0894	153	46.73	2.86	3.0	46.71	0.05	2.66	0.20	2.6	0.4
8-215-8-V	1,238	0.0894	111	46.73	2.86	3.0	46.72	0.01	2.71	0.15	2.7	0.3
8-215-10-V	1,187	0.0894	106	46.73	2.86	3.0	46.72	0.01	2.71	0.15	2.7	0.3
8-220-8-V	1,019	0.0894	91	46.73	2.86	3.0	46.72	0.01	2.72	0.14	2.8	0.2
8-220-10-V	737	0.0894	99	46.73	2.86	3.0	46.73	0.00	2.76	0.10	2.8	0.5
8-225-6-V	2,814	0.0894	251	46.73	2.86	3.0	46.70	0.03	2.45	0.41	2.5	0.5
8-225-8-V	1,145	0.0894	102	46.73	2.86	3.0	46.72	0.01	2.70	0.16	2.7	0.3
8-230-4-V	2,096	0.0894	187	46.73	2.86	3.0	46.71	0.02	2.54	0.32	2.6	0.4
8-230-6-V	646	0.0894	58	46.73	2.86	3.0	46.73	0.00	2.76	0.10	2.8	0.5
8-235-6-V	1,491	0.0894	133	46.73	2.86	3.0	46.72	0.01	2.62	0.24	2.7	0.3
8-235-8-V	2,073	0.0894	185	46.73	2.86	3.0	46.72	0.01	2.52	0.34	2.6	0.4
Negative Cha	Negative Change in KG is downwards	downwards.		Change ir	Change in Trim is Aft by filling tanks	y filling tanks		Chan	Change in List is Port of Centerline	ort of Ce	nterline	

Tanks	Capa	Capacity (Water Ballast)	last)			In	Intact Trim and Stability	nd Stal	bility			
	Volume	Density	Weight	Initial KG	Initial Trim	Initial List	Final	⊲ <u>გ</u>	Final Trim	∆ Trim	Final	□ Tist
	\ \ !	,				(Starboard)						
2nd Deck												
2-165-8-V	3,969	0.0279	111	46.73	2.86	3.0	46.75	0.05	2.80	90.0	2.6	0.4
2-180-6-V	6,273	0.0279	175	46.73	2.86	3.0	46.76	0.03	2.70	0.16	2.4	9.0
2-250-4-V	3,362	0.0279	94	46.73	2.86	3.0	46.75	0.02	2.66	0.20	2.7	0.3
4th Deck												
4-96-2-V	544	0.0279	15	46.73	2.86	3.0	46.73	0.00	2.88	-0.02	2.9	0.1
4-100-4-V	544	0.0279	15	46.73	2.86	3.0	46.73	0.00	2.88	-0.02	2.9	0.1
4-104-2-V	544	0.0279	15	46.73	2.86	3.0	46.73	0.00	2.88	-0.02	2.9	0.1
4-108-2-V	629	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.88	-0.02	2.9	0.1
4-113-4-V	680	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.87	-0.01	2.9	0.1
4-118-2-V	680	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.87	-0.01	2.9	0.1
4-123-6-V	680	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.87	-0.01	2.9	0.1
4-148-2-V	544	0.0279	15	46.73	2.86	3.0	46.73	0.00	2.86	0.00	2.9	0.1
4-152-2-V	544	0.0279	15	46.73	2.86	3.0	46.73	0.00	2.86	0.00	2.9	0.1
4-156-2-V	553	0.0279	15	46.73	2.86	3.0	46.73	0.00	2.86	00.0	2.9	0.1
4-160-2-V	717	0.0279	20	46.73	2.86	3.0	46.73	0.00	2.86	00.0	2.9	0.1
4-165-4-V	700	0.0279	20	46.73	2.86	3.0	46.73	0.00	2.86	00'0	2.9	0.1
4-170-2-V	8/9	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.85	0.01	2.9	0.1
4-175-6-V	680	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.85	0.01	2.9	0.1
4-180-6-V	681	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.85	0.01	2.9	0.1
4-185-4-V	999	0.0279	19	46.73	2.86	3.0	46.73	0.00	2.85	0.01	2.9	0.1
8th Deck												-
8-210-10-V	1,716	0.0279	48	46.73	2.86	3.0	46.73	0.00	2.80	90.0	2.9	0.1
8-215-8-V	1,238	0.0279	35	46.73	2.86	3.0	46.73	0.00	2.81	0.05	2.9	0.1
8-215-10-V	1,187	0.0279	33	46.73	2.86	3.0	46.73	0.00	2.82	0.04	2.9	0.1
8-220-8-V	1,019	0.0279	28	46.73	2.86	3.0	46.73	0.00	2.82	0.04	2.9	0.1
8-220-10-V	737	0.0279	21	46.73	2.86	3.0	46.73	0.00	2.83	0.03	2.9	0.1
8-225-6-V	2,814	0.0279	79	46.73	2.86	3.0	46.72	0.01	2.74	0.12	2.8	0.2
8-225-8-V	1,145	0.0279	32	46.73	2.86	3.0	46.73	0.00	2.81	0.05	2.9	0.1
8-230-4-V	2,096	0.0279	58	46.73	2.86	3.0	46.73	0.00	2.76	0.10	2.9	0.1
8-230-6-V	646	0.0279	18	46.73	2.86	3.0	46.73	0.00	2.83	0.03	2.9	0.1
8-235-6-V	1,491	0.0279	42	46.73	2.86	3.0	46.73	0.00	2.79	0.07	2.9	0.1
8-235-8-V	2,073	0.0279	58	46.73	2.86	3.0	46.73	0.00	2.76	0.10	2.8	0.2
Negative Cha	Negative Change in KG is downwards	downwards.		Change	in Trim is Afi	Change in Trim is Aft by filling tanks	σ.		Change ii	n List is P	Change in List is Port of Centerline	erline



Tanks	3	Capacity (325 lbs/ft ³	F												Degrees											
					0.5				-		F		1.5			2				2.5		H		3		
	Volume (ft ³)	Density (LT/ft²)	Weight (LT	Weight (LT) Weight (LT)	Percent (%)	ΔKG	∆ Trim	Weight (LT)	Percent (%)	∆ KG	∆ Trim We	Weight (LT) P	Percent (%)	∆ KG ∆ Trim	rim Weight (LT)	T) Percent (%)	(%) A KG	G ∆ Trim	Weight (LT)	Percent (%)	∆ KG	∆ Trim	Weight (LT) P	Percent (%)	∆KG ∆	∆ Trim
											\dashv			\dashv		-	\dashv	\int							\dashv	Т
2nd Deck											+	1		+	-	-	\dashv	\int							+	
2-165-8-V	3,969	0.1451	276	127	22			285	46		+	403	2	+	547	8		-	547	ક્ક		1	247	88	+	Т
2-180-6-V		0.1451	910											-			-	1	136	22			523	8	-	
2-250-4-V		0.1451	488																							7
Total (LT)	l			127	6.4	000	A10		13.4	800	15.4	403	20.4	0.05 33A	3A 547	27.7	90:0	31A	889	34.6	600	44A	078	41.6	0.10	.58A
Number of Tanks				#181F				101				Tank		-	1 Tank	2000	\vdash	L	2 Tanks				2.Tanks		-	Г
						L								-	L		\vdash								H	
4th Deck											\vdash					_	_								H	Γ
4-96-2-V	54	0.1451	79								\vdash			\vdash		L	\vdash	L							┞	Г
4-100-4-V	₹	0.1451	79								_						\vdash								┝	Г
4-104-2-V	544	0.1451	79		_					L	\vdash			_		L	\vdash									
4-108-2-V	629	0.1451	88								\vdash			\vdash	_		\vdash				L					_
4-113-4-V	89	0.1451	83								\vdash			\vdash		_	-							8	\vdash	Γ
4-118-2-V	089	0.1451	83							L	<u> </u>			-		_	\vdash							ક્ક	┝	Γ
4-123.6-V	089	0.1451	83							L	H			\vdash			\vdash		೫				æ	83	H	Γ
4-148-2-V	544	0.1451	79	72	8	200		120	86		H	75	88		75	88	200	L	75				75	88	H	Γ
4-152-2-V	544	0.1451	g	2	- 38	Ĉ.		32	86			75	83		75	88			75				22	88	H	Γ
4.156.2.V	553	0.1451	8					92	ક		l	92	8	+	2	8		L	92	Ŕ	J.		92	8	┞	Γ
4-160-2-V	717	0.1451	Ş			Ĺ		8	8		ŀ	8	88	-	8	8		L	8				89	58	H	Γ
4.165.4.V	ω. Ε	0.1451	9		L	L				I	\dagger	8	8	+	ક	8	\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	L	8	10		T	8	8	t	Τ
4-170-2-V	829	0.1451	8									8.	8 8	1	S S.	88	1	L	35	S			8	88	t	Τ
4-175-6-V	589	0.1451	g		L	F					H			_	3	8		L	3.	10	1.3		75	8	 	Γ
4-180-6-V	884	0.1451	g			F					t		T	H	₹.	8	, S.	l	3.			L	85	8	t	
A-105.A-V	98	0.1451	8 8			F	I			T	t		T	ł	-	1	H	\mid	8	15	L		8	8	t	Π
Total a Ti	8	2	\$	93	10.0	WU	ACT O	305	200	N IS	NO.	R15		au au	174 7/13	6 LP	T	J100 218		L	A. P.	y	107	73.3	A) U	9,64
Total (L1)				3	١	3	4300	050	┸	2		010	T		91.0		t	Ξ.		L	3	ŝ	AN TOTAL	Т		3
Number of Faliks				S TOWN		$ \downarrow $				1		O SIERS		\dagger	0	2	+	$\frac{1}{1}$	ST IN THE		\perp		S I di III		\dagger	T
8th Deck										L	t					-	+								+	Τ
8-210-10-V	1,716	0.1451	549					237	ક્ક		\vdash	237	જ	\vdash	237	8	1000	L	237	88	L		237	88		
8-215-8-V	1,238	0.1451	180								H	171	-88		171	88	H		171	35			171	38	Н	
8-215-10-V	1,187	0.1451	172	164	38			161	- 38		Н	164	88		164	ස්	- 1 25		164	38	249		164	88		
8-220-8-V	1,019	0.1451	148												141	ಕ	0 T V.S		141	ક્ક			141	38	_	
8-220-10-V	737	0.1451	107								-						_		102	- 36			102	95		
8-225-6-V	2,814	0.1451	408								_			H			Н						388	- 36		
8-225-8-V	1,145	0.1451	166		L						Н			\vdash		H	Н		158	38						
8-230-4-V	2,096	0.1451	304															_								
8-230-6-V	646	0.1451	2 4												88	88							88	ક્ક		
8-235-6-V	1,491	0.1451	216														\dashv	_							1	٦
8-235-8-V	2,073	0.1451	န္တ					t				1	╗			\dashv					_			7		
Total (L.T)				164	7.0	-0.02	.24A		17.1	-0.05	.58A	572	24.3	-0.08	.83A 802	34.1		4011 12A	0.00	41.4	40.14	1.49A	1292	220	0.19	2.05A
Number of Tanks				1 Tank				*2Tanks		_		3 Tanks			5.Tanks	un.		_	6 Tanks				7 Tanks		_	
																										ĺ

Tanks		Capacity (200 lbs/ft ³)	(III)											أ	Degraps											Γ
	Ц				0.5				-		H		1.5			2				2.5		H		9	l	T
	Volume (ff*)	Volume (ft.) Density (LT/ft.) Weight (LT) Weight (LT)	Weight (LT)	$\overline{}$	Percent (%) A KG A Trim) AKG	∆Trim	Weight (LT)	Percent (%)	ΔKG	∆ Trim Weig	Weight (LT) Perce	ent (%) 1	⟨G ∆ Trir	Percent (%) A KG A Trim Weight (LT)	Percent (%) 1 KG) 4 KG	∆ Trim	Weight (LT)	Percent (%) A KG) 3 KG	ΔTrim	Weight (LT) Pe	Percent (%)	ΔKGΔ	ΔTrim
2nd Deck						\prod				1	+	+	\dagger	\downarrow			4				\int	1		\top	+	Т
2-165-8-V	3,969	0.0893	88	124	88	L		282	74	T		╀	88	\downarrow	337	89	1		337	s	$oldsymbol{\perp}$		337	8	\dagger	Τ
2-180-6-V	6,273	0.0893	299				Ĺ			T	L	88	=	L	303	æ	L		342	19	Ĺ		23	8	t	Τ
2-250-4-V	3,362	0.0893	88								L	}	-	L			L				ſ				t	Τ
Total (LT)	Ц			124	10.2	20.0	V/0	æ	21.6	100	:15A	L	32.8	35 SA	_	443	00	Æ	679	55.8	60 0	25	88	715	0.12	¥.
Number of Tanks				AMBIN SA				and I larke		-		2 Tanks	t	_	2 Tanks				2 Tanks				2 Tanks	†	_	ī
													-	L			L							T	t	Τ
4th Deck							L			I	L			L			L						I		\dagger	Τ
4-96-2-V	54	0.0833	49				Ĺ			T		L	L	L					9	ક્ક					\dagger	Τ
4-100-4-V	544	0.0833	49							T	L		H	-					\$	8					+	Τ
4-104-2-V	₹	0.0833	49							T	\vdash		F	\vdash			L		\$	ક્ક	L	l			\dagger	Τ
4-108-2-V	629	0.0833	150							T			H	L	25	L	L		57	89		L	l	t	t	Τ
4-113-4-V	089	0.0893	150				[H	L	L	\vdash	$oxed{\perp}$	57	L			22	88		$oxed{1}$		T	ł	T
4-118-2-V	089	0.0833	19							r		-	\mid	L	57	L	L		25	89	\prod	İ		T	+	Τ
4-123-6-V	089	0.0893	19							-	Ĺ		2	L	25	L			57	89		\perp	I	1	t	T
4-148-2-V	\$ 7	0.0833	49							\dagger	Ĺ	L	ير	L	46	L			\$	ક		\vdash		T	t	T
4-152-2-V	*	0.0893	8	\$	ક્ક		Γ	\$	89	T		L	, s	L	48	L	L	Ī	48	ક	I	\dagger			+	Т
4-156-2-V	553	0.0833	\$	47	89			1.5	89	t		L	عرا	Ļ		L	L		47	ક		f	\dagger	\dagger	+	Т
4-160-2-V	717	0.0833	B	150	88	L		19	88	t		L	90	\perp	9	L	L		120	3 8		$\Big $	$\frac{1}{1}$		$^{+}$	Т
4-165-4-V	902	0.0833	83					89	ક્ક	H			 		89	L		Ī	83	8		l		T	t	Τ
4-170-2-V	8/9	0.0893	61					88	ઝ	T	Ľ	89	ક્ક	L	25	ક્ક			88	83		I		T	╁	Т
4-175-6-V	88	0.0893	19					88	ક્ક		Ľ	L	مو		88				88	ક્ક		\mid			H	Τ
4-180-6-V	681	0.0833	19								Ĺ		مر		88	L		Γ	88	ક્ક		ig			╁	Т
4-185-4-V	999	0.0893								F			yp.	L	_	L		Ī	22	જ		\mid			-	Γ
Total (LT)				25	17.1	0.00	0.04A	828	36.4	001 0	0.14A 5-		9.0	13. ZA	_		9	111	88	ક્ક	ğ	76				Т
Number of Tanks				STEEL STEEL				September 1		H	ē	anks	H	Ц	13 Tanks				:16 Tanks						H	Π
										1	-	_													-	Г
Sth Deck										-	_	-	_											-	\vdash	_
8-210-10-V	1,716	0.0833	छ				1	146	ક્ક	\dashv		146	88		146	88			146	ક્ક		Н	146	88	Н	П
4215-8-V	877	0.0833					1	- 105	ક્ક	1		-	اء	_	105	ક્ક			55	ક્ક			105	ક્ક		
8-215-10-V	1,187	0.0833	<u>8</u>	ē	ક્ક		1	ē	ક્ક	\dashv		-	اء	\downarrow	ē	ક્ક			101	32		Н	101	ક્ક	Н	П
4-229-4-V	1019	0.0833	5				1				اند	\dashv	٥	_	98	ક્ક			æ	ક્ક			98	ક્ક		
8-220-10-V	737	0.0833	98	83	ક્ક		1			\dashv	۳	-	او		ន	ક્ક			83	ક્ક		Н	ಣ	ક્ક	H	
8-225-6-V	2,814	0.0893	331	1			1			1	-	$\frac{1}{2}$	-		239	ક્ક			239	88			539	ક્ક		
8-225-8-V	1,145	0.0833	52												26	ક્ક			62	ક્ક		-	97	ક્ક	-	Γ
8-230-4-V	2,036	0.0833	187											Ц					178	88		ig	178	ક્ક	H	Γ
8-230-6-V	98	0.0883	æ							H	2	55 89	5						55	88		-	55	ક્ક	┞	Г
8-235-6-V	1491	0.0833	133							Н			Н	Ц								\vdash	126	ક્ક	L	Γ
8-235-8-V	2,073	0.0893	П		П																	-	176	ક્ષ	H	Г
Total (LT)				臸	11.3	88	24A	362	24.3	-0.05	.51A 556	38.5	12 -0.08	8 0.85A	837	57.9	0.12	1.33A	1070	74.0	-0.15	1.75A	1372		-0.18 2.334	క్ట
Number of Tanks				2 Tanks		_	٦	S Tanks		\dashv	- 6T _k	age of	_	_	7 Tanks			příp-vý	9 Tanks			al l	11 Tanks		Н	П
																										l

Tanks	Car	Capacity (Water Ballast)	llast)						Degrees						
					0.4				1				1.3		
	Volume (ft³)	/olume (ft³) Density (LT/ft³)	Weight (LT)	Weight (LT)	Percent (%)	7 KG √	1 Trim	Veight (LT)	Percent (%)	∆ KG	∆ Trim	Weight (LT)	eight (LT) Percent (%) \triangle KG \triangle Trim Weight (LT) Percent (%) \triangle KG \triangle Trim Weight (LT) Percent (%) \triangle KG \triangle Trim	∠ KG	∆ Trim
2nd Deck															
2-165-8-V	3,969	0.0279	111	105	- 36			105	- 36			105	- 62		
2-180-6-V	6,273	0.0279	175					166	- 36			166	95		
2-250-4-V	3,362	0.0279	94									89	92		
Total (LT)				105	27.7	0.02 .06A	.06A	271	71.5	0.04 .22A	.22A	360	92	0.05	.42A
Number of ranks				Face falls and				#2 Tanks				3 Tanks			

		∆ Trim																		0.01F	
		∆ KG ∆																		-0.01	-
	9.0	Percent (%)		- 32	- 62	96	- 95	96	95	92	95	95	95	96	95.	- 32	6	~ 62	- 62	95	
Degrees		Weight (LT)		14	14	14	18	18	18	18	14	14	15	19	19	18	18	18	18	267	16 Table
Deg		∆ Trim																		A70.0	
		ру √																		00'0	
	0.5	Percent (%)									95	95	95	95	. 95	. 95	- 65	95	06	76.2	
		Weight (LT)					100				14	14	15	19	19	18	18	18	- 17	152	The state of the s
		Weight (LT)		15	15	15	19	19	19	19	15	15	15	20	20	19	19	19	19		
Capacity		Density (LT/ft³)		0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279	0.0279		
		Volume (ft²)		544	544	544	629	089	089	089	544	544	553	717	200	678	089	681	999		
Tanks			4th Deck	4-96-2-V	4-100-4-V	4-104-2-V	4-108-2-V	4-113-4-V	4-118-2-V	4-123-6-V	4-148-2-V	4-152-2-V	4-156-2-V	4-160-2-V	4-165-4-V	4-170-2-V	4-175-6-V	4-180-6-V	4-185-4-V	Total (LT)	Mumbor of Tanke

Tanks		Capacity					Deg	Degrees			
					0.5				+		
	Volume (ft ³)	olume (ft³) Density (LT/ft³)	Weight (LT)	Weight (LT)	Percent (%)	∆ KG	Δ Trim	△ KG △ Trim Weight (LT)	Percent (%) △ KG △ Trim	∆ KG	$\Delta \; Trim$
8th Deck											
8-210-10-V	1,716	0.0279	48	45	82			45	95		
8-215-8-V	1,238	0.0279	35	33	. 36			33	95		
8-215-10-V	1,187	0.0279	33	31				31	- 95		
8-220-8-V	1,019	0.0279	28	27	85			27			
8-220-10-V	737	0.0279	21					20	95		
8-225-6-V	2,814	0.0279	6/	75				75	95		
8-225-8-V	1,145	0.0279	32					30	- 35		
8-230-4-V	2,096	0.0279	58					56	95		
8-230-6-V	646	0.0279	18					17	- 35		
8-235-6-V	1,491	0.0279	42					40	95		
8-235-8-V	2,073	0.0279	28					55	95		
Total (LT)				211	46.9	-0.03	.33A	429	95	-0.05	.73A
Number of Tanks				5 Tanks				* 11 Tanks			

Appendix C: Preliminary Cost Estimation Data and Worksheets

ESTIMATORS NAME BACK

DATE SEPT 2003

ESTIMATE SHEET

			,						, ,					
75K A										912				
99 TS						48				48				
72 R1			:			260				260				
71 AA									256	256				
64 A6						120			384	504				
64 A1														
56 P1														
51 E1														
38 YY														
35 GA														
26 W1			112		120	714	99			1002				
17 SC														-
24 J1		4								4				
111 SS		10	32				16	48		106				
77 YY														
11 S6					30					30				
11 S2					09					09				
11 S1			96		120	969	48			098				
WORK TO BE ACCOMPLISHED	600 SF - CANNING PLATES FOR 2-165-8-V AT 100%	OPEN & CLOSE VOID	CUT/INST ACCESSES FOR CANNING PLTS	4 LOCATIONS	FAB CANNING PLTS & SUPPORT STR	INST CANNING PLTS & SUPPORT STR	CUT/INST ACCESSES (6) 12" FOR FILLING	TEST CANNING PLTS & ACCESSES	PREPARE AND PAINT STRUCT	TOTALS:	TOTAL 4096 MH'S TOTAL 512 MD'S	TOTAL PROD \$328,704	MATERIAL: \$22,126	ADD'L WEIGHT: 10,556 #'S
PC NO.														

MATERIAL WORK SHEET

PERMA BALLAST® INSTALLATION

SHOP	PC #	MATERIAL	STOCK NO.	QТУ	SS/PUR	COST
		CANNING PLATES: 600 SF AREA FOR 2-165-8-V AT 100% COVERAGE				
1151	001	10.2# OSS PLATE (10.2#/SF)	9515-00-153-3184	600	\$4.12/SF	\$2,472
1151	002	2-1/2 X 2-1/2" ANGLE (4.1#LF)	9520-00-277-4920	170	\$2.12/FT	\$360
		SPONSON STIFFENING:				
11S1	003	5" X 4" ANGLE (7.5#LF)	9520-00-277-5978	225	\$6.46/FT	\$1,454
71AA	004	PAINT				\$1,200
74YY	N/A	BLAST AND PAINT FACILITY				\$12,000
ALL	N/A	CONSUMABLES				\$4,640
		TOTAL MATERIAL COSTS:				\$22,126
		TOTAL MATERIAL WEIGHT:				10,556 #'S

ESTIMATORS NAME BACK

ESTIMATE SHEET

DATE SEPT 2003

1537 **75K** 92 8 416 72 R1 410 410 908 **A6** 614 192 4 45 41 41 56 P1 51 E1 35 GA 1142 1604 180 192 26 W1 8 SC 4 4 164 **S**5 92 26 10 52 74 Y **9**S 48 88 **S**2 96 8 1376 192 154 954 26 S1 OPEN & CLOSE VOID INST CANNING PLTS CUT/INST ACCESSES (6) 12" FOR FILLING TEST CANNING PLTS CUT/INST ACCESSES ADD'L WEIGHT: FOR CANNING PLTS FAB CANNING PLTS TOTAL 6554 MH'S TOTAL 819.25 MD'S **ACCOMPLISHED** 2-180-6-V AT 100% 960 SF - CANNING TOTAL PROD & SUPPORT STR & SUPPORT STR **WORK TO BE** MATERIAL: PAINT STRUCT PREPARE AND PLATES FOR 4 LOCATIONS & ACCESSES 16,890 #'S TOTALS: \$525,958 \$35,402 S S \mathbb{R}^{C}

MATERIAL WORK SHEET

PERMA BALLAST® INSTALLATION

SHOP	PC #	MATERIAL	STOCK NO.	QTY	SS/PUR	COST
		CANNING PLATES: 960 SF AREA FOR 2-180-6-V AT 100% COVERAGE				
1151	001	10.2# OSS PLATE (10.2#/SF)	9515-00-153-3184	960	\$4.12/SF	\$3,955
11S1	002	2-1/2 X 2-1/2" ANGLE (4.1#LF)	9520-00-277-4920	275	\$2.12/FT	\$583
		SPONSON STIFFENING:				
11S1	003	5" X 4" ANGLE (7.5#LF)	9520-00-277-5978	360	\$6.46/FT	\$2,326
71AA	004	PAINT				\$1,920
74YY	N/A	BLAST AND PAINT FACILITY				\$19,200
ALL	N/A	CONSUMABLES				\$7,418
		TOTAL MATERIAL COSTS:				\$35,402
		TOTAL MATERIAL WEIGHT:				16,890 #'S
						-

				T	T	T	Τ	T								
		75K	A									58				
		66	TS				22					2				
		72	R1													
2003			AA			24	œ					32				
SEPT		64	A6				16					16				
TE	TES	64								24		24				
DA	G PL/	99	P1													
	NINN	51														
	UT CA	38	YY													
	THO	35	GA				4					4				
	OR TYPICAL VOID WITHOUT CANNING PLATES	76	W1				28					28				
	AL VC	17							32			32				
	YPIC	24	11		4							4				
	FOR 1	11	S5		10			∞				18				
	ESTIMATE SHEET F	74	YY													
	TE SF	11	9S													
	TIMA	11					4					4				
BACK	ES	11	S 1				70					20				
ESTIMATORS NAME F			NO ACCOMPLISHED .	TYPICAL VOID FOR PERMA BALLAST(R)(R)	OPEN & CLOSE VOID	TOUCH-UP PAINT	CUT/INST ACCESSES (2) 12" FOR FILLING	TEST ACCESSES	REMOVE/REINSTALL INTERFERENCES	PATCH & REPAIR DECK COVERING		TOTALS:	TOTAL 264 MH'S TOTAL 33 MD'S	TOTAL PROD \$21,186	MATERIAL: \$500	NO ADDITIONAL WEIGHT ADDED
A		P	Z .			<u> </u>			<u> </u>	<u></u>						

OPTION 1 (2ND DECK VOIDS 0.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MATS	MAT#
2-165-8-V	0.5°	35%	200LBS/FT ³	\$115,046	\$7.744	3,695 #'S
2-180-6-V	0.5°					
2-250-4-V	0.5°					
TOTALS				\$115,046	\$7,744	3,695 #'S

OPTION 2 (2ND DECK VOIDS 0.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	0.5°	22%	325LBS/FT ³	\$72,314	\$4,867	2,322 #'S
2-180-6-V	0.5°					
2-250-4-V	0.5°					
TOTALS				\$72,314	\$4,867	2,322 #'S

OPTION 3 (4TH DECK VOIDS 0.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	# TVW
4-96-2-V	0.5°					
4-100-4-V	0.5°					
4-104-2-V	0.5°					
4-108-2-V	0.5°					
4-113-4-V	0.5°					
4-118-2-V	0.5°					
4-123-6-V	0.5°					
4-148-2-V	0.5°					
4-152-2-V	0.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-156-2-V	0.5°	%56	200LBS/FT ³	\$21,186	\$500	0
4-160-2-V	0.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-165-4-V	0.5°					
4-170-2-V	0.5°					
4-175-6-V	0.5°					
4-180-6-V	0.5°					
4-185-4-V	0.5°					
TOTALS				\$63.558	\$1.500	0

(4TH DECK VOIDS 0.5° LIST)

TANK	TOT	DEDCENT	TVDE	ENT TYPE PRODUC	MATE	MAT#
IAIN	LIST	LENCELL	1112	INOD®	INTER I	# TVIAI
4-96-2-V	0.5°					
4-100-4-V	0.5°					
4-104-2-V	0.5°					
4-108-2-V	0.5°					
4-113-4-V	0.5°					
4-118-2-V	0.5°					
4-123-6-V	0.5°					
4-148-2-V	0.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-152-2-V	0.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-156-2-V	0.5°					
4-160-2-V	0.5°					
4-165-4-V	0.5°					
4-170-2-V	0.5°					,
4-175-6-V	0.5°					
4-180-6-V	0.5°					
4-185-4-V	0.5°					
TOTALS				\$42,372	\$1,000	0

OPTION 5 (8TH DECK VOIDS 0.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
8-210-10-V	0.5°					
8-215-8-V	0.5°					
8-215-10-V	0.5°	% 56	200LBS/FT ³	\$21,186	\$500	0
8-220-8-V	°5°0					
8-220-10-V	°5.0	% 56	200LBS/FT ³	\$21,186	\$500	0
8-225-6-V	°5.0					
8-225-8-V	°5.0					
8-230-4-V	0.5°					
8-230-6-V	0.5°		=			
8-235-6-V	0.5°					
8-235-8-V	0.5°					
TOTALS				\$42,372	\$1,000	0

OPTION 6 (8TH DECK VOIDS 0.5° LIST)

_		ι –				Γ	_	,	т			т
MAT#			0									0
MAT \$			\$500									\$500
PROD \$			\$21,186									\$21,186
TYPE			325LBS/FT3									
PERCENT			95%									
LIST	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

(2ND DECK VOIDS 1.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	# TVM
2-165-8-V	1.0°	74%	200LBS/FT ³	\$243,240	\$16,373	7,811 #'S
2-180-6-V	1.0°					
2-250-4-V	1.0°					
TOTALS				\$243,240	\$16,373	7,811 #'S

OPTION 8 (2ND DECK VOIDS 1.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	1.0°	46%	325LBS/FT ³	\$151,204	\$10,178	4,856 #'S
2-180-6-V	1.0°					
2-250-4-V	1.0°					
TOTALS				\$151,204	\$10,178	4,856 #'S

OPTION 9 (4TH DECK VOIDS 1.0° LIST)

TANK	TSI	PERCENT	TVPF	PROD®	MAT¢	MAT#
4-96-2-V	1.0°					
4-100-4-V	1.0°					
4-104-2-V	1.0°					
4-108-2-V	1.0°					
4-113-4-V	1.0°					
4-118-2-V	1.0°					
4-123-6-V	1.0°					
4-148-2-V	1.0°					
4-152-2-V	1.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-156-2-V	1.0°	%56	200LBS/FT ³	\$21,186	\$500	0
4-160-2-V	1.0°	%56	200LBS/FT3	\$21,186	\$500	0
4-165-4-V	1.0°	%56	200LBS/FT ³	\$21,186	\$500	0
4-170-2-V	1.0°	%56	200LBS/FT ³	\$21,186	\$500	0
4-175-6-V	1.0°	%56	200LBS/FT ³	\$21,186	\$500	0
4-180-6-V	1.0°					
4-185-4-V	1.0°					
TOTALS				\$127,116	\$3,000	0

(4TH DECK VOIDS 1.0° LIST)

		10 ±)	DECR VOIDS I.U LIST,	LISI)		
TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	# WAT#
4-96-2-V	1.0°					
4-100-4-V	1.0°					
4-104-2-V	1.0°					
4-108-2-V	0'1					
4-113-4-V	1.0°					
4-118-2-V	1.0°					
4-123-6-V	1.0°					
4-148-2-V	1.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-152-2-V	1.0°	%S6	325LBS/FT ³	\$21,186	\$500	0
4-156-2-V	1.0°	%56	325LBS/FT ³	\$21,186	\$500	0
4-160-2-V	1.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-165-4-V	1.0°					
4-170-2-V	1.0°					
4-175-6-V	1.0°					
4-180-6-V	1.0°					
4-185-4-V	1.0°					
TOTALS				\$84,744	\$2,000	0

${\rm OPTION~11} \\ {\rm (8^{TH}\,DECK~TANKS~1.0^{\circ}~LIST)}$

MAT#	0	0	0									0
MAT \$	\$200	\$200	\$200									\$1,500
PROD \$	\$21,186	\$21,186	\$21,186									\$63,558
TYPE	200LBS/FT3	200LBS/FT ³	200LBS/FT ³		·							
PERCENT	%56	%56	%56									
LIST	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

${\rm OPTION~12} \\ {\rm (8^{TH}\,DECK~TANKS~1.0^{\circ}~LIST)}$

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
8-210-10-V	1.0°	%56	325LBS/FT ³	\$21,186	\$500	0
8-215-8-V	1.0°					
8-215-10-V	1.0°	%56	325LBS/FT ³	\$21,186	\$500	0
8-220-8-V	1.0°					
8-220-10-V	1.0°					
8-225-6-V	1.0°					
8-225-8-V	1.0°					
8-230-4-V	1.0°					
8-230-6-V	1.0°					
8-235-6-V	1.0°					
8-235-8-V	1.0°					
TOTALS				\$42,372	\$1,000	0

OPTION 13 (2ND DECK VOIDS 1.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	1.5°	%56	200LBS/FT ³	\$164,352	\$11,063	5,278 #'S
7-180-6-V	1.5°	11%	200LBS/FT ³	\$57,852	\$3,894	1,161 #'S
2-250-4-V	1.5°					
TOTALS				\$222,204	\$14,957	6,439 #'S

OPTION 14 (2ND DECK VOIDS 1.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	1.5°	2002	325LBS/FT ³	\$230,093	\$15,488	7,389#'S
2-180-6-V	1.5°					
2-250-4-V	1.5°					
TOTALS				\$230,093	\$15,488	7,389#'S

OPTION 15 (4TH DECK VOIDS 1.5° LIST)

		(4 DI	DECIN VOIDS IN LIST,	(1017)		
TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
4-96-2-V	1.5°					
4-100-4-V	1.5°					
4-104-2-V	1.5°					
4-108-2-V	1.5°					
4-113-4-V	1.5°					
4-118-2-V	1.5°					
4-123-6-V	1.5°	95%	200LBS/FT ³	\$21,186	\$200	0
4-148-2-V	1.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-152-2-V	1.5°	95%	200LBS/FT3	\$21,186	\$500	0
4-156-2-V	1.5°	95%	200LBS/FT3	\$21,186	\$500	0
4-160-2-V	1.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-165-4-V	1.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-170-2-V	1.5°	95%	200LBS/FT3	\$21,186	\$500	0
4-175-6-V	1.5°	95%	200LBS/FT	\$21,186	\$200	0
4-180-6-V	1.5°	95%	SOOTBS/FT	\$21,186	\$500	0
4-185-4-V	1.5°	95%	200LBS/FT	\$21,186	\$500	0
TOTALS				\$211,860	\$5,000	0

(4TH DECK VOIDS 1.5° LIST)

TANK	T 10T	DEDCENT	ENT TVPE DD	PDOD ¢	MATE	MAT#
UNIWI	1017	IENCENI	1111	i MOD #	IVICA I D	H T WIAT
4-96-2-V	1.5°					
4-100-4-V	1.5°					
4-104-2-V	1.5°					
4-108-2-V	1.5°					
4-113-4-V	1.5°					
4-118-2-V	1.5°					
4-123-6-V	1.5°					
4-148-2-V	1.5°	%56	325LBS/FT3	\$21,186	\$500	0
4-152-2-V	1.5°	%56	325LBS/FT ³	\$21,186	\$500	0
4-156-2-V	1.5°	95%	325LBS/FT	\$21,186	\$500	0
4-160-2-V	1.5°	%56	325LBS/FT ³	\$21,186	\$200	0
4-165-4-V	1.5°	%56	325LBS/FT ³	\$21,186	\$500	0
4-170-2-V	1.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-175-6-V	1.5°					
4-180-6-V	1.5°					
4-185-4-V	1.5°					
TOTALS				\$127,116	\$3,000	0

OPTION 17 (8TH DECK TANKS 1.5° LIST)

31 -												
MAT#	0	0	0	0	0				0			0
MAT \$	\$500	\$500	\$500	\$500	\$500				\$500			\$3,000
PROD \$	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186				\$21,186			\$127,116
TYPE	200LBS/FT3	200LBS/FT ³	200LBS/FT ³	200LBS/FT3	200LBS/FT ³				200LBS/FT3			
PERCENT	%56	% 56	%56	%S6	%S6				%56			
LIST	1.5°	1.5°	1.5°	1.5°	1.5°	1.5°	1.5°	1.5°	1.5°	1.5°	1.5°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

OPTION 18 (8TH DECK TANKS 1.5° LIST)

	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
1.5°	0	95%	325LBS/FT ³	\$21,186	\$500	0
1.6	1.5°	95%	325LBS/FT ³	\$21,186	\$200	0
1.	1.5°	%56	325LBS/FT ³	\$21,186	\$500	0
1.5	1.5°					
1.5	1.5°					
1.	1.5°					
1.	1.5°					
1.	1.5°					
1.	1.5°					
1.	1.5°					
1.	1.5°					
				\$63,558	\$1,500	0

OPTION 19 (2ND DECK VOIDS 2.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	2.0°	%56	200LBS/FT3	\$164,352	\$11,063	5,278 #'S
2-180-6-V	2.0°	36%	200LBS/FT ³	\$184,074	\$12,745	3,800 #'S
2-250-4-V	2.0°					
TOTALS				\$348,426	\$23,808	S.# 8L0'6

OPTION 20 (2ND DECK VOIDS 2.0° LIST)

TANK	LSIT	PERCENT	TYPE	PROD \$	MAT \$	MAT #
2-165-8-V	2.0°	% 5 6	325LBS/FT ³	\$164,352	\$11,063	S'# 872,2
2-180-6-V	2.0°					
2-250-4-V	2.0°					
TOTALS				\$164,352	\$11,063	S,# 872,8

OPTION 21 (4TH DECK VOIDS 2.0° LIST)

IWIND	LISI	PERCENT	TYPE	PROD \$	MATS	MAT#
4-96-2-V	0.2					
4-100-4-V	2.0°					
4-104-2-V	2.0°					
4-108-2-V	2.0°	%56	200LBS/FT	\$21,186	\$500	0
4-113-4-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-118-2-V	2.0°	95%	200LBS/FT	\$21,186	\$500	0
4-123-6-V	2.0°	%56	200LBS/FT	\$21,186	\$500	0
4-148-2-V	2.0。	95%	200LBS/FT3	\$21,186	\$500	0
4-152-2-V	2.0°	%56	200LBS/FT ³	\$21,186	\$500	0
4-156-2-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-160-2-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-165-4-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-170-2-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-175-6-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-180-6-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
4-185-4-V	2.0°	95%	200LBS/FT ³	\$21,186	\$500	0
TOTALS				\$275,418	\$6,500	0

OPTION 22

			OF 11011 22			
		(4 TH D)	(4 TH DECK VOIDS 2.0° LIST)	LIST)		
TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	# TW
4-96-2-V	2.0°					
4-100-4-V	2.0°					
4-104-2-V	2.0°					
4-108-2-V	2.0°					
4-113-4-V	2.0°					
4-118-2-V	2.0°					
4-123-6-V	2.0。					
4-148-2-V	2.0°	% 5 6	325LBS/FT ³	\$21,186	\$500	0
4-152-2-V	2.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-156-2-V	2.0°	%56	325LBS/FT ³	\$21,186	\$500	0
4-160-2-V	2.0°	%56	325LBS/FT ³	\$21,186	\$500	0
4-165-4-V	2.0°	% S6	325LBS/FT ³	\$21,186	\$500	0
4-170-2-V	2.0°	% 56	325LBS/FT ³	\$21,186	\$500	0
4-175-6-V	2.0°	%56	325LBS/FT ³	\$21,186	\$500	0
4-180-6-V	2.0°	% S 6	325LBS/FT ³	\$21,186	\$500	0
4-185-4-V	2.0°					

\$4,000

\$169,488

TOTALS

${\rm OPTION~23} \\ {\rm (8^{TH}\,DECK~TANKS~2.0^{\circ}~LIST)}$

MAT#	0	0	0	0	0	0	0					0
MAT \$	\$500	\$500	\$500	\$500	\$200	\$500	\$500					\$3.500
PROD \$	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186					\$148,302
TYPE	200LBS/FT ³	200LBS/FT3	200LBS/FT3	200LBS/FT ³	200LBS/FT ³	200LBS/FT ³	200LBS/FT ³					
PERCENT	%56	%56	%56	%56	%56	95%	%56					
LIST	2.0°	2.0°	2.0°	2.0°	2.0°	2.0°	2.0°	2.0°	2.0°	2.0°	2.0°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

OPTION 24 (8TH DECK TANKS 2.0° LIST)

TANK	LISIT	PERCENT	TYPE	PROD \$	MAT \$	MAT#
8-210-10-V	2.0°	95%	325LBS/FT ³	\$21,186	\$500	0
8-215-8-V	2.0°	95%	325LBS/FT ³	\$21,186	\$500	0
8-215-10-V	2.0°	95%	325LBS/FT ³	\$21,186	\$500	0
8-220-8-V	2.0°	95%	325LBS/FT ³	\$21,186	\$500	0
8-220-10-V	2.0°					
8-225-6-V	2.0°					
8-225-8-V	2.0°					
8-230-4-V	2.0°					
8-230-6-V	2.0°	95%	325LBS/FT ³	\$21,186	\$500	0
8-235-6-V	2.0°					
8-235-8-V	2.0°					
TOTALS				\$105,930	\$2,500	0

OPTION 25 (2ND DECK VOIDS 2.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	2.5°	95%	200LBS/FT3	\$164,352	\$11,063	5,278 #'S
2-180-6-V	2.5°	61%	200LBS/FT ³	\$320,815	\$21,595	10,302 #'S
2-250-4-V	2.5°					
TOTALS				\$485,167	\$32,658	15,580 #'S

OPTION 26 (2ND DECK VOIDS 2.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	2.5°	95%	325LBS/FT ³	\$164,352	\$11,063	5,278 #'S
2-180-6-V	2.5°	15%	325LBS/FT ³	\$78,889	\$5,310	2,533 #'S
2-250-4-V	2.5°					
TOTALS				\$243,241	\$16.373	7,811 #'S

OPTION 27 (4TH DECK VOIDS 2.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
4-96-2-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-100-4-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-104-2-V	2.5°	95%	200LBS/FT	\$21,186	\$500	0
4-108-2-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-113-4-V	2.5°	95%	200LBS/FT3	\$21,186	\$500	0
4-118-2-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-123-6-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-148-2-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-152-2-V	2.5°	95%	200LBS/FT3	\$21,186	\$500	0
4-156-2-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-160-2-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-165-4-V	2.5°	95%	200LBS/FT ³	\$21,186	\$200	0
4-170-2-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-175-6-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
4-180-6-V	2.5°	95%	200LBS/FT ³	\$21,186	\$200	0
4-185-4-V	2.5°	95%	200LBS/FT ³	\$21,186	\$500	0
TOTALS				\$338,976	88,000	0

(4TH DECK VOIDS 2.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
4-96-2-V	2.5°					
4-100-4-V	2.5°					
4-104-2-V	2.5°					
4-108-2-V	2.5°					
4-113-4-V	2.5°					
4-118-2-V	2.5°					
4-123-6-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-148-2-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-152-2-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-156-2-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-160-2-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-165-4-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-170-2-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-175-6-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-180-6-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
4-185-4-V	2.5°	95%	325LBS/FT ³	\$21,186	\$500	0
TOTALS				\$211,860	\$5.000	0

OPTION 29 (8TH DECK TANKS 2.5° LIST)

			,			,						
MAT#	0	0	0	0	0	0	0	0	0			0
MAT \$	\$500	\$500	\$500	\$500	\$500	\$500	\$500	\$500	\$500			\$4,500
PROD \$	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186			\$190,674
TYPE	200LBS/FT3	200LBS/FT3	200LBS/FT3	200LBS/FT3	200LBS/FT3	200LBS/FT3	200LBS/FT	200LBS/FT3	200LBS/FT			
PERCENT	95%	95%	95%	95%	%56	95%	95%	95%	95%			
LIST	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

OPTION 30 (8TH DECK TANKS 2.5° LIST)

±												
MAT #												0
MAT \$	\$500	\$500	\$500	\$500	\$200		\$500	:				\$3,000
PROD \$	\$21,186	\$21,186	\$21,186	\$21,186	\$21,186		\$21,186					\$127,116
TYPE	325LBS/FT ³	325LBS/FT ³	325LBS/FT3	325LBS/FT3	325LBS/FT ³		325LBS/FT ³					
PERCENT	95%	95%	%56	95%	95%		%56					
LIST	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	2.5°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

OPTION 31 (2ND DECK VOIDS 3.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	# LVW
2-165-8-V	3.0°	%56	200LBS/FT ³	\$164,352	\$11,063	5,278 #'S
2-180-6-V	3.0°	95%	200LBS/FT ³	\$226,161	\$15,222	7,262 #'S
2-250-4-V	3.0°					
TOTALS				\$390,513	\$26,285	12,540 #'S

OPTION 32 (2ND DECK VOIDS 3.0° LIST)

10,345 #'S	\$21,684	\$322,130				TOTALS
					3.0°	2-250-4-V
2,067 #'S	\$10,621	\$157,778	325LBS/FT ³	30%	3.0°	2-180-6-V
5,278 #'S	\$11,063	\$164,352	325LBS/FT ³	95%	3.0°	2-165-8-V
WAT #	MAT \$	PROD \$	TYPE	PERCENT	LIST	TANK

OPTION 33 (4TH DECK VOIDS 3.0° LIST)

- 1 >	ECA	THE CHANGE	TALLED SOLUTION TO THE TALLED	2001)	* 6.	
LIST		PERCENT	TYPE	PROD \$	MATS	MAT#
3.0°						
3.0°	-					
3.0°	_					
3.0°						
3.0°						
3.0°						
3.0°						
3.0°						
3.0°						
3.0°						
3.0°						
3.0°						
3.0°	L					
3.0°						
3.0°						
3.0°						
	ı				+	

(4TH DECK VOIDS 3.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
4-96-2-V	3.0°					
4-100-4-V	3.0°					
4-104-2-V	3.0°				,	
4-108-2-V	3.0°					
4-113-4-V	3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-118-2-V	3.0°	95%	325LBS/FT	\$21,186	\$500	0
4-123-6-V	3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-148-2-V	3.0°	95%	325LBS/FT	\$21,186	\$500	0
4-152-2-V	3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-156-2-V	3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-160-2-V	3.0°	%56	325LBS/FT	\$21,186	\$500	0
4-165-4-V	3.0°	<i>%</i> 56	325LBS/FT3	\$21,186	\$500	0
4-170-2-V	3.0°	%56	325LBS/FT ³	\$21,186	\$500	0
4-175-6-V	3.0°	%56	325LBS/FT ³	\$21,186	\$500	0
4-180-6-V	3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
4-185-4-V	3.0°	%56	325LBS/FT ³	\$21,186	\$500	0
TOTALS				\$254,232	86,000	0

$\begin{array}{c} \text{OPTION 35} \\ \text{(8$^{\text{TH}}$DECK VOIDS 3.0° LIST)} \end{array}$

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
8-210-10-V	3.0°	95%	200LBS/FT ³	\$21,186	\$200	0
8-215-8-V	3.0∘	95%	200LBS/FT3	\$21,186	\$500	0
8-215-10-V	3.0°	95%	200LBS/FT ³	\$21,186	\$500	0
8-220-8-V	3.0°	95%	200LBS/FT3	\$21,186	\$500	0
8-220-10-V	3.0°	95%	200LBS/FT ³	\$21,186	\$500	0
8-225-6-V	3.0°	95%	200LBS/FT3	\$21,186	\$500	0
8-225-8-V	3.0°	95%	200LBS/FT	\$21,186	\$500	0
8-230-4-V	3.0°	26	200LBS/FT ³	\$21,186	\$500	0
8-230-6-V	3.0°	% 56	200LBS/FT	\$21,186	\$500	0
8-235-6-V	3.0°	%S6	200LBS/FT3	\$21,186	\$500	0
8-235-8-V	3.0°	95%	200LBS/FT ³	\$21,186	\$500	0
TOTALS				\$233,046	\$5,500	0

OPTION 36 (8TH DECK VOIDS 3.0° LIST)

LIST	r PERCENT	TYPE	PROD \$	MAT \$	MAT#
3.0°	%56	325LBS/FT ³	\$21,186	\$500	0
3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
3.0°	%56	325LBS/FT ³	\$21,186	\$500	0
3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
3.0°	%56	325LBS/FT ³	\$21,186	\$500	0
3.0°	%56	325LBS/FT ³	\$21,186	\$500	0
3.0°	c				
3.0°					
3.0°	95%	325LBS/FT ³	\$21,186	\$500	0
3.0°	0				
3.0°	0				
			\$148,302	\$3,500	0

Appendix D: Preliminary Cost Comparison

												ľ						
List (Degrees)			0.5	v					_							1.5		
Rallast Density		(200 lbs/ft ³)			(325 lbs/ft ³)			(200 lbs/ft ²)			(325 lbs/ft³)			(200 lbs/ft³)			(325 lbs/ft ³)	
Tank Location	2nd Deck	Г	8th Deck	2nd Deck		8th Deck	2nd Deck	4th Deck	8th Deck	2nd Deck	┢	8th Deck	2nd Deck	4th Deck	8th Deck	2nd Deck	4th Deck	8th Deck
Production Cost (\$)	115,046	63,558	42,372	72,314	42,372	21,186	243,240	127,116	63,558	151,204	84,744	42,372	222,204	211,860	127,116	230,093	127,116	63,558
Material Cost (\$)	7,744	1,500	1,000	4,867	1,000	200	16,373	3,000	1,500	10,178	2,000	1,000	14,957	5,000	3,000	15,488	3,000	1,500
Perma Cost (\$)	127,900	134,650	136,900	198,025	211,250	219,300	158,950	174,025	179,200	277,375	311,875	355,575	189,775	223,075	225,100	356,725	421,125	453,900
Total Cost (\$)	250,690	199,708	180,272	275,206	254,622	240,986	418,563	304,141	244,258	438,757	398,619	398,947	426,936	439,935	355,216	602,306	551,241	518,958
Material Weight (Ltons)	1.65	0	0	1.04	0	0	3.49	0	ő	2.17	0	0	2.87	0	0	3.30	0	0
Perma Weight (Ltons)	124	154	164	127	150	164	262	329	352	265	325	401	399	547	556	403	515	572
Total Weight (Ltons)	125.65	154	164	128.04	150	164	265.49	329	352	267.17	325	401	401.87	547	556	406.30	515	572
ΔKG (ft)	0.02	00.00	-0.02	0.02	00:0	-0.02	0.04	-0.01	-0.05	0.03	-0.01	-0.05	0.05	-0.03	-0.08	0.05	-0.02	-0.08

**Perma cost estimates based on \$100,000 + \$225/It for 200 lbs/ft^3 and \$125,000 + \$575/It for 325 lbs/ft^3

List (Degrees)			2						2.5							3		
Ballast Density		(200 lbs/ft ²)			(325 lbs/ft ³)			(200 lbs/ft ³)			(325 lbs/ft ³)			(200 lbs/ft ³)			(325 lbs/ft ³)	
Tank Location	2nd Deck	4th Deck	8th Deck	2nd Deck 4th Deck	_	8th Deck	2nd Deck	4th Deck	8th Deck	2nd Deck	4th Deck	8th Deck	2nd Deck	4th Deck	8th Deck	2nd Deck	4th Deck	8th Deck
Production Cost (\$)	348.426	275.418	148,302	164,352	169,488	105,930	485,167	338,976	190,674	243,241	211,860	127,116	390,513		233,046	322,130	254,232	148,302
Material Cost (\$)	23.808	6,500	3,500	11.063	4,000		32,658	8,000	4,500	16,373	5,000	3,000	26,285		5,500	21,684	6,000	3,500
Perma Cost (\$)	221.275	261,550	288,325	439,525	529,225	586,150	252,775	292,600	340,750	517,725	636,175	684,475	295,525		408,700	296,500	744,275	867,900
Total Cost (\$)	593,509	543,468	440,127	614,940	702,713	694,580	770,600	639,576	535,924	777,339	853,035	814,591	712,323	N/A	647,246	940,314	1,004,507	1,019,702
Material Weight (Ltons)	4.05	0	0	2.36	0	0	96.9	0	0	3.49	0	0	9.60		0	4.62	0	0
Perma Weight (Ltons)	539	718	837	547	703	802	629	856	0/0'1	683	688	973	698		1,372	820	1,077	1,292
Total Weight (Ltons)	543.05	718	837	549.36	703	802	685.96	856	1,070	686.49	889	973	874.60	N/A	1,372	824.62	1,077	1,292
A KG (ft)	0.07	0.04	-0.11	80:0	-0.03	-0.11	60:0	-0.04	-0.15	60.0	-0.05	-0.14	0.12		-0.18	0.10	-0.06	-0.19

**Perma cost estimates based on \$100,000 + \$225/ft for 200 lbs/ft²3 and \$125,000 + \$575/ft for 325 lbs/ft²7 **Yellow indicates voids not sufficient on deck to ballast necessary amount

Appendix E: Complete Structural Analysis

Structural Analysis for Compartment 2-165-8-V: Shell Plating

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\text{ft}^3}$$

Density of Perma Ballast®

%Perma_fill := .95

Based on 95% Fillable Volume

Conversion Factors

lton := 2240b

$$\rho_{sw} := 64.4 \frac{lb}{ft^3}$$

Density of Seawater

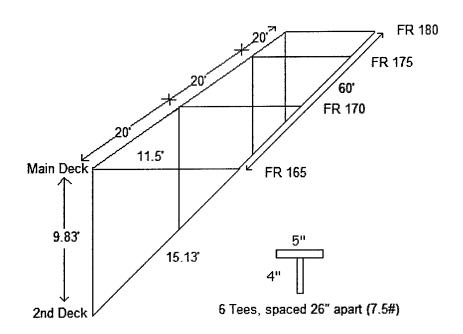
Compartment 2-165-8-V

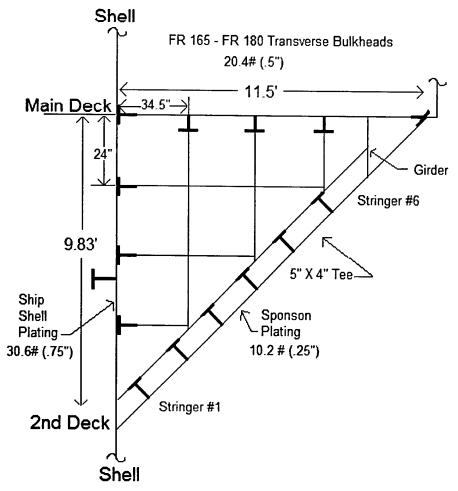
Volume := 3969ft³

Volume of Compartment

 $W_{Perma} := 354 ton$

Weight of Permanent Ballast





Ship Motion Factors:

V := 1.25 Vertical
$$\alpha := asin\left(\frac{11.5}{15.13}\right)$$
 $\alpha = 49.471deg$

A := 0.75 Athwartship $\beta := asin\left(\frac{9.83}{15.13}\right)$ $\beta = 40.519deg$

F := 0.4 Fore/Aft

Loading on Shell Plating:

$$W_{Perma} = 7.93 \times 10^5 \text{ lb}$$
 Weight of Perma Ballast®

$$F_V := V \cdot W_{Perma}$$
 $F_V = 9.912 \times 10^5 \text{ lb}$ Vertical Load (Downward)

$$F_A := A \cdot W_{Perma}$$
 $F_A = 5.947 \times 10^5 \text{ lb}$ Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$
 $F_F = 3.172 \times 10^5 \text{ lb}$ Fore/Aft Load

$$F_N := F_V \cdot \cos(\beta) + F_A \cdot \cos(\alpha)$$
 $F_N = 1.14 \times 10^6 \text{ lb}$ Normal Force

Resulting Pressure on Sponson Plating:

Compartment Dimensions:

$$L := 60ft$$

$$Ht := 15.13ft$$

Area :=
$$L \cdot H$$

Area :=
$$L \cdot Ht$$
 Area = 907.8ft²

$$P := \frac{F_N}{Area}$$

$$P := \frac{F_{N}}{Area} \qquad P = 1.256 \times 10^{3} \frac{lb}{ft^{2}} \qquad P = 8.72 \frac{lb}{in^{2}}$$

$$P = 8.72 \frac{lb}{in^2}$$

$$P_{95\%} := \%_{Perma_fill} \cdot 1$$

$$P_{95\%} = 1.193 \times 10^3 \frac{1}{10^3}$$

 $P_{95\%} := \%_{Perma_fill} \cdot P \quad P_{95\%} = 1.193 \times 10^3 \frac{lb}{ft^2}$ Based on 95% Fillable Volume

$$H := \frac{P_{95\%}}{\rho_{\text{sw}}}$$

$$H = 18.524ft$$

Equivalent Head

In accordance with CVN 76 Specs, sponson plating aft of Frame 88 shall be designed for 1000 psf. Therefore, less ballast must be added to these sponson voids.

Design of Plating for Surface Ships, IAW General Specs (Section 100)

Panels of plating shall be proportioned so as not to exceed the breadth-thickness ratios found below:

$$AR := \frac{b}{a}$$

$$AR := \frac{b}{a}$$
 $AR = 0.108$

 $t_{actual} := .25in$ Thickness of Plate, HTS

H = 18.524ft

Head of Water

K := 1

For AR = 0.108, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

$$\frac{b}{t} \leq \frac{C}{K \cdot \sqrt{H}}$$

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{\min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$$

Note: The stiffeners are assumed to be spaced evenly across

the sponson shell plating with 6 stiffeners spanning 15.13'.

 $\mathsf{t}_{\min} = 0.28\mathsf{in}$

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .28 in. The actual plate thickness is .25 in. Therefore, the 2nd deck shell plating must either be stiffened or have less ballast added. It should be noted,

however, that some structural margin exists within these calculations and that .28 is close enough to .25 in. As a result, it is possible that stiffening is not required for the 2nd deck shell sponson plating.

t_min > t_actual

Calculations to Determine Maximum Ballast Weight:

Ship Motion Factors:

V := 1.25Vertical

Athwartship A := 0.75

Fore/Aft F := 0.4

Resulting Pressure on Sponson Plating:

Compartment Dimensions:

$$L := 60 \text{ft}$$
 $Ht := 15.13 \text{ft}$

Area := $L \cdot Ht$ Area = 907.8ft²

 $P := 1000 \cdot \frac{lb}{c.^2}$ Maximum pressure allowed for sponson shell plating IAW CVN 76 Specs

 $F_{N_new} := P \cdot Area$ $F_{N_new} = 9.078 \times 10^5 lb$

 $H := \frac{P}{\rho_{SW}}$ H = 15.528ft New Equivalent Head

$$\frac{Loading \ on \ Shell \ Plating:}{\alpha := asin \left(\frac{11.5}{15.13}\right)} \qquad \alpha = 49.471 deg$$

$$\beta := a\sin\left(\frac{9.83}{15.13}\right) \qquad \beta = 40.519 deg$$

$$F_{N_new} = F_{V_new} \cdot \cos(\beta) + F_{A_new} \cdot \cos(\alpha)$$

 $F_{V_new} = V \cdot W_{Perma}$ Vertical Load (Downward)

 $F_{A_new} = A \cdot W_{Perma}$ Athwartship Load (Port)

 $F_{N_new} = V \cdot W_{Perma} \cdot \cos(\beta) + A \cdot W_{Perma} \cdot \cos(\alpha)$

 $W_{Perma} := \frac{F_{N_new}}{(v \cdot \cos(\beta) + A \cdot \cos(\alpha))}$

W_{Perma} = 281.904ton Weight of <u>Perma Ballast®</u> that cannot be exceeded

Calculations to Determine Strength Corrections:

t_{min} := .25in Thickness of Plate, HTS

H = 15.528ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1 Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{limit} := t_{min} \cdot \frac{C}{\frac{1}{2}}$$
 $b = 26in$
 $\frac{b}{2} = 13in$

 $b_{limit} = 25.377$ in

25.377 in is the maximum distance that the stiffeners can be apart and have no plastic deformation to the existing .25 in plating. The current stiffeners are located 26 in apart. It is recommended that the stiffeners be placed less than maximum distance apart. Therefore, adding a vertical stiffener in the middle of each existing stiffener on the shell sponson plating will put b at 13 in and meet the 25.377 in requirement.

Results

A combination of Lesser Weight Addition to meet the 1000psf limit along with adding a stiffener in the middle of each existing stiffener will alleviate any structural concerns.

Stress Calculations:

k =coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := \mathbf{k} \cdot \mathbf{P} \cdot \left(\frac{\mathbf{b}}{\mathbf{t}}\right)^2$$

p = pressure based on 1000 psf on sponson shell plating

b = panel width

a = panel lengtht = plate thickness

$$k_{\text{simply_supported}} := .75$$

k_{clamped} := .5 k values are most conservative

b := 13in

based on 13 in separation between stiffeners

$$P = 6.944 \frac{lb}{in^2}$$

t := .25in

$$\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$$

IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{simply_supported}} = 14083 \frac{\text{lb}}{\text{in}^2}$$

$$\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

Maximum Allowable Stress for HTS is 40,000 psi. By adding stiffeners at 13 in, the stress falls well below that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress should be well below maximum allowable.

σ_{clamped}<<σ_{max} allowable

C coefficient derivation from beam theory:

Panel is considered a series of beams having FIXED ends.

$$I := \frac{1}{12} \cdot 1 \cdot t^3 \qquad q := \frac{\gamma \cdot H}{144}$$

$$q := \frac{\gamma \cdot H}{144}$$

$$\sigma_{x_max} \coloneqq \frac{\frac{1}{12} \cdot q \cdot b^2}{I} \cdot \frac{t}{2}$$
 plate bending now equivalent to Navy's standard, see Eqn 2

b_over_t :=
$$\sqrt{\frac{\sigma_{x_max} \cdot 288}{\gamma}} \frac{1}{\sqrt{H}}$$
 $\gamma := 64.4$

$$\sigma_{x_{max}} := 40000$$

without introducing K factor, used for shorter panels

$$C := \sqrt{\frac{\sigma_{x_max} \cdot 288}{\gamma}} \qquad C = 422.944$$

$$b_over_t \le \frac{353.861}{\sqrt{H}}$$

Calculations to Determine Strength Corrections:

 $t_{\min} := .25in$

Thickness of Plate, HTS

H = 15.528ft

Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1

Assume AR remains less than .5

C := 422.944t. For HTS. No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

$$\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{limit} := t_{min} \cdot \frac{C}{\frac{1}{1}}$$

$$K \cdot H^{2}$$

 $b_{limit} = 26.833in$

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2 p$$
 = pressure based on 1000 psf on sponson shell plating

b = panel width

a = panel length

t = plate thickness

$$k_{simply_supported} := .75$$

$$k_{clamped} := .5$$

$$a := 20ft$$

$$b_{limit} = 26.833in$$

$$AR := \frac{a}{b}$$

$$AR = 18.462$$

$$P = 6.944 \frac{lb}{in^2}$$

$$t := .25in$$

$$\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$$
 IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b_{\text{limit}}}{t}\right)^2$$

$$\sigma_{simply_supported} = 60000 \frac{lb}{in^2}$$

$$\sigma_{clamped} := k_{clamped} \cdot P \cdot \left(\frac{b_{limit}}{t}\right)^{2}$$

$$\sigma_{clamped} = 40000 \frac{lb}{in^2}$$

Structural Analysis for Compartment 2-165-8-V: Ship Shell Plating Inner Longitudinal Bulkhead

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\text{ft}^3}$$

Density of Perma Ballast®

%Perma_fill := .95

Based on 95% Fillable Volume

Conversion Factors

lton := 2240b

$$\rho_{SW} := 64.4 \frac{lb}{ft^3}$$

Density of Seawater

Compartment 2-165-8-V

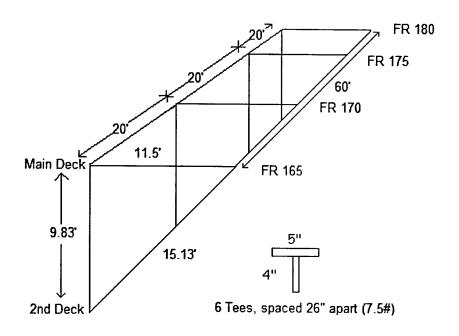
Volume := 3969ft³

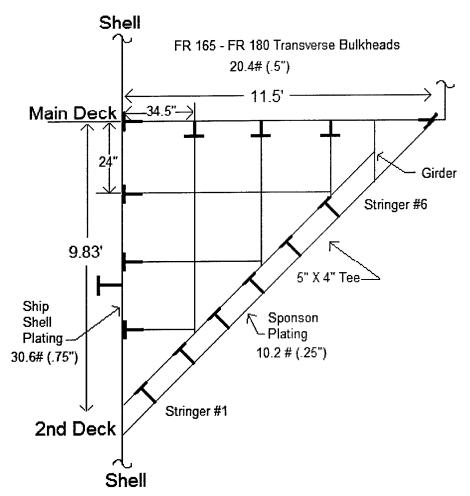
Volume of Compartment

 $W_{Perma} := 2811ton$

Maximum Weight of Permanent Ballast that can be added in accordance with shell plating

stress calculations





Ship Motion Factors:

V := 1.25	Vertical
A := 0.75	Athwartship
$\mathbf{F} = 0.4$	Fore/Aft

Loading on Inner Longitudinal Bulkhead Plating:

$W_{Perma} = 6.294 \times 10^5 lb$ Weight of <u>Perma Ballast®</u>

$F_{\mathbf{V}} := \mathbf{V} \cdot \mathbf{W}_{\mathbf{Perma}}$	$F_V = 7.868 \times 10^5 lb$	Vertical Load (Downward)
$F_A := A \cdot W_{Perma}$	$F_A = 4.721 \times 10^5 lb$	Athwartship Load (Port)
$F_F := F \cdot W_{Perma}$	$F_F = 2.518 \times 10^5 lb$	Fore/Aft Load
$F_{\mathbf{N}} := F_{\mathbf{A}}$	$F_N = 4.721 \times 10^5 lb$	Normal Force

Resulting Pressure on Inner Longitudinal Bulkhead Plating:

Compartment Dimensions:

$$L := 60 \text{ft} \qquad \qquad \text{Ht} := 9.83 \text{ft}$$

Area :=
$$L \cdot Ht$$
 Area = 589.8ft^2

$$L := 60 \text{ft} \qquad Ht := 9.83 \text{ft}$$

$$Area := L \cdot Ht \qquad Area = 589.8 \text{ft}^2$$

$$P := \frac{F_N}{Area} \qquad P = 800.407 \frac{lb}{\text{ft}^2} \qquad P = 5.558 \frac{lb}{\text{in}^2}$$

$$H := \frac{P}{\rho_{SW}} \qquad H = 12.429 \text{ft} \qquad \text{Equivalent Head}$$

$$P = 5.558 \frac{lb}{in^2}$$

$$H := \frac{P}{\rho_{sw}} \qquad H = 1$$

Design of Plating for Surface Ships, IAW General Specs (Section 100)

$$b := \frac{9.83}{2} f$$

$$b := \frac{9.83}{2} ft$$
 $a := 240 in$ $AR := \frac{b}{a}$ $AR = 0.246$

$$AR = 0.246$$

t_{actual} := .75in Thickness of Plate, HTS

H = 12.429ft

Head of Water

K := 1

For AR = 0.246, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$$

$$t_{\min} = 0.52in$$

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .52 in. The actual plate thickness is .75 in. Therefore, stiffening of the 2nd deck inner longitudinal bulkhead plating is not required.

t_min << t_actual

Calculations to Determine Strength Corrections:

 $t_{min} := .75in$ Thickness of Plate, HTS

H = 12.429ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1 Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{\kappa \sqrt{H}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{\text{limit}} := t_{\text{min}} \cdot \frac{C}{\frac{1}{2}} \qquad b = 58.98 \text{in}$$

$$K \cdot H^{\frac{2}{2}}$$

 $b_{limit} = 85.096$ in

85.096 in is the maximum distance that longitudinal stiffeners can be located apart and have no deformation occur to the .75 in plating. The current stiffener spacing is approximately 58" and well within the limiting width between stiffeners. As a result, the inner longitudinal plating does not need to be stiffened.

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$

p = pressure based on 95% of maximum ballast addition IAW sponson stress calculations

b = panel width

a = panel length

t = plate thickness

$$k_{simply_supported} := .75$$

$$k_{clamped} := .5$$

k values are most conservative

$$b := 58in$$

based on no stiffener addition

$$P = 5.558 \frac{lb}{in^2}$$

$$\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$$

IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{simply_supported}} = 24931 \frac{\text{lb}}{\text{in}^2}$$

$$\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{clamped}} = 16621 \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Stress for HTS is 40,000 psi. The calcuated stresses fall well below that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress should be well below maximum allowable.

 $\sigma_{clamped} << \sigma_{max_allowable}$

Structural Analysis for Compartment 2-165-8-V: Transverse Bulkhead Frames 165, 170, 175, and 180

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\text{ft}^3}$$
 Density of Perma Ballast®

Conversion Factors

Iton := 224Qb

$$\rho_{SW} := 64.4 \frac{lb}{ft^3}$$
 Density of Seawater

Compartment 2-165-8-V

calculations

Ship Motion Factors:

$$A := 0.75$$
 Athwartship

$$F := 0.4$$
 Fore/Aft

Loading on Transverse Plating:

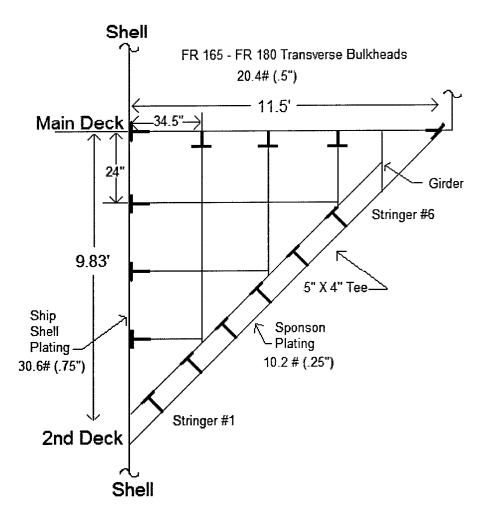
 $W_{Perma} = 6.294 \times 10^5 lb$ Weight of <u>Perma Ballast®</u>

$$F_V := V \cdot W_{Perma}$$
 $F_V = 7.868 \times 10^5 \text{ lb}$ Vertical Load (Downward)

$$F_A := A \cdot W_{Perma}$$
 $F_A = 4.721 \times 10^5 \, lb$ Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$
 $F_F = 2.518 \times 10^5 \, lb$ Fore/Aft Load

$$F_N := F_F$$
 $F_N = 2.518 \times 10^5 \, lb$ Normal Force



Resulting Pressure on Transverse Plating:

Compartment Dimensions:

$$\begin{array}{ll} L:=11.5 ft & Ht:=9.83 ft \\ Area:=.5 L. Ht & Area=56.523 ft^2 \\ P:=\frac{F_N}{Area} & P=4.454 \times 10^3 \frac{lb}{ft^2} & P=30.934 \frac{lb}{in^2} \\ H:=\frac{P}{\rho_{SW}} & H=69.168 ft & Equivalent Head \end{array}$$

Design of Plating for Surface Ships, IAW General Specs (Section 100)

The transverse plating at Frames 165, 170, 175, and 180 is stiffened by 3 vertical stiffeners and 3 horizontal stiffeners to include a vertical girder that runs from the Main Deck to the shell plating above stringer #6. The largest section of transverse plating that is unstiffened is approximately a 24"X34.5" section of plate. An assumption was made that all vertical stiffeners are spaced equally apart. All structural analysis was performed on this section of plate.

$$AR := \frac{b}{a}$$

$$AR := \frac{b}{a}$$
 $AR = 0.696$

t_{actual} := .50in Thickness of Plate, HTS

H = 69.168ft

Head of Water

K := .94

For AR = 0.696, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$$

$$t_{min} = 0.469in$$

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .469 in. The actual plate thickness is .50 in. Therefore, the 2nd deck transverse plating at Frame 165 does not require stiffeneing. Frames 170,175 and 180 are similar in design.

t_min < t_actual

Calculations to Determine Strength Corrections:

t_{min} := .50in Thickness of Plate, HTS

H = 69.168ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := .94 Assume AR remains .696

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \sqrt{M}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{limit} := t_{min} \cdot \frac{C}{\frac{1}{K \cdot H^2}}$$
 $b = 24 in$

 $b_{limit} = 25.583$ in

25.583 in is the maximum distance that longitudinal stiffeners can be located apart and have no deformation occur to the .5 in plating. The transverse plating at Frame 165 has 3 horizontal stiffeners and 3 vertical stiffeners. The current horizontal stiffener spacing is approximately 24" and within the limiting width between stiffeners. As a result, the transverse plating at Frame 165 does not need to be stiffened. Frames 170,175 and 180 are similar in design.

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$
 p = pressure based on 1000 psf on sponson shell plating

b = panel widtha = panel length t = plate thickness

$$k_{\text{simply_supported}} := .75$$

$$k_{clamped} := .5$$

k values are most conservative

$$a := 34.5 \text{in}$$
 $b := 24 \text{in}$

based on no stiffener addition

$$P = 30.934 \frac{lb}{in^2}$$
 $t := .50in$

$$\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$$
 IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{simply_supported} = 53453 \frac{lb}{in^2}$$

$$\sigma_{clamped} := k_{clamped} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{clamped}} = 35636 \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Stress for HTS is 40,000 psi. The calcuated stresses falls between that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress should be well below maximum allowable.

σ_{clamped}<<σ_{max} allowable

Structural Analysis for Compartment 2-180-6-V: Shell Plating

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\text{fr}^3}$$

Density of Perma Ballast®

%Perma_fill := .95

Based on 95% Fillable Volume

Conversion Factors

$$\rho_{SW} := 64.4 \frac{lb}{ft^3}$$

Density of Seawater

Compartment 2-180-6-V

Volume :=
$$6273$$
ft³

Volume of Compartment

 $W_{Perma} := 560 ton$

Weight of Permanent Ballast

Ship Motion Factors:

Ship Motion Factors. $V := 1.25 \qquad \text{Vertical } \alpha := a \sin \left(\frac{11.5}{15.13} \right) \qquad \alpha = 49.47 \text{ ldeg}$ $A := 0.75 \qquad \text{Athwartship} \qquad \beta := a \sin \left(\frac{9.83}{15.13} \right) \qquad \beta = 40.519 \text{deg}$

$$A := 0.75$$

F := 0.4

Fore/Aft

Loading on Shell Plating:

W_{Perma} = 1.254× 10⁶ lb Weight of <u>Perma Ballast®</u>

$$F_{V} := V \cdot W_{Perma}$$

$$F_V := V \cdot W_{Perma}$$
 $F_V = 1.568 \times 10^6 lb$

Vertical Load (Downward)

$$F_A := A \cdot W_{Perma}$$

$$F_{A} := A \cdot W_{Perma} \qquad F_{A} = 9.408 \times 10^{5} \, lb$$

Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$

$$F_F := F \cdot W_{Perma}$$
 $F_F = 5.018 \times 10^5 \text{ lb}$

Fore/Aft Load

$$F_N := F_V \cdot \cos(\beta) + F_A \cdot \cos(\alpha)$$
 $F_N = 1.803 \times 10^6 \text{ lb}$

Normal Force

Resulting Pressure on Sponson Plating:

Compartment Dimensions:

$$Ht := 15.13ft$$

Area :=
$$L \cdot H$$

Area :=
$$L \cdot Ht$$
 Area = $1.452 \times 10^3 \text{ ft}^2$

$$P := \frac{F_N}{Area}$$

$$P := \frac{F_N}{Area}$$
 $P = 1.242 \times 10^3 \frac{lb}{ft^2}$ $P = 8.622 \frac{lb}{in^2}$

$$P = 8.622 \frac{lb}{lm^2}$$

$$P_{95\%} := \%_{Perma_fill} \cdot P$$

$$P_{95\%} = 1.179 \times 10^3 \frac{11}{5}$$

$$P_{95\%} := \%_{Perma_fill} \cdot P \quad P_{95\%} = 1.179 \times 10^3 \frac{lb}{ft^2}$$
 Based on 95% Fillable Volume

$$H := \frac{P_{95\%}}{\rho_{sw}}$$

$$H = 18.315 ft$$

In accordance with CVN 76 Specs, sponson plating aft of Frame 88 shall be designed for 1000 psf. Therefore, less ballast must be added to these sponson voids.

1179 psf > 1000 psf

Design of Plating for Surface Ships, IAW General Specs (Section 100)

Panels of plating shall be proportioned so as not to exceed the breadth-thickness ratios found below:

$$AR := \frac{b}{a} \qquad AR = 0.108$$

$$AR = 0.108$$

tactual := .25in Thickness of Plate, HTS

H = 18.315ft

Head of Water

K := 1

For AR = 0.108, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

$$\frac{b}{t} \leq \frac{C}{K \cdot \sqrt{H}}$$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{\min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$$

Note: The stiffeners are assumed to be spaced evenly across

the sponson shell plating with 6 stiffeners spanning 15.13'.

 $t_{\min} = 0.278$ in

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .28 in. The actual plate thickness is .25 in. Therefore, the 2nd deck shell plating must either be stiffened or have less ballast added. It should be noted,

however, that some structural margin exists within these calculations and that .28 is close enough to .25 in. As a result, it is possible that stiffening is not required for the 2nd deck shell sponson plating.

t_min > t_actual

Calculations to Determine Maximum Ballast Weight:

Ship Motion Factors:

V := 1.25

Vertical

A := 0.75

Athwartship

F := 0.4

Fore/Aft

Resulting Pressure on Sponson Plating:

Compartment Dimensions:

Ht := 15.13ft

 $P := 1000 \cdot \frac{lb}{c^2}$ Maximum pressure allowed for sponson shell plating IAW CVN 76 Specs

Area :=
$$L \cdot Ht$$
 Area = $1.452 \times 10^3 \text{ ft}^2$

$$F_{N \text{ new}} := P \cdot Area$$

$$F_{N_new} := P \cdot Area$$
 $F_{N_new} = 1.452 \times 10^6 lb$

$$H := \frac{P}{\rho_{ev}}$$

 $H := \frac{P}{\rho_{SW}}$ H = 15.528ft New Equivalent Head

Loading on Shell Plating:

$$\alpha := a \sin \left(\frac{11.5}{15.13} \right)$$

$$\alpha = 49.47 \, \text{ldeg}$$

$$\alpha := a \sin \left(\frac{11.5}{15.13}\right) \qquad \alpha = 49.47 \text{ ldeg}$$

$$\beta := a \sin \left(\frac{9.83}{15.13}\right) \qquad \beta = 40.519 \text{deg}$$

$$\beta = 40.519 \text{deg}$$

$$F_{N_new} = F_{V_new} \cdot \cos(\beta) + F_{A_new} \cdot \cos(\alpha)$$

$$F_{V_new} = V \cdot W_{Perma}$$

 $F_{V_new} = V \cdot W_{Perma}$ Vertical Load (Downward)

$$F_{A_new} = A \cdot W_{Perma}$$
 Athwartship Load (Port)

$$F_{N_new} = V \cdot W_{Perma} \cdot \cos(\beta) + A \cdot W_{Perma} \cdot \cos(\alpha)$$

$$W_{Perma} := \frac{F_{N_new}}{(v \cdot \cos(\beta) + A \cdot \cos(\alpha))}$$

$$W_{Perma} = 451.047$$
ltor

W_{Perma} = 451.047/ton Weight of <u>Perma Ballast®</u> that cannot be exceeded

Calculations to Determine Strength Corrections:

 $t_{min} := .25in$ Thickness of Plate, HTS

H = 15.528ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1 Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{\text{limit}} := t_{\text{min}} \cdot \frac{C}{\frac{1}{2}}$$
 $K \cdot H^{2}$
 $b = 26 \text{ in}$
 $\frac{b}{2} = 13 \text{ in}$

 $b_{limit} = 25.377in$

25.377 in is the maximum distance that the stiffeners can be apart and have no plastic deformation to the existing .25 in plating. The current stiffeners are located 26 in apart. It is recommended that the stiffeners be placed less than maximum distance apart. Therefore, adding a vertical stiffener in the middle of each existing stiffener on the shell sponson plating will put b at 13 in and meet the 25.377 in requirement.

Results

A combination of Lesser Weight Addition to meet the 1000psf limit along with adding a stiffener in the middle of each existing stiffener will alleviate any structural concerns.

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2 p$$
 = pressure based on 1000 psf on sponson shell plating

b = panel widtha = panel length t = plate thickness

$$k_{\text{simply_supported}} := .75$$

$$k_{clamped} := .5$$

k values are most conservative

based on 13 in separation between stiffeners

$$P = 6.944 \frac{lb}{in^2}$$
 $t := .25in$

$$\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$$

 $\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$ IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{simply_supported} = 14083 \frac{lb}{in^2}$$

$$\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{clamped}} = 9389 \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Stress for HTS is 40,000 psi. By adding stiffeners at 13 in, the stress falls well below that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress should be well below maximum allowable.

$$\sigma_{clamped} << \sigma_{max_allowable}$$

Structural Analysis for Compartment 4-165-4-V: Deck Plating

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\text{ft}^3}$$

Density of Perma Ballast®

%Perma fill ≔ .95

Based on 95% Fillable Volume

Conversion Factors

$$\rho_{sw} := 64.4 \frac{lb}{ft^3}$$

Density of Seawater

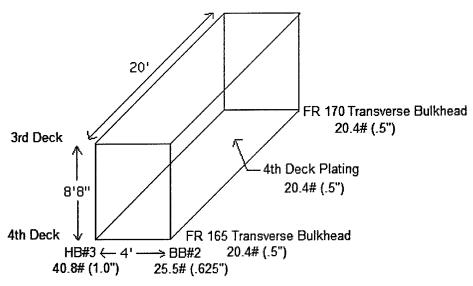
Compartment 4-165-4-V

Volume :=
$$700$$
ft³

Volume of Compartment

W_{Perma} := 62lton

Weight of Permanent Ballast



Ship Motion Factors:

Vertical

A := 0.75

Athwartship

F := 0.4

Fore/Aft

Loading on Deck Plating:

$W_{Perma} = 1.389 \times 10^5 lb$ Weight of <u>Perma Ballast®</u>

$$F_V := V \cdot W_{Perma}$$

$$F_V = 1.736 \times 10^5 \, lb$$

Vertical Load (Downward)

$$F_{\Lambda} := A \cdot W_{Perma}$$

$$F_A := A \cdot W_{Perma}$$
 $F_A = 1.042 \times 10^5 \text{ lb}$ $F_F := F \cdot W_{Perma}$ $F_F = 5.555 \times 10^4 \text{ lb}$

Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$

$$F_F = 5.555 \times 10^4 \text{ l}$$

Fore/Aft Load

$$F_{\mathbf{N}} := F_{\mathbf{V}}$$

$$F_N = 1.736 \times 10^5 \, lb$$

Normal Force

Resulting Pressure on Deck Plating:

Compartment Dimensions:

$$W := 4ft$$

Area =
$$80 \text{ft}^2$$

$$P := \frac{F_N}{I}$$

Area := L·W Area =
$$80 \text{ft}^2$$

$$P := \frac{F_N}{\text{Area}} \qquad P = 2.17 \times 10^3 \frac{\text{lb}}{\text{ft}^2} \qquad P = 15.069 \frac{\text{lb}}{\text{in}^2}$$

$$P = 15.069 \frac{lb}{lm^2}$$

$$P_{95\%} := \%_{Perma_fill}$$

$$P_{95\%} = 1.007 \times 10^4$$

 $P_{95\%} := \%_{Perma_fill}P$ $P_{95\%} = 1.007 \times 10^4 \frac{kg}{m^2}$ Based on 95% Fillable Volume

$$H := \frac{P_{95\%}}{\rho_{sw}}$$

$$H = 32.01 \, lft$$

Equivalent Head

The actual head for flooding of the 4th deck to the 6" above the 2nd deck (DC Deck) will result in a head of 27.7'. This value is below the head calculated above of 32'. As a result, the deck plating must be stiffened.

$$32 \text{ ft} > 27.7 \text{ ft}$$

Design of Plating for Surface Ships, IAW General Specs (Section 100)

$$b := 48in$$

$$AR := \frac{b}{a}$$
 $AR = 1$

$$AR = 1$$

t_{actual} := .50in Thickness of Plate, OS

H := 27.7ft

Maximum Head of Water for 4th Deck (which really implies permanent set and may still not be conservative

K := .78

For AR = 1, IAW with Table I of the General Specs (Section 100)

 $C := 350 \text{ft}^{.5}$

For OS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{\min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$$

$$t_{\min} = 0.563$$
in

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .563 in. The actual plate thickness is .5 in. Therefore, the 4th deck shell plating must either be stiffened or have less ballast added.

t_min >> t_actual

Calculations to Determine Maximum Ballast Weight:

Ship Motion Factors:

V := 1.25

Vertical

A := 0.75

Athwartship

F := 0.4

Fore/Aft

Resulting Pressure on Deck Plating:

Compartment Dimensions:

$$L := 20ft$$

$$W := 4ft$$

Area :=
$$L \cdot W$$

Area =
$$80 \text{ft}^2$$

H = 27.7ft

Based on Maximum Equivalent Head or Design Load

$$P_{\text{new}} := H \cdot \rho_{\text{sw}} \quad P_{\text{new}} = 12.388 \frac{\text{lb}}{\text{in}^2}$$

$$F_{\text{N new}} := P_{\text{new}} \cdot Area$$

$$F_{N_{new}} = 1.427 \times 10^5 \, lb$$

Loading on Deck Plating:

$$F_{N_{new}} = F_{V_{new}}$$

$$F_{V_new} = V \cdot W_{Perma}$$

Vertical Load (Downward)

$$W_{Perma} \coloneqq \frac{F_{N_new}}{V}$$

$$W_{Perma} = 50.968lton$$

Weight of Perma Ballast® that cannot be exceeded

Calculations to Determine Strength Corrections:

t_{min} := .5in Thickness of Plate, OS

H = 27.7ft Head of Water

K := .78 Assume AR remains 1

 $C := 350 \text{ft}^{.5}$ For OS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \sqrt{H}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{limit} := t_{min} \cdot \frac{C}{\frac{1}{K \cdot H^2}}$$

 $b_{limit} = 42.629in$

42.629" is the maximum distance that the stiffeners can be apart and have no elastic deformation to the .5 in plating. The current stiffeners are located 40 in apart. Therefore, according to the Navy Formula, adding a stiffener to the deck plating will not be required IF 50 ltons or less of ballast is added.

Results

Either less ballast must be added to the 4th deck tanks OR stiffening of the deck plating must occur to be less than design load. The design load is the calculated head of water for that deck. Similarly, adding a stiffener down the middle of the deck plating would alleviate any structural concerns. Adding a stiffener cannot, however, be done due to the positioning of the plating. Welding would be required on the bottom deck plating and a watch would need to be stationed on the other side of the deck. The tank below this 4th deck tank is foam filled and a fire watch cannot be stationed. Therefore, the 4th Deck is no longer an option unless the ballast weight to be added be decreased to less than 50 ltons.

Stress Calculations: No Stiffening

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$
 p = pressure based on 50 Itons of ballast addition

b = panel width a = panel length

t = plate thickness

$$k_{simply_supported} := .75$$
 $k_{clamped} := .5$ k values are most conservative $a := 20$ ft $b := 48$ in based on no stiffener addition

$$a := 20$$
ft $b := 48$ in based on no stiffener addition $P_{new} = 12.388 \frac{lb}{in^2}$ $t := .5$ in

$$\sigma_{\text{max_allowable}} = 28,000 \frac{\text{lb}}{\text{in}^2}$$
 IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P_{\text{new}} \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{simply_supported}} = 85626 \frac{\text{lb}}{\text{in}^2}$$

$$\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P_{\text{new}} \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{clamped}} = 57084 \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Stress for OS is 28,000 psi. If no stiffeners are added, the calculated stress is much greater than maximum allowable stress. As a result, stiffeners must still be added to be less than the maximum allowable stress (even with less ballast added and based off design load using maximum head of water).

 $\sigma_{clamped} >> \sigma_{max_allowable}$

Stress Calculations: Stiffening

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P_{new} \cdot \left(\frac{b}{t}\right)^2$$

$$p = \text{pressure based on 50 ltons of ballast addition}$$

$$b = \text{panel width}$$

$$a = \text{panel length}$$

$$t = \text{plate thickness}$$

$$k_{simply_supported} := .75$$
 $k_{clamped} := .5$ k values are most conservative $a := 20$ ft $b := 24$ in based on adding a stiffener down the middle of the deck

a := 20ft b := 24in

$$P_{\text{new}} = 12.388 \frac{\text{lb}}{\text{in}^2}$$

$$t := .5in$$

$$\sigma_{\text{max_allowable}} = 28,000 \frac{\text{lb}}{\text{in}^2}$$
 IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P_{\text{new}} \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{simply_supported}} = 21407 \frac{\text{lb}}{\text{in}^2}$$

$$\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P_{\text{new}} \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{clamped}} = 14271 \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Stress for OS is 28,000 psi. By adding stiffeners at 24 in AND with the addition of less ballast, the stress falls below that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress should be well below maximum allowable. As mentioned previously, however, adding ballast to the 4th deck is not feasible due to the inability to stiffen the 4th deck.

Structural Analysis for Compartment 8-225-6-V: Shell Plating

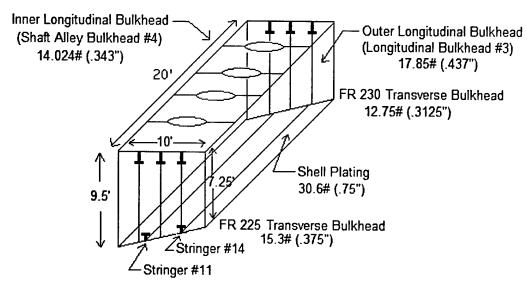
Perma Ballast® Data

$$\rho_{Perma} := 200 \frac{lb}{c^3}$$
Density of Perma Ballast®

Conversion Factors lton := 2240b

$$\rho_{sw} := 64.4 \frac{lb}{ft^3}$$

Compartment 8-225-6-V



Ship Motion Factors:

$$A := 0.75$$
 Athwartship

$$F := 0.4$$
 Fore/Aft

Loading on Shell Plating:

 $W_{Perma} = 5.622 \times 10^5 lb$ Weight of <u>Perma Ballast®</u>

$$F_V := V \cdot W_{Perma}$$
 $F_V = 7.028 \times 10^5 \text{ lb}$ Vertical Load (Downward)

$$F_A := A \cdot W_{Perma}$$
 $F_A = 4.217 \times 10^5 \text{ lb}$ Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$
 $F_F = 2.249 \times 10^5 \text{ lb}$ Fore/Aft Load

$$F_A := A \cdot W_{Perma}$$
 $F_A = 4.217 \times 10^5 \text{ lb}$ Atle
 $F_F := F \cdot W_{Perma}$ $F_F = 2.249 \times 10^5 \text{ lb}$ For $\beta := asin\left(\frac{27}{123}\right)$ $\beta = 12.68deg$

$$\alpha := 90 \text{deg} - \beta \qquad \alpha = 77.32 \text{deg}$$

$$F_N := F_V \cos(\beta) + F_A \cdot \cos(\alpha)$$
 $F_N = 7.782 \times 10^5 \text{ lb}$ Normal Force

Resulting Pressure on Shell Plating:

Compartment Dimensions:

$$Area = 205ft^2$$

$$P := \frac{F_N}{-}$$

Area := L·Ht Area =
$$205\text{ft}^2$$

 $P := \frac{F_N}{\text{Area}}$ $P = 3.796 \times 10^3 \frac{\text{lb}}{\text{ft}^2}$ $P = 26.363 \frac{\text{lb}}{\text{in}^2}$

$$P = 26.363 \frac{lb}{in^2}$$

$$P_{95\%} := \%_{Perma_filf} P$$

$$P_{95\%} = 3.606 \times 10^3$$

 $P_{95\%} := \%_{Perma_fill}P$ $P_{95\%} = 3.606 \times 10^3 \frac{lb}{ft^2}$ Based on 95% Fillable Volume

$$H := \frac{P_{95\%}}{\rho_{sw}} \qquad \qquad H = 56 ft \qquad \qquad Equivalent \ Head$$

$$H = 56ft$$

Design of Plating for Surface Ships, IAW General Specs (Section 100)

$$AR := \frac{b}{a}$$

$$AR = 0.171$$

t_{actual} := .75in Thickness of Plate, HTS

H = 56ft

Head of Water

K := 1

For AR = 0.171, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Se, IAW with Table I of the General Specs (Section 100)t

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

$$\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$$

$$t_{min} = 0.767in$$

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .767 in. The actual plate thickness is .75 in. Therefore, the 8th deck shell plating must either be stiffened or have less ballast added.

t_min >> t_actual

Calculations to Determine Strength Corrections:

 $t_{\min} := .75in$

Thickness of Plate, HTS

H = 56ft

Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1

Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$

For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{\text{limit}} := t_{\text{min}} \cdot \frac{C}{\frac{1}{2}}$$

$$K.H^{2}$$

 $b_{limit} = 40.089in$

40.089 in is the maximum distance that longitudinal stiffeners can be located apart and have no deformation occur to the .75 in plating. The shell plating has 2 longitudinal stiffeners or stringers located appropriately 41 in apart. Therefore, stiffening of the existing shell plating will be required for 251 ltons of ballast.

Calculations to Determine Maximum Ballast Weight without stiffening:

 $t_{\min} := .75in$

Thickness of Plate, HTS

 $b_{limit} = 41in$

Short Dimension of Panel

K := 1

Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$

For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$H_{\text{new}} := \left(\frac{C \cdot t_{\text{min}}}{K \cdot b_{\text{limit}}}\right)^2$$
 $H_{\text{new}} = 53.54 \text{ft}$

$$H_{\text{new}} = 53.54 \text{ft}$$

$$P_{new} := H_{new} \cdot \rho_{sw}$$

$$P_{\text{new}} := H_{\text{new}} \cdot \rho_{\text{sw}}$$
 $P_{\text{new}} = 23.944 \frac{\text{lb}}{\text{in}^2}$

$$F_{N_{new}} := P_{new} \cdot Area$$

$$F_{N_new} := P_{new} \cdot Area$$
 $F_{N_new} = 7.068 \times 10^5 lb$

$$F_{N_new} = F_{V_new} \cdot \cos(\beta) + F_{A_new} \cdot \cos(\alpha)$$

$$F_{V_{new}} = V \cdot W_{Perma}$$

 $F_{V \text{ new}} = V \cdot W_{Perma}$ Vertical Load (Downward)

$$F_{A_new} = A \cdot W_{Perma}$$
 Athwartship Load (Port)

$$F_{N_new} = V \cdot W_{Perma} \cdot \cos(\beta) + A \cdot W_{Perma} \cdot \cos(\alpha)$$

$$W_{Perma} := \frac{F_{N_new}}{(V \cdot \cos(\beta) + A \cdot \cos(\alpha))}$$

 $W_{Perma} = 227.974$ ton

Weight of Perma Ballast® that cannot be exceeded so that stiffening of the 8th deck shell plating will not be required.

Stress Calculations for 227 Itons of ballast:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P_{\text{new}} \cdot \left(\frac{b}{t}\right)^2$$

$$p = \text{pressure based on limiting ballast add for shell plating calculations}$$

$$b = \text{panel width}$$

a = panel length t = plate thickness

 $k_{\text{simply_supported}} := .75$ $k_{\text{clamped}} := .5$ $k_{\text{values are most conservative}}$ a := 20 ft b := 4 lin based on 41 in separation between the sequence of the sequence based on 41 in separation between stiffeners

 $\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$ IAW with the General Specs (Section 100)

 $\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P_{\text{new}} \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{simply_supported}} = 53667 \frac{\text{lb}}{\text{in}^2}$

 $\sigma_{clamped} := k_{clamped} \cdot P_{new} \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{clamped}} = 35778 \frac{\text{lb}}{\text{in}^2}$

Maximum Allowable Stress for HTS is 40,000 psi. Without any stiffener addition, the calculated stress falls between that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress falls below maximum allowable.

^σclamped^{< σ}max allowable

Structural Analysis for Compartment 8-225-6-V: Inner Longitudinal **Bulkhead Plating**

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\text{ft}^3}$$

Density of Perma Ballast®

%Perma fill := .95

Based on 95% Fillable Volume

Conversion Factors

Iton := 2240b

$$\rho_{sw} := 64.4 \frac{lb}{ft^3}$$

Density of Seawater

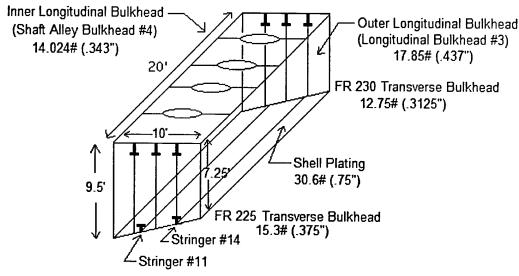
Compartment 8-225-6-V

Volume := 2814ft³

Volume of Compartment

 $W_{Perma} := 227 Iton$

Weight of Permanent Ballast based off limiting shell calculations



Ship Motion Factors:

Vertical

A := 0.75

Athwartship

F := 0.4

Fore/Aft

Loading on Inner Longitudinal Bulkhead Plating:

W_{Perma} = 5.085× 10⁵ lb Weight of <u>Perma Ballast®</u>

$$F_{\mathbf{V}} := \mathbf{V} \cdot \mathbf{W}_{\mathbf{Perma}}$$

$$F_V = 6.356 \times 10^5 \, \text{lb}$$

Vertical Load (Downward)

$$F_A := A \cdot W_{Perma}$$

$$F_A := A \cdot W_{Perma}$$
 $F_A = 3.814 \times 10^5 lb$

Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$

$$F_F = 2.034 \times 10^5 \, \text{lb}$$

Fore/Aft Load

$$F_N := F_A$$

$$F_N := F_A$$
 $F_N = 3.814 \times 10^5 \, lb$

Normal Force

Resulting Pressure on Inner Longitudinal Bulkhead Plating:

Compartment Dimensions:

Area := L·Ht Area =
$$190 \text{ft}^2$$

$$P := \frac{F_N}{\text{Area}} \qquad P = 2.007 \times 10^3 \frac{\text{lb}}{\text{ft}^2} \qquad P = 13.939 \frac{\text{lb}}{\text{in}^2}$$

$$P = 13.939 \frac{lb}{in^2}$$

$$H := \frac{P}{\rho_{SW}} \qquad \qquad H = 31.167 ft$$

$$H = 31.167ft$$

Equivalent Head

Design of Plating for Surface Ships, IAW General Specs (Section 100)

$$AR := \frac{b}{a} \qquad AR = 0.421$$

$$AR = 0.421$$

tactual := .343in Thickness of Plate, HTS

H = 31.167ft

Head of Water

K := 1

For AR = 0.421, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{min} := \frac{K \cdot \sqrt{H \cdot b}}{C}$$

$$t_{\min} = 0.67$$
in

The 8th deck inner longitudinal bulkhead (Shaft Alley Bulkhead #4) currently has 4 vertical stiffeners spaced 4 ft apart. The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .67 in. The actual plate thickness is .343 in. Therefore, the 8th deck inner longitudinal bulkhead plating must be stiffened.

t_min >> t_actual

Calculations to Determine Strength Corrections:

 $t_{min} = .343in$ Thickness of Plate, HTS

H = 31.167ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1 Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{\text{limit}} := t_{\text{min}} \cdot \frac{C}{\frac{1}{2}}$$
 $b = 48 \text{ in}$ $\frac{b}{2} = 24 \text{ in}$ $K \cdot H^2$

 $b_{\text{limit}} = 24.576 \text{in}$

24.576 in is the maximum distance that stiffeners can be located apart and have no deformation occur to the .343 in plating. The inner longitudinal bulkhead plating currently has 4 vertical stiffeners spaced 4 ft apart. It is recommended that a vertical stiffener be placed in the middle of each existing stiffener on the inner longitudinal bulkhead. Therefore, adding a total of 5 stiffeners to the inner longitudinal bulkhead will put b at approximately 24 in and meet the 24.576 in requirement.

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$

p = pressure based on limiting ballast add for shell plating calculations

b = panel width a = panel length t = plate thickness

 $k_{\text{simply_supported}} := .75$

 $k_{clamped} := .5$ k values are most conservative

a := 114in b := 24in

based on 24 in separation between stiffeners

 $P = 13.939 \frac{lb}{in^2}$ t := .343in

 $\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$

IAW with the General Specs (Section 100)

 $\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{simply_supported}} = 51182 \frac{\text{lb}}{\text{in}^2}$

 $\sigma_{clamped} \coloneqq k_{clamped} \cdot P \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{clamped}} = 34121 \frac{\text{lb}}{\text{in}^2}$

Maximum Allowable Stress for HTS is 40,000 psi. By adding vertical stiffeners at 24 in, this stress falls between that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress falls well below maximum allowable.

 $\sigma_{clamped} < \sigma_{max_allowable}$

Structural Analysis for Compartment 8-225-6-V: Outer Longitudinal Bulkhead Plating

Perma Ballast® Data

$$\rho_{Perma} := 200 \frac{1b}{rt^3}$$

Density of Perma Ballast®

%Perma fill := .95

Based on 95% Fillable Volume

Conversion Factors

$$\rho_{sw} := 64.4 \frac{lb}{ft^3}$$

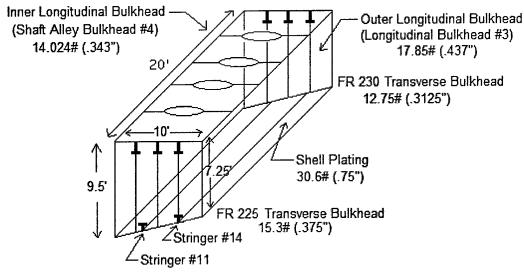
Density of Seawater

Compartment 8-225-6-V

Volume of Compartment

 $W_{Perma} := 227 lton$

Weight of Permanent Ballast



Ship Motion Factors:

V := 1.25

Vertical

A := 0.75

Athwartship

F := 0.4

Fore/Aft

Loading on Outer Longitudinal Bulkhead Plating:

$W_{Perma} = 5.085 \times 10^5 lb$ Weight of <u>Perma Ballast®</u>

$$F_V := V \cdot W_{Perma}$$

$$F_V := V \cdot W_{Perma}$$
 $F_V = 6.356 \times 10^5 lb$

Vertical Load (Downward)

$$F_A := A \cdot W_{Perma}$$

$$F_A := A \cdot W_{Perma}$$
 $F_A = 3.814 \times 10^5 \text{ lb}$ $F_F := F \cdot W_{Perma}$ $F_F = 2.034 \times 10^5 \text{ lb}$

Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$

$$F_F = 2.034 \times 10^5 \, lb$$

Fore/Aft Load

$$F_N := F_A$$

$$F_N := F_A$$
 $F_N = 3.814 \times 10^5 \, lb$

Normal Force

Resulting Pressure on Outer Longitudinal Bulkhead Plating:

Compartment Dimensions:

$$L := 20ft$$

$$Area = 145 ft^2$$

$$P := \frac{F_N}{Area}$$

Area := L·Ht Area =
$$145$$
ft²

$$P := \frac{F_N}{Area} \qquad P = 2.63 \times 10^3 \frac{lb}{ft^2} \qquad P = 18.264 \frac{lb}{in^2}$$

$$P = 18.264 \frac{lb}{in^2}$$

$$H:=\frac{P}{\rho_{SW}} \qquad \qquad H=40.84 ft$$

$$H = 40.84ft$$

Equivalent Head

Design of Plating for Surface Ships, IAW General Specs (Section 100)

$$b := 48in$$

$$AR := \frac{b}{a} \qquad AR = 0.552$$

$$AR = 0.552$$

tactual := .437in Thickness of Plate, HTS

$$H = 40.84ft$$

Head of Water

$$K := .99$$

For AR = 0.552, IAW with Table I of the General Specs (Section 100)

$$C := 400 \text{ft}^{.5}$$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

$$\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$$

$$t_{\min} = 0.759$$
in

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .759 in. The actual plate thickness is .437 in. Therefore, the 8th deck outer longitudinal bulkhead plating must be stiffened.

t_min >> t_actual

Calculations to Determine Strength Corrections:

t_{min} := .437/in Thickness of Plate, HTS

H = 40.84ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := .9 Assume AR remains less than .552

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{\text{limit}} := t_{\text{min}} \cdot \frac{C}{\frac{1}{2}}$$
 $b = 48 \text{ in}$ $\frac{b}{2} = 24 \text{ in}$

 $b_{limit} = 30.392in$

30.392 in is the maximum distance that stiffeners can be located apart and have no deformation occur to the .437 in plating. The outer longitudinal bulkhead plating currently has 4 vertical stiffeners spaced 4 ft apart. It is recommended that a vertical stiffener be placed in the middle of each existing stiffener on the outer longitudinal bulkhead. Therefore, adding a total of 5 stiffeners to the outer longitudinal bulkhead will put b at approximately 24 in and meet the 30.392 in requirement.

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$

p = pressure based on limiting ballast add for shell plating calculations

b = panel width

a = panel length t = plate thickness

 $k_{simply_supported} := .75$

 $k_{clamped} := .5$ k values are most conservative

based on 24 in separation between stiffeners

a := 20ft b := 24in $P = 18.264 \frac{lb}{in^2}$ t := .437in

 $\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$ IAW with the General Specs (Section 100)

 $\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{simply_supported}} = 41317 \frac{\text{lb}}{\text{in}^2}$

 $\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{clamped}} = 27544 \frac{\text{lb}}{\text{in}^2}$

Maximum Allowable Stress for HTS is 40,000 psi. By adding vertical stiffeners at 24 in, this stress falls below that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress falls well below maximum allowable.

^σclamped^{<< σ}max allowable

Structural Analysis for Compartment 8-225-6-V: Transverse Bulkhead Frame 225

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\text{ft}^3}$$

Density of Perma Ballast®

%Perma_fill := .95

Based on 95% Fillable Volume

Conversion Factors

lton := 2240b

$$\rho_{sw} := 64.4 \frac{lb}{ft^3}$$

Density of Seawater

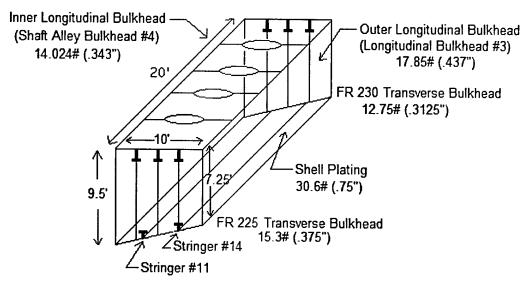
Compartment 8-225-6-V

Volume := 2814ft³

Volume of Compartment

 $W_{Perma} := 227 lton$

Weight of Permanent Ballast



Ship Motion Factors:

V := 1.25

Vertical

A := 0.75

Athwartship

F := 0.4

Fore/Aft

Loading on Transverse Bulkhead Plating:

 $W_{Perma} = 5.085 \times 10^5 lb$ Weight of <u>Perma Ballast®</u>

$$F_V := V \cdot W_{Perma}$$

$$F_V = 6.356 \times 10^5 \, lb$$

Vertical Load (Downward)

$$F_A := A \cdot W_D$$

$$F_A := A \cdot W_{Perma}$$
 $F_A = 3.814 \times 10^5 \text{ lb}$

Athwartship Load (Port)

$$F_F := F \cdot W_{Perma}$$

$$F_F = 2.034 \times 10^5 \, lb$$

Fore/Aft Load

$$F_N := F_F$$

$$F_N = 2.034 \times 10^5 \, lb$$

Normal Force

Resulting Pressure on Transverse Bulkhead Plating:

Compartment Dimensions:

Width := 120in Dimensions of rectangular section Ht := 114in

$$\beta := a\sin\left(\frac{114 - 87}{123}\right) \qquad \beta = 12.68 deg$$

base := 120in Dimensions of small triangle

ht :=
$$\sin(\beta)$$
-base ht = 26.341in

$$ht = 26.341ir$$

Area := $Ht \cdot Width - (.5 \cdot base \cdot ht)$

Area =
$$1.21 \times 10^4 \text{ in}^2$$

$$P := \frac{F_{N}}{Area} \qquad P = 2.421 \times 10^{3} \frac{lb}{ft^{2}} \qquad P = 16.81 \frac{lb}{in^{2}}$$

$$H := \frac{P}{\rho_{sw}} \qquad H = 37.587ft \qquad Equivalent Head$$

$$P = 16.81 \frac{lb}{in^2}$$

$$H := \frac{P}{\rho_{SW}} \qquad H$$

$$H = 37.587f$$

Design of Plating for Surface Ships, IAW General Specs (Section 100)

The transverse bulkhead is trapezoidal in shape with 3 vertical stiffeners at Frames 225 and 230 respectively. The transverse bulkhead is evenly divided into 4 rectangular panels so that a cylindrical plate bending analysis can be performed on each panel.

base := 30in

Dimensions of small triangle

ht = 26.341in

ht := $sin(\beta) \cdot base$

b := 30in

 $a_A := 114in$

 $AR_A := \frac{b}{a_A}$

 $AR_A = 0.263$

 $a_B := 114in - ht$

 $a_{B} = 107.415$ in

 $\mathsf{AR}_B \coloneqq \frac{\mathsf{b}}{\mathsf{a}_B}$

 $AR_B = 0.279$

 $a_C := a_B - ht$

 $a_{C} = 100.829in$

 $AR_C := \frac{b}{a_C}$

 $AR_C = 0.298$

 $a_D := a_C - ht$

 $a_{D} = 94.244in$

 $AR_D := \frac{b}{a_D}$

 $AR_{D} = 0.318$

 $t_{actual} := .375in$

Thickness of Plate, HTS

H = 37.587ft

Head of Water

K := 1

For AR < 0.5, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{K \cdot \sqrt{H}}$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$t_{min} := \frac{K \cdot \sqrt{H \cdot b}}{C}$$

$$t_{\min} = 0.46$$
in

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .46 in. The actual plate thickness is .375 in. Therefore, the 8th deck transverse plating at Frame 225 must be stiffened.

t_min >> t_actual

Calculations to Determine Strength Corrections:

 $t_{min} := .375 in$ Thickness of Plate, HTS

H = 37.587ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1 Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \le \frac{C}{V \sqrt{H}}$ K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{\text{limit}} := t_{\text{min}} \cdot \frac{C}{\frac{1}{2}}$$
 $b = 30 \text{ in}$ $\frac{b}{2} = 15 \text{ in}$ $K \cdot H^{\frac{1}{2}}$

 $b_{limit} = 24.466in$

24.466 in is the maximum distance that vertical stiffeners can be located apart and have no deformation occur to the .375 in plating. The transverse bulkhead plating has 3 vertical stiffeners. It is recommended that a stiffener be placed between these existing vertical stiffeners. Therefore, adding 4 stiffeners to the transverse bulkhead plating will put b at approximately 15 in and meet the 24.466 in requirement.

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$
 p = pressure based on limiting ballast add for shell plating calculations
$$b = \text{panel width}$$

$$a = \text{panel length}$$

a = panel lengtht = plate thickness

$$k_{\text{simply_supported}} := .75$$
 $k_{\text{clamped}} := .5$ $k_{\text{values are most conservative}}$ $a := 114\text{in}$ $b := 15 \cdot \text{in}$ based on 15 in separation between stiffeners $P = 16.81 \frac{\text{lb}}{\text{in}^2}$ $t := .375\text{in}$

$$\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$$
 IAW with the General Specs (Section 100)

$$\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{simply_supported}} = 20172 \frac{\text{lb}}{\text{in}^2}$$

$$\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P \cdot \left(\frac{b}{t}\right)^2$$

$$\sigma_{\text{clamped}} = 13448 \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Stress for HTS is 40,000 psi. By adding stiffeners at 15 in, this stress falls below that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress falls well below maximum allowable.

$$\sigma_{clamped} << \sigma_{max_allowable}$$

Structural Analysis for Compartment 8-225-6-V: Transverse Bulkhead Frame 230

Perma Ballast® Data

$$\rho_{\text{Perma}} := 200 \frac{\text{lb}}{\alpha^3}$$

Density of Perma Ballast®

%Perma fill := .95

Based on 95% Fillable Volume

Conversion Factors

lton := 2240b

$$\rho_{sw} := 64.4 \frac{lb}{ft^3}$$

Density of Seawater

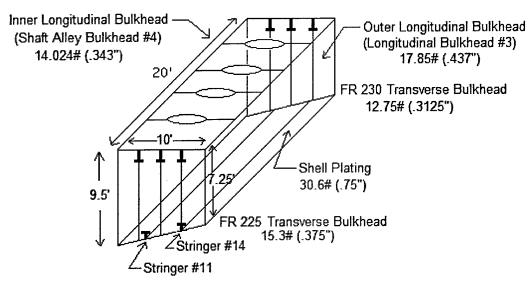
Compartment 8-225-6-V

Volume := 2814ft³

Volume of Compartment

W_{Perma} := 227lton

Weight of Permanent Ballast



Ship Motion Factors:

V := 1.25

Vertical

A := 0.75

Athwartship

F := 0.4

Fore/Aft

Loading on Transverse Bulkhead Plating:

W_{Perma} = 5.085× 10⁵ lb Weight of <u>Perma Ballast®</u>

 $F_{V} := V \cdot W_{Perma}$

 $F_V = 6.356 \times 10^5 \, lb$

Vertical Load (Downward)

 $F_A := A \cdot W_{Perma}$

 $F_A = 3.814 \times 10^5 \, lb$

Athwartship Load (Port)

 $F_F := F \cdot W_{Perma}$

 $F_F = 2.034 \times 10^5 \, lb$

Fore/Aft Load

 $F_N := F_F$

 $F_{N} = 2.034 \times 10^{5} \, \text{lb}$

Normal Force

Resulting Pressure on Transverse Bulkhead Plating:

Compartment Dimensions:

Ht := 114in Dimensions of rectangular section

Width := 120in

$$\beta := a\sin\left(\frac{114 - 87}{123}\right) \qquad \beta = 12.68 deg$$

ht :=
$$sin(\beta)$$
-base ht = 26.341in

Area :=
$$Ht \cdot Width - (.5 \cdot base \cdot ht)$$

Area =
$$1.21 \times 10^4 \text{ in}^2$$

$$P := \frac{F_{N}}{\text{Area}} \qquad P = 2.421 \times 10^{3} \frac{\text{lb}}{\text{ft}^{2}} \qquad P = 16.81 \frac{\text{lb}}{\text{in}^{2}}$$

$$H := \frac{P}{\rho_{SW}} \qquad H = 37.587 \text{ft} \qquad \text{Equivalent Head}$$

$$P = 16.81 \frac{lb}{in^2}$$

$$H := \frac{P}{\rho_{sw}} \qquad H = 37.587ft$$

Design of Plating for Surface Ships, IAW General Specs (Section 100)

The transverse bulkhead is trapezoidal in shape with 3 vertical stiffeners at Frames 225 and 230 respectively. The transverse bulkhead is evenly divided into 4 rectangular panels so that a cylindrical plate bending analysis can be performed on each panel.

base := 30in

Dimensions of small triangle

ht = 26.341in

 $ht := sin(\beta) \cdot base$

b := 30in

 $a_{\Delta} := 114in$

 $AR_A := \frac{b}{a_A} \qquad AR_A = 0.263$

 $a_{\mathbf{B}} := 114 \text{in} - \text{ht}$ $a_{\mathbf{B}} = 107.415 \text{in}$

 $AR_{B} := \frac{b}{a_{B}} \qquad AR_{B} = 0.279$

 $a_{C} := a_{B} - ht$ $a_{C} = 100.829in$

 $AR_C := \frac{b}{a_C} \qquad AR_C = 0.298$

 $a_D := a_C - ht$ $a_D = 94.244in$

 $AR_{D} := \frac{b}{a_{D}} \qquad AR_{D} = 0.318$

 $t_{actual} := .3125n$

Thickness of Plate, HTS

H = 37.587ft

Head of Water

K := 1

For AR < 0.5, IAW with Table I of the General Specs (Section 100)

 $C := 400 \text{ft}^{.5}$

For HTS, No Set, IAW with Table I of the General Specs (Section 100)

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \leq \frac{C}{K \cdot \sqrt{H}}$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

 $t_{\min} := \frac{K \cdot \sqrt{H} \cdot b}{C}$

 $t_{\min} = 0.46$ in

The minimum thickness calculated for this section of plate to meet Navy Structural Standards with no set is .46 in. The actual plate thickness is .3125 in. Therefore, the 8th deck transverse plating at Frame 230 must be stiffened.

t_min >> t_actual

Calculations to Determine Strength Corrections:

 $t_{min} := .3125n$ Thickness of Plate, HTS

H = 37.587ft Head of Water (With new head of water calculated using maximum allowable ballast addition)

K := 1 Assume AR remains less than .5

 $C := 400 \text{ft}^{.5}$ For HTS, No Set

b = short dimension of the panel (inches)

t = thickness of the plate (inches)

C = Coefficient that is a function of the plating material and the location of the plating on the ship

 $\frac{b}{t} \leq \frac{C}{K \cdot \sqrt{H}}$

K = coefficient that depends on the aspect ratio (AR) of the panel

H = Head of salt water (feet)

$$b_{\text{limit}} := t_{\text{min}} \cdot \frac{C}{\frac{1}{2}}$$
 $b = 30 \text{ in}$ $\frac{b}{2} = 15 \text{ in}$ $K \cdot H^{\frac{1}{2}}$

 $b_{limit} = 20.389$ in

20.389 in is the maximum distance that vertical stiffeners can be located apart and have no deformation occur to the .3125 in plating. The transverse bulkhead plating has 3 vertical stiffeners. It is recommended that a stiffener be placed between these existing vertical stiffeners. Therefore, adding 4 stiffeners to the transverse bulkhead plating will put b at approximately 15 in and meet the 20.389 in requirement.

Stress Calculations:

k = coefficient that depends on the plate edge conditions, aspect ratio (AR) of the panel, and position of point being considered

$$\sigma := k \cdot P \cdot \left(\frac{b}{t}\right)^2$$
 p = pressure based on limiting ballast add for shell plating calculations
$$b = \text{panel width}$$

b = panel widtha = panel length

t = plate thickness

 $k_{simply supported} := .75$ a := 114in b := 15·in

k values are most conservative $k_{clamped} := .5$

based on 15 in separation between stiffeners

 $P = 16.81 \frac{lb}{in^2}$ t := .3125n

 $\sigma_{\text{max_allowable}} = 40,000 \frac{\text{lb}}{\text{in}^2}$ IAW with the General Specs (Section 100)

 $\sigma_{\text{simply_supported}} := k_{\text{simply_supported}} \cdot P \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{simply_supported}} = 29048 \frac{\text{lb}}{\text{in}^2}$

 $\sigma_{\text{clamped}} := k_{\text{clamped}} \cdot P \cdot \left(\frac{b}{t}\right)^2$

 $\sigma_{\text{clamped}} = 19365 \frac{\text{lb}}{\text{in}^2}$

Maximum Allowable Stress for HTS is 40,000 psi. By adding stiffeners at 15 in, this stress falls below that of simply supported and that of clamped. Simply supported and clamped structural cases are idealizations of structural member support illustrating zero stiffness and infinite stiffness, neither of which exists in any real-world structural system. On board ship, structural systems can be conveniently approximated by one or the other case, but in fact have stiffness between one or the other. Navy structural analysis is based off clamped ends, therefore the stress falls well below maximum allowable.

σ_{clamped}<<σ_{max_allowable}

Appendix F: Final POSSE Modeling Results

Tanks		Capacity (200 lbs/ft ³)													Degrees												
					0.5				-		-		1.5		H		2		Н		2.5		Н		3		П
	Volume (ft²)	Density (LT/ft²)	Weight (LT)	Weight (LT) Weight (LT)	Percent (%)	ΔKG	∆ Trim \	∆ Trim Weight (LT)	Percent (%)	∆ KG	∆ Trim V	Weight (LT)	Percent (%)	⊿кс∣	∆ Trim Weight (5	Percent (%)	∆ KG ∆	∆ Trim Weight (נו	Percent (%)	ΔKGΔ	∆ Trim We	Weight (LT)	Percent (%)	∆ KG ∆	∆ Trim
																_			Н			П	H				
2nd Deck						Ц									Н			H	Н				\parallel				
2-165-8-V	3,969	0.0893	354	124	3 €			292	14.14		H	280	79			280	79			280	2.6		Н	280	. 79		
2-180-6-V	6,273	0.0893	290									123	- 22			263	47	H		403	72		H	448	- 80		
2-250-4-V	3,362	0.0893	300								-							H	Ц				H	111	37		
Total (LT)				124	10.2	0.02	100	585	21.6	0.04	15.A	£03	33.2	0.05	-28A	S#3	44.7	0.07	AIA	88	56.2	600	55A	688	69.1	0.11	8 54
Number of Tanks				* (Tank	3,00		20150	- A Tank			A550	2 Tanks			6	2 Tanks		\vdash	. 2	2 Tanks		-		3 Tanks			
						L									_				_			-	-				
8th Deck						L													_				\vdash			-	
8-210-10-V	1,716	0.0893	£			L		138				138	06		H	138	6	\vdash	\vdash	138	06	H	H	138	∘ 06 ⊹		
8-215-8-V	1,238	0.0893	F					66	06			66	- 06		L	66	90		H	66	06		H	66	- 06		Π
8-215-10-V	1,187	0.0893	106	36	06			92	86			96	06			95	06			92	- 30		-	95	06		
8-220-8-V	1,019	0.0893	91	88	06							85	06			S 28	- 06			82	96	-		85	06		
8-220-10-V	737	0.0893	99			Щ						- 29	06			29	06			- 69	06		Н	59	- 06		
8-225-6-V	2,814	0.0893	251													526	- 06	Н					Н	526	- 30	H	
8-225-8-V	1,145	0.0893	102			Ц						95	06							95	- 06		Н	95	- 06		
8-230-4-V	2,096	0.0893	187			Ц										168	06			168	60		Н	168	ે 30		
8-230-6-V	949	0.0893	28			Ц		25	- 06											52	- 06	_		25	- 06		
8-235-6-V	1,491	0.0893	133			Ц									_			H		120	6	Н		120	≥ 06	H	
8-235-8-V	2,073	0.0893	185			Ш														三 291	- 06			167	⊸ 30 ⊸		
Total (LT)				<u>, 11</u>	12.3	-0.02	-27A	384	26.7	-0.05	.58A	- 265	39.2	-0.08	0.87A	:	60.2	-0.13	1,414	1072	74.3	-0.14	1.824	1298	90	4.0	2.21A
Number of Tanks				2 Tanks		_		- 4-Tanks			9230	6 Tanks			Later Control	7 Tanks		\vdash		TO Tanks				11 Tanks			
To secure description of the secure of the s				Marian Carrest Contract		-		The second secon				CONTROL CONTRO		1				1]	1	1				1

Appendix G: Final Cost Estimation Data and Worksheets
Provided by Norfolk Naval Shipyard Structural Engineering and Planning Office (Code 256)

BACK
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ESTIN

DATE NOV 2003

ESTIMATE SHEET

(This work sheet was created for the final costing estimation due to the lesser Iton addition for each 2nd deck void, thus requiring compartment 2-250-4-V to be utilized)

M .													
75K A									211				
99 TS					48				48				
72 R1					22				24				
71 AA								256	256				
64 A6								384	384				
64 A1													
56 P1													
51 E1													
38 YY													
35 GA													
26 W1			112		257	26			425				
17 SC													
24 J1		4							4				
111 S5		10					32		42				
74 YY													
111 S6													
11 S2													
11 S1			96		198	48			342				
WORK TO BE ACCOMPLISHED	STIFFENING FOR 2-250-4-V	OPEN & CLOSE VOID	CUT/INST ACCESSES FOR STIFFENING	4 LOCATIONS	INST SUPPORT STR	CUT/INST ACCESSES (6) 12" FOR FILLING	TEST ACCESSES	PREPARE AND PAINT STRUCT	TOTALS:	TOTAL 1736 MH'S TOTAL 217 MD'S	TOTAL PROD \$139,216	MATERIAL: \$9,371	ADD'L WEIGHT: 4,470 #'S
PC NO.													

MATERIAL WORK SHEET

PERMA BALLAST® INSTALLATION

(This work sheet was created for the final costing estimation due to the lesser Iton addition for each 2nd deck void, thus requiring compartment 2-250-4-V to be utilized)

SHOP	PC #	MATERIAL	STOCK NO.	QTY	SS/PUR	COST
		SPONSON STIFFENING: 2-250-4-V				
11S1	003	5" X 4" ANGLE (7.5#LF)	9520-00-277-5978	225	\$6.46/FT	\$1,454
71AA	004	PAINT				\$1,200
74YY	N/A	BLAST AND PAINT FACILITY				\$4,500
ALL	N/A	CONSUMABLES				\$2,217
		TOTAL MATERIAL COSTS:				\$9,371
		TOTAL MATERIAL WEIGHT:				4,470 #'S
						· · · · · · · · · · · · · · · · · · ·

MATERIAL WORK SHEET

PERMA BALLAST® INSTALLATION

(This work sheet was created for the final costing estimation due to the stiffening required for the 8th deck and is not reflected on the previous estimate page)

SHOP	PC #	MATERIAL	STOCK NO.	QTY	55/PUR	COST
		BHD STIFFENING: 8 th DECK				
11S1	001	5" X 4" ANGLE (7.5#LF)	9520-00-277-5978	120	\$6.46/FT	\$775
71AA	002	PAINT				\$1,000
74YY	N/A	BLAST AND PAINT FACILITY				\$2,400
ALL	N/A	CONSUMABLES				\$250
		TOTAL MATERIAL COSTS:				\$4,425
		TOTAL MATERIAL WEIGHT:				900 #'S
						

OPTION 1 (2ND DECK VOIDS 0.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	0.5°	35%	200LBS/FT ³	\$115,046	\$7,744	3,695 #'S
2-180-6-V	0.5°					
2-250-4-V	0.5°					
TOTALS				\$115,046	\$7,744	3,695 #'S

PERMA BALLAST® INSTALLATION

OPTION 2 (8TH DECK VOIDS 0.5° LIST)

Г	<u> </u>	Τ		Γ		Ι	Ι		Г	Т		Τ
MAT#			S.# 006	S.# 006								1,800 #'S
MAT \$			\$4,425	\$4,425								\$8,850
PROD \$			\$21,186	\$21,186								\$42,372
TYPE			200LBS/FT3	200LBS/FT								-
PERCENT			%06	%06								
LIST	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	0.5°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

OPTION 3 (2ND DECK VOIDS 1.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	1.0°	74%	500LBS/FT ³	\$243,240	\$16,373	7,811 #'S
2-180-6-V	1.0°					
2-250-4-V	1.0°					
TOTALS				\$243,240	\$16,373	7,811 #'S

PERMA BALLAST® INSTALLATION

OPTION 4 (8TH DECK TANKS 1.0° LIST)

#	Ş	Š	Š						S			S,#
MAT#	S.# 006	S.# 006	S.# 006						S.# 006			3,600 #'S
MAT \$	\$4,425	\$4,425	\$4,425						\$4,425			\$17,700
PROD \$	\$21,186	\$21,186	\$21,186						\$21,186			\$84,744
TYPE	500LBS/FT3	200LBS/FT3	500LBS/FT3						200LBS/FT ³			
PERCENT	%06	2606	%06						%06			
LIST	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	1.0°	
TANK	8-210-10-V	8-215-8-V	8-215-10-V	8-220-8-V	8-220-10-V	8-225-6-V	8-225-8-V	8-230-4-V	8-230-6-V	8-235-6-V	8-235-8-V	TOTALS

OPTION 5 (2ND DECK VOIDS 1.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	*LYW	MAT#
2-165-8-V	1.5°	2662	200LBS/FT ³	\$164,352	\$11,063	5,278 #'S
2-180-6-V	1.5°	22%	200LBS/FT3	\$115,710	\$7,788	3,715 #'S
2-250-4-V	1.5°					
TOTALS				\$280,062	\$18,851	8,993 #'S

PERMA BALLAST® INSTALLATION

OPTION 6 (8TH DECK TANKS 1.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
8-210-10-V	1.5°	2606	200LBS/FT3	\$21,186	\$4,425	S.# 006
8-215-8-V	1.5°	%06	200LBS/FT3	\$21,186	\$4,425	S.# 006
8-215-10-V	1.5°	2606	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-220-8-V	1.5°	%06	200LBS/FT	\$21,186	\$4,425	S.# 006
8-220-10-V	1.5°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-225-6-V	1.5°					
8-225-8-V	1.5°	206	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-230-4-V	1.5°					
8-230-6-V	1.5°					
8-235-6-V	1.5°					
8-235-8-V	1.5°					
TOTALS				\$127,116	\$26,550	5,400 #'S

OPTION 7 (2ND DECK VOIDS 2.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	2.0°	2662	200LBS/FT ³	\$164,352	\$11,063	5,278 #'S
2-180-6-V	2.0°	47%	200LBS/FT ³	\$247,200	\$16,638	7,938 #'S
2-250-4-V	2.0°					
TOTALS				\$411,552	\$27,701	13,216 #'S

PERMA BALLAST® INSTALLATION

$\begin{array}{c} \text{OPTION 8} \\ \text{(8^{TH} DECK TANKS 2.0° LIST)} \end{array}$

	LISI	PERCENT	TYPE	PROD \$	MAT \$	MAT #
	2.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
	2.0°	%06	200LBS/FT3	\$21,186	\$4,425	S.# 006
	2.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
	2.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S,# 006
	2.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
	2.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S,# 006
	2.0°					
	2.0°	2606	200LBS/FT ³	\$21,186	\$4,425	S.# 006
	2.0°					
	2.0°					
	2.0°					
TOTALS				\$148,302	\$30,975	S,# 00£'9

OPTION 9 (2ND DECK VOIDS 2.5° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	2.5°	%6L	200LBS/FT ³	\$164,352	\$11,063	5,278 #'S
2-180-6-V	2.5°	72.7%	200LBS/FT3	\$262,979	\$17,701	8,445 #'S
2-250-4-V	2.5°					
TOTALS				\$427,331	\$28,764	13,723 #'S

PERMA BALLAST® INSTALLATION

OPTION 10 (8TH DECK TANKS 2.5° LIST)

PERCENT 90%
%06
200%
%06
2006
2006
%06
26 06
200%
%06

OPTION 11 (2ND DECK VOIDS 3.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
2-165-8-V	3.0°	%6L	ELH/ST1007	\$164,352	\$11,063	5,278 #'S
2-180-6-V	3.0°	%08	200LBS/FT ³	\$262,979	\$17,701	8,445 #'S
2-250-4-V	3.0°	37%	500LBS/FT ³	\$51,509	\$3,467	1,654 #'S
TOTALS				\$478,840	\$32,231	15,377 #'S

PERMA BALLAST® INSTALLATION

OPTION 12 (8TH DECK VOIDS 3.0° LIST)

TANK	LIST	PERCENT	TYPE	PROD \$	MAT \$	MAT#
8-210-10-V	3.0°	2606	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-215-8-V	3.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-215-10-V	3.0°	2606	200LBS/FT3	\$21,186	\$4,425	S.# 006
8-220-8-V	3.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-220-10-V	3.0°	2606	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-225-6-V	3.0°	2606	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-225-8-V	3.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-230-4-V	3.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-230-6-V	3.0°	%06	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-235-6-V	3.0°	206	200LBS/FT ³	\$21,186	\$4,425	S.# 006
8-235-8-V	3.0°	2606	200LBS/FT ³	\$21,186	\$4,425	S.# 006
TOTALS				\$233,046	\$48,675	S.# 006'6